

# Helium tank comparison

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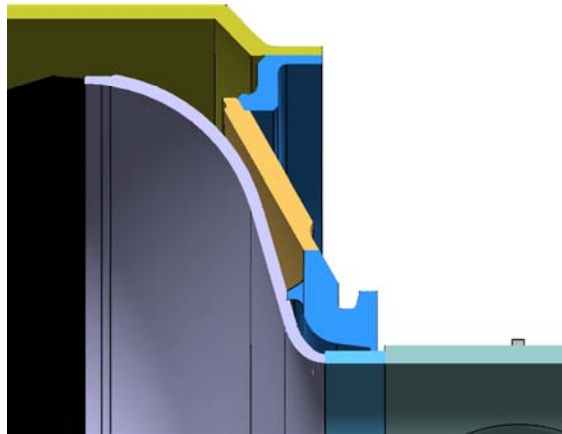
*LASA*



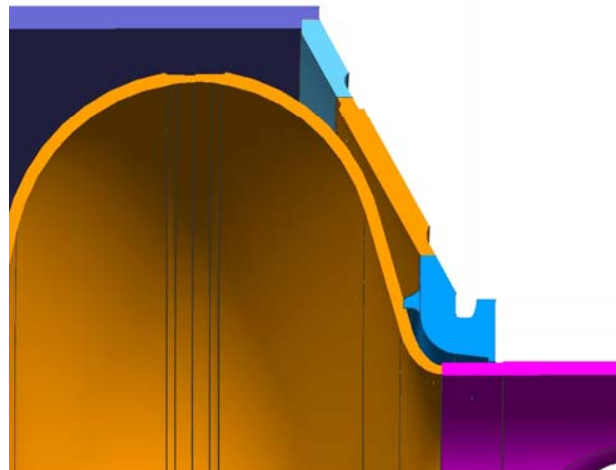
# history - versions

HT Version	Cavity	Tuner	Pad distance	End cones
TTF type	Long	Saclay	750 mm	Old
ILC 1.0	Long/Short	Coaxial	600 mm	Old + "L" adapting ring
ILC 1.3	Long/Short	Coaxial slim	750 mm	Old + straight adapting ring
ILC 2.0	Long/Short	Coaxial slim	750 mm	New

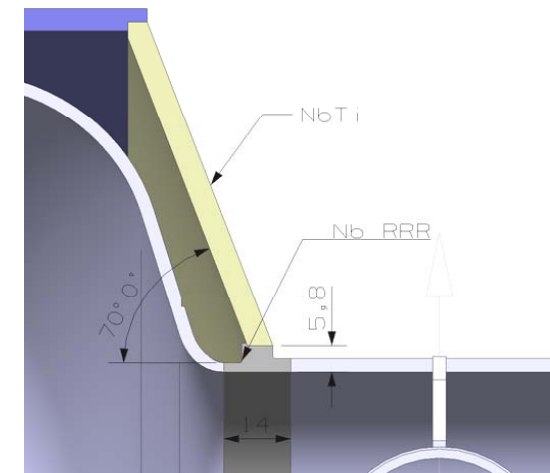
ILC 1.0



ILC 1.3



ILC 2.0





# Materials

## **Ti Grade 1**

ASME VIII, div 2, part AM, table ANF-1.4, density = 4500 kg/m<sup>3</sup>

- Bellow: spec. no. SB265/UNSR50400

$$f_t = 240 \text{ MPa}; f_{0.2} = 170 \text{ MPa}; \quad \epsilon_t > 24\% \quad S_m = 80 \text{ MPa @ } 20^\circ\text{C}; \quad E = 106870 \text{ MPa}$$

## **Ti Grade 2**

ASME VIII, div 2, part AM, table ANF-1.4, density = 4500 kg/m<sup>3</sup>

- Vessel: spec. no. SB265/UNSR50250                      Forged rings: spec. no. SB381/F2

$$f_t = 345 \text{ MPa}; f_{0.2} = 275 \text{ MPa}; \quad \epsilon_t > 20\% \quad S_m = 115 \text{ MPa @ } 20^\circ\text{C} \quad E = 106870 \text{ MPa}$$

## **NbTi**

The niobium / titanium ally is not considered by the ASME pressure vessel code. The mechanical characteristics are taken from the literature, while the  $S_m$  value is assumed equal to  $f_y/(1.5*1.6)$

$$f_t = 550 \text{ MPa}; \quad f_{0.2} = 480 \text{ MPa}; \quad \epsilon_t > 30\% \quad S_m = 200 \text{ MPa @ } 20^\circ\text{C}; \quad E = 62000 \text{ MPa}$$

density = 5700 kg/m<sup>3</sup>

## **Nb RG**

The niobium material is not considered by the ASME pressure vessel code. The mechanical characteristics are taken from the test performed by Myneni and Umezawa [1], while the  $S_m$  value is assumed equal to  $f_y/(1.5*1.6)$ . The heat treatment at 800°C has been taken into account.

$$f_t = 90 \text{ MPa}; \quad f_{0.2} = 60 \text{ MPa}; \quad \epsilon_t > 50\% \quad S_m = 25 \text{ MPa @ } 20^\circ\text{C}; \quad E = 102700 \text{ MPa}$$

density = 8700 kg/m<sup>3</sup>

## **Nb RRR**

Like for the RG niobium, the mechanical characteristics are taken from the test performed by Myneni and Umezawa [1], while the  $S_m$  value is assumed equal to  $f_y/(1.5*1.6)$ . The heat treatment at 800°C has been taken into account.

$$f_t = 130 \text{ MPa}; \quad f_{0.2} = 40 \text{ MPa}; \quad \epsilon_t > 47\% \quad S_m = 16 \text{ MPa @ } 20^\circ\text{C}; \quad E = 102700 \text{ MPa}$$

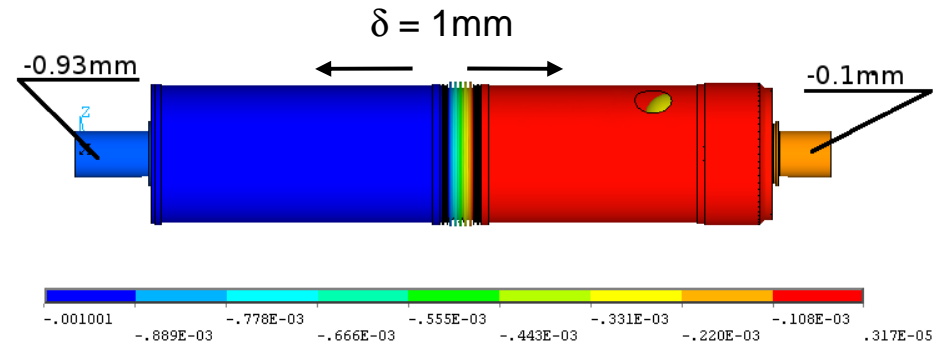
density = 8700 kg/m<sup>3</sup>



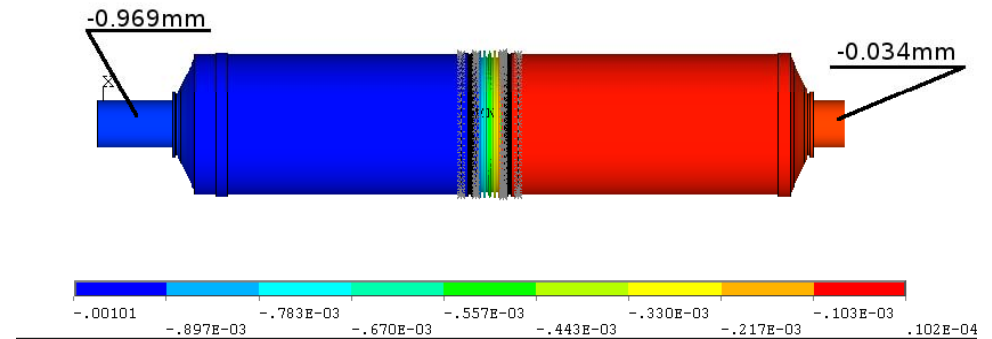
# tuning efficiency

- The elongation of the cavity due to the tuning, depends from the end-cones stiffness

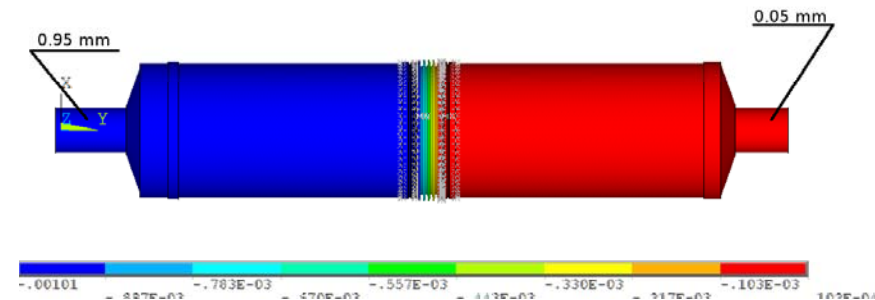
ILC 1.0: efficiency 83%



ILC 1.3: efficiency 93%



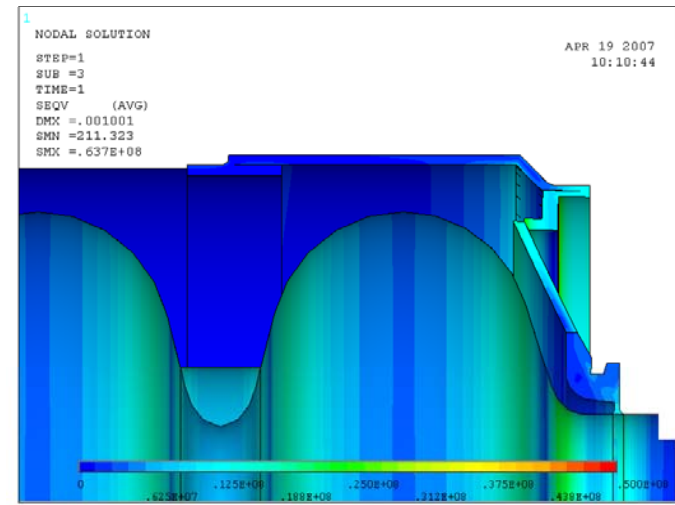
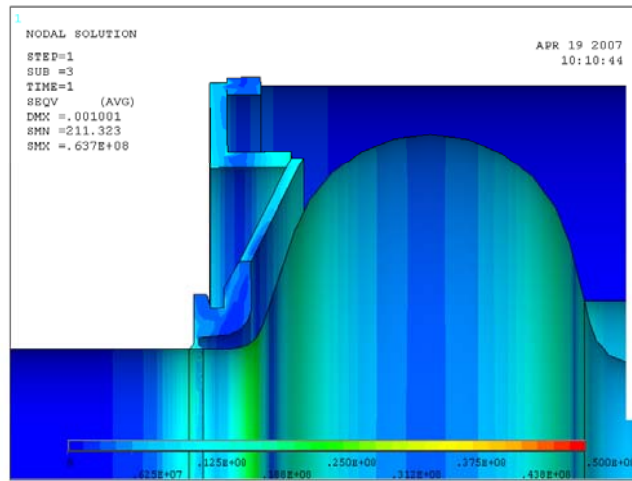
ILC 2.0: efficiency 90%



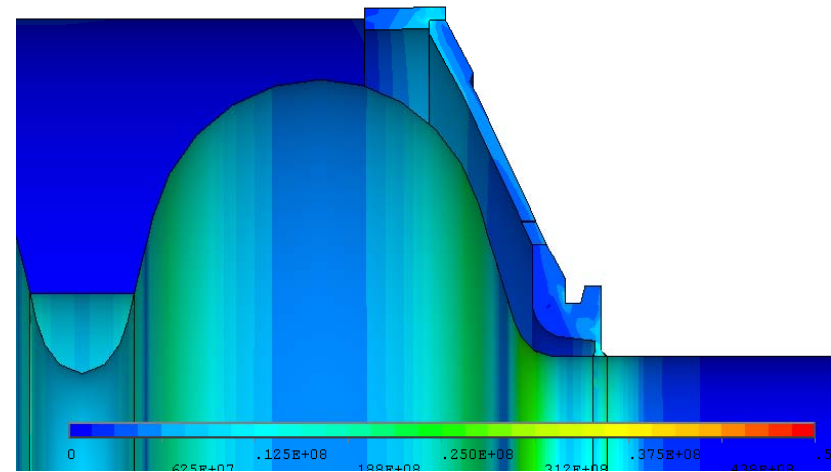
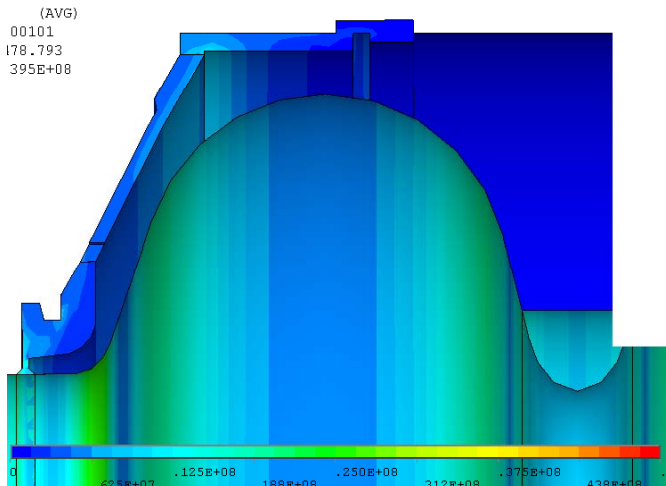


# stress during tuning

ILC 1.0



ILC 1.3

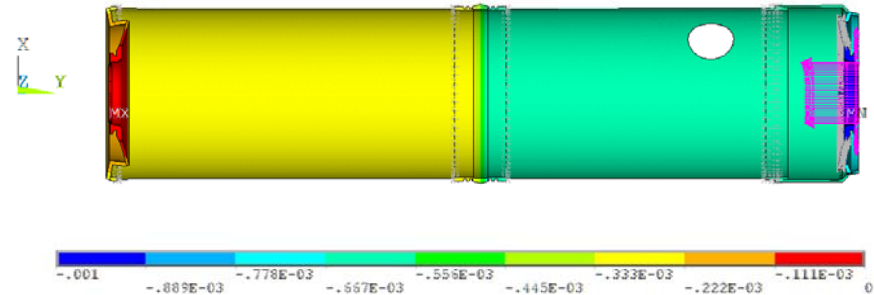




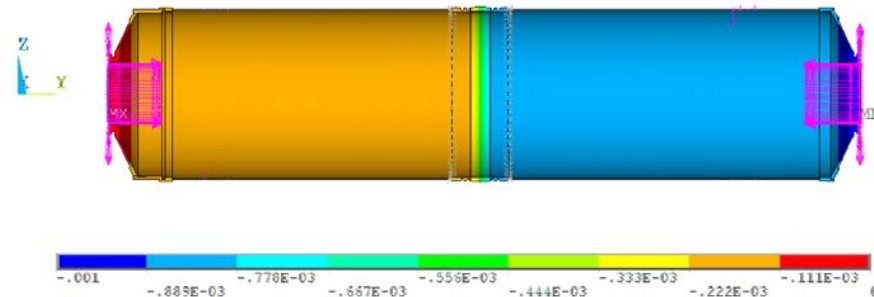
# Lorentz force detuning

- The helium tank + tuner represents the external constraint to the cavity length
- The piezo+tuner has been replaced by 3D elements with the same stiffness (22.3 kN/mm)

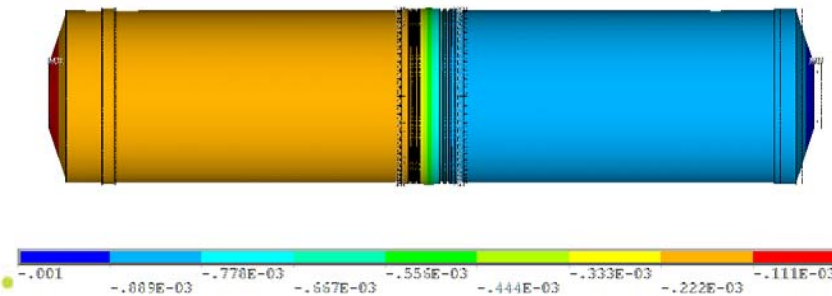
ILC 1.0:  $K_{\text{ext}} = 9700 \text{ N/mm}$



ILC 1.3:  $K_{\text{ext}} = 17500 \text{ N/mm}$



ILC 2.0:  $K_{\text{ext}} = 15300 \text{ N/mm}$





# ASME check

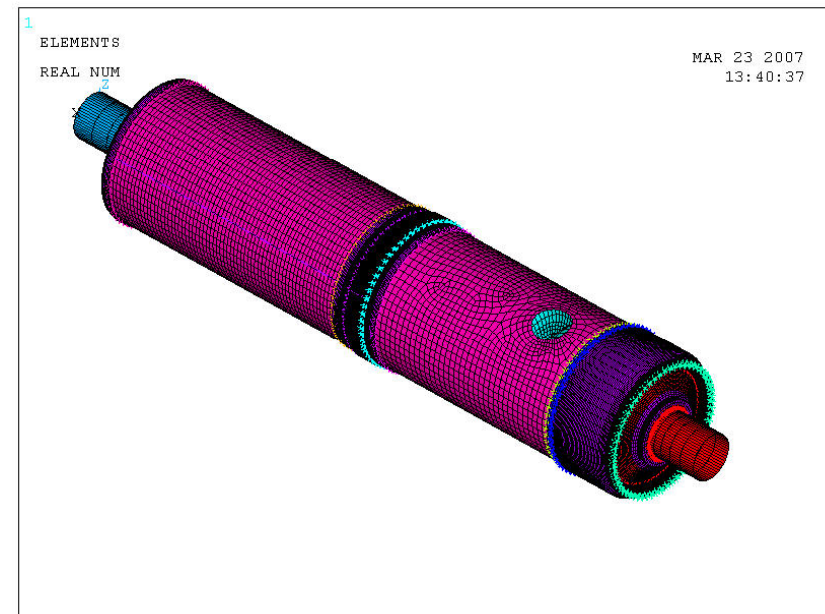
- 3D FE model with brick and shell elements
- connections/welds with contact elements

## ASME requirements:

- Materials: Ti is considered by ASME, Nb not.
  - for this first check I would use the material characteristics of Nb RRR as obtained by Myneni and Umezawa
- For check at cryogenic temperature, structural properties higher than that at room temperature can not be considered

## Load cases:

- From T.P. january presentation:
  - 2bar at warm (only during filling)
  - 4bar at cold (emergency case)
  - 20mar working pressure
- The bellow is tested at 3bar



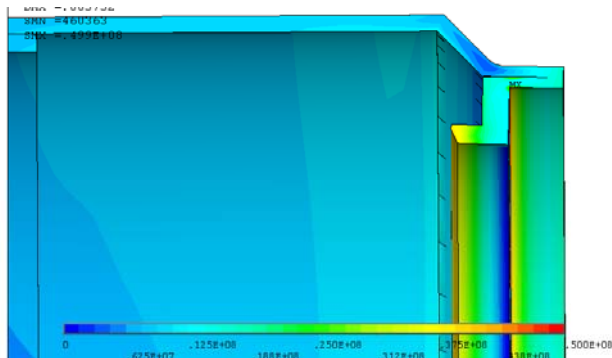


# check of helium tank (Ti Gr2 parts)

- $p = 2 \text{ bar @ RT}$
- $S_m = 115 \text{ MPa}$ ; joint efficiency = 0.9  $\rightarrow$  max stress =  $1.5 \cdot 0.9 \cdot 115 = 155 \text{ MPa}$

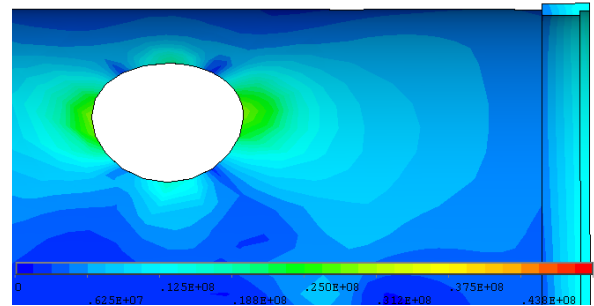
ILC 1.0

$\sigma_{\max} = 47 \text{ MPa}$



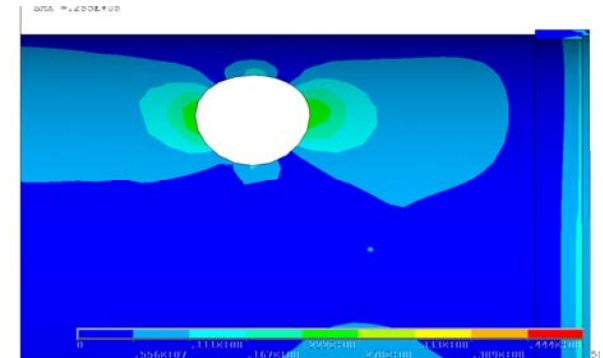
ILC 1.3

$\sigma_{\max} = 30 \text{ MPa}$



ILC 2.0

$\sigma_{\max} = 28 \text{ MPa}$





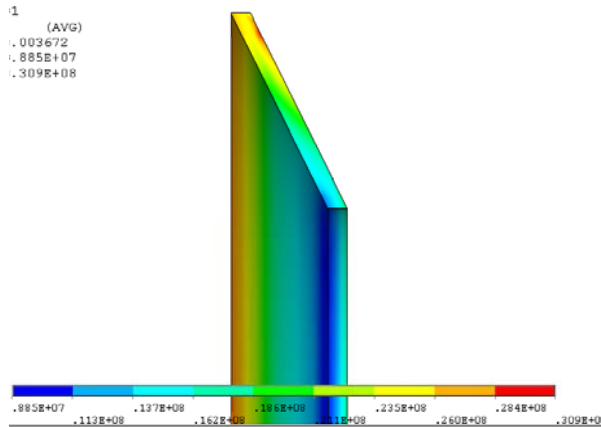


# check of end cones (NbTi parts)

- $S_m = 200 \text{ MPa}$ ; joint efficiency = 0.9  $\rightarrow$  max stress =  $1.5 \cdot 0.9 \cdot 200 = 270 \text{ MPa}$

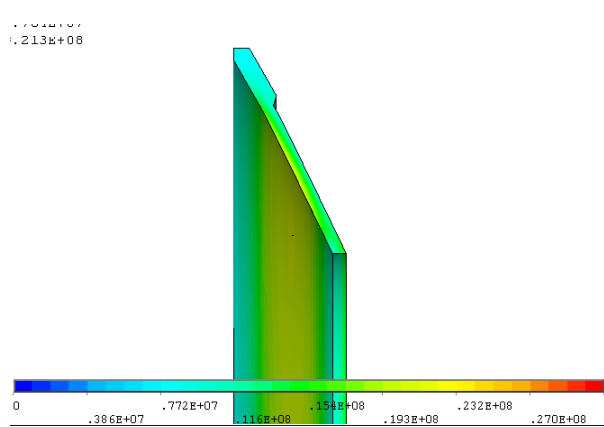
ILC 1.0

$\sigma_{\max} = 30.9 \text{ MPa}$



ILC 1.3

$\sigma_{\max} = 21.3 \text{ MPa}$



ILC 2.0

$\sigma_{\max} = 11.6 \text{ MPa}$

