



Magnetic and Mechanical FEA of SiD

IRENG07

Bob Wands

September 18, 2007

Global Design Effort

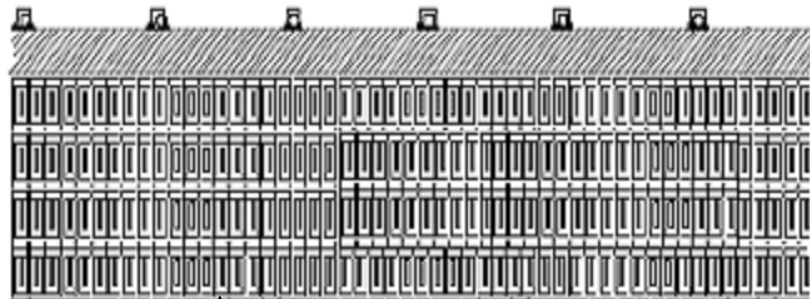


Brief Review of Solenoid Design

- **Design is based on a scaling of the successful CMS solenoid**
- **Six layers of CMS conductor are used vs. four layers for CMS**
- **Operating current is 17500 amps vs. 19500 amps for CMS**
- **Central Field is 5 T vs. 4 T for CMS**
- **Stored energy is 1.4 GJ vs. 2.7 GJ for CMS**

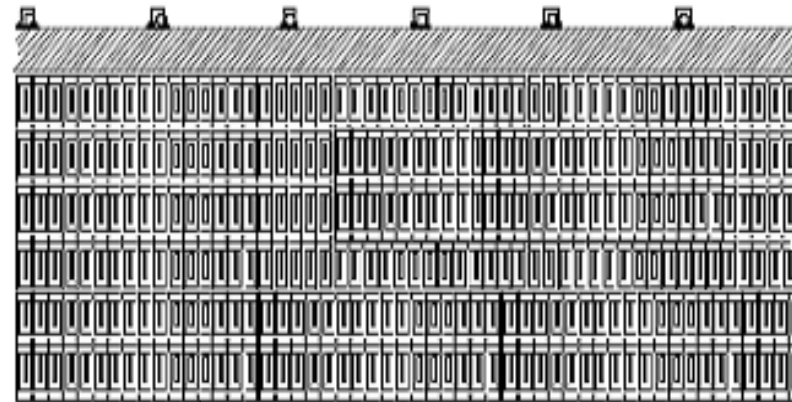


Comparison of Coils – SiD and CMS



CMS (L = 12.5m)

R = 3.095m



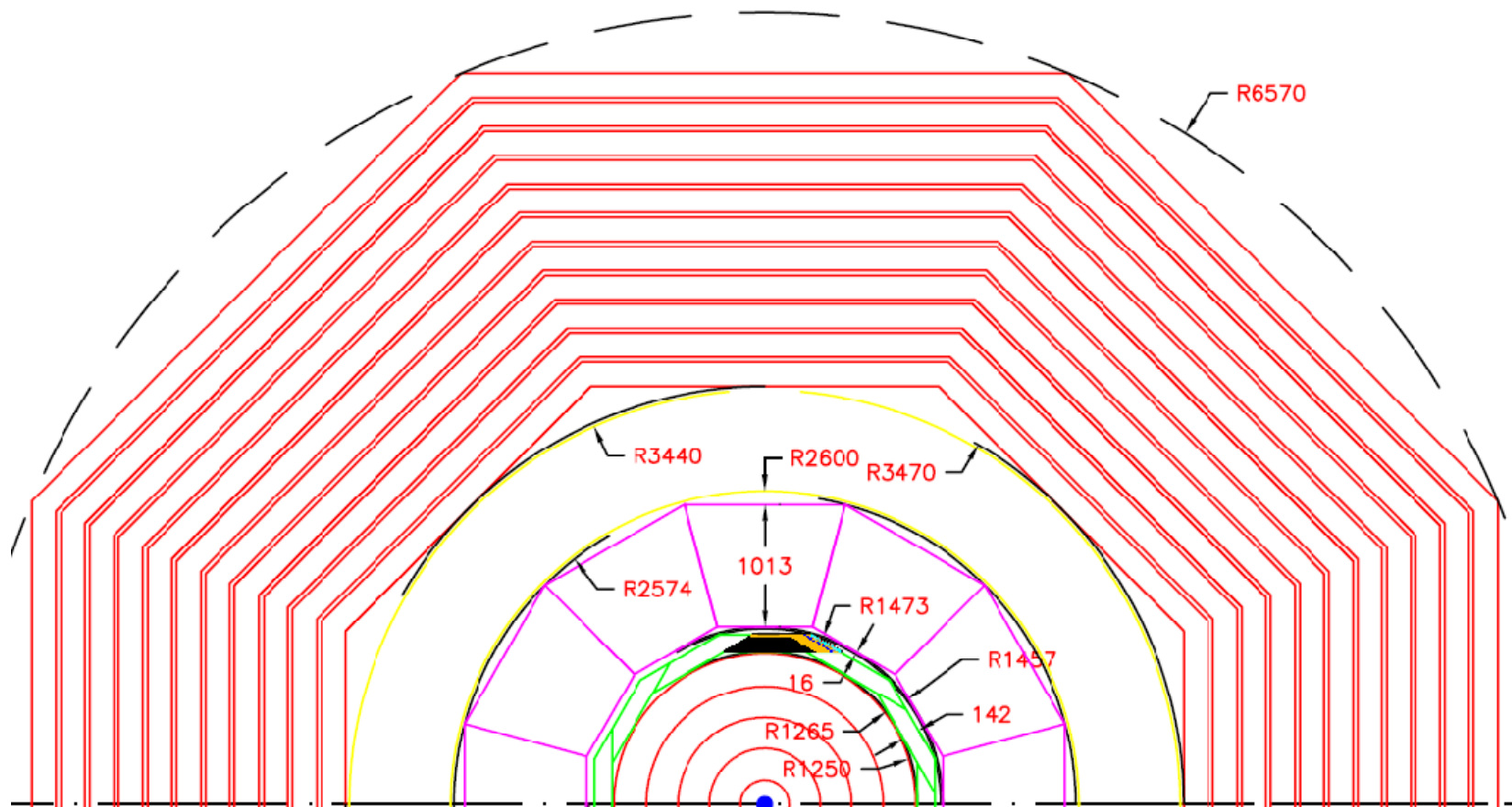
SiD (L = 5.18m)

R = 3.098m

R = 2.645m

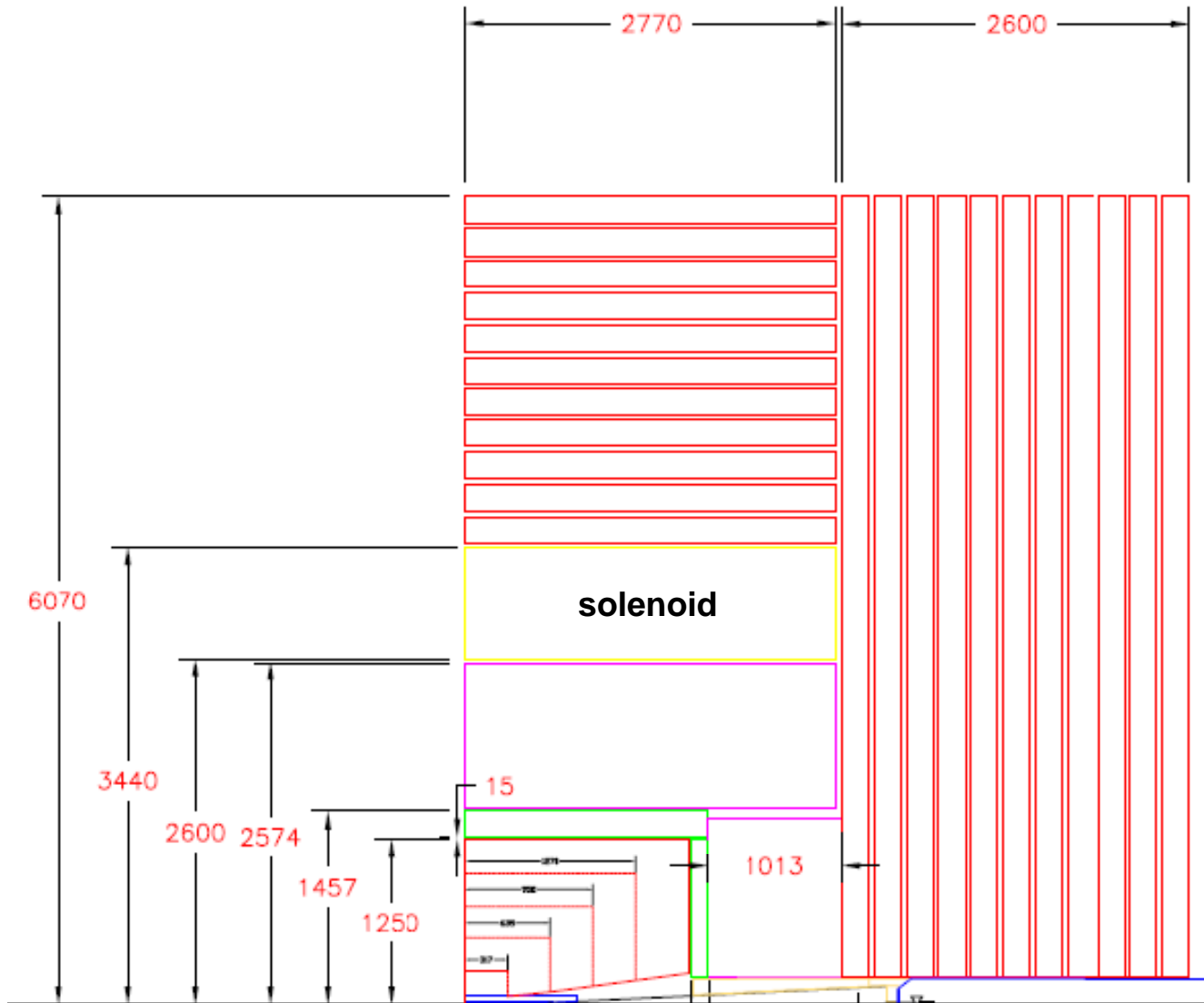


SiD Geometry – End View





SiD Geometry – Side View



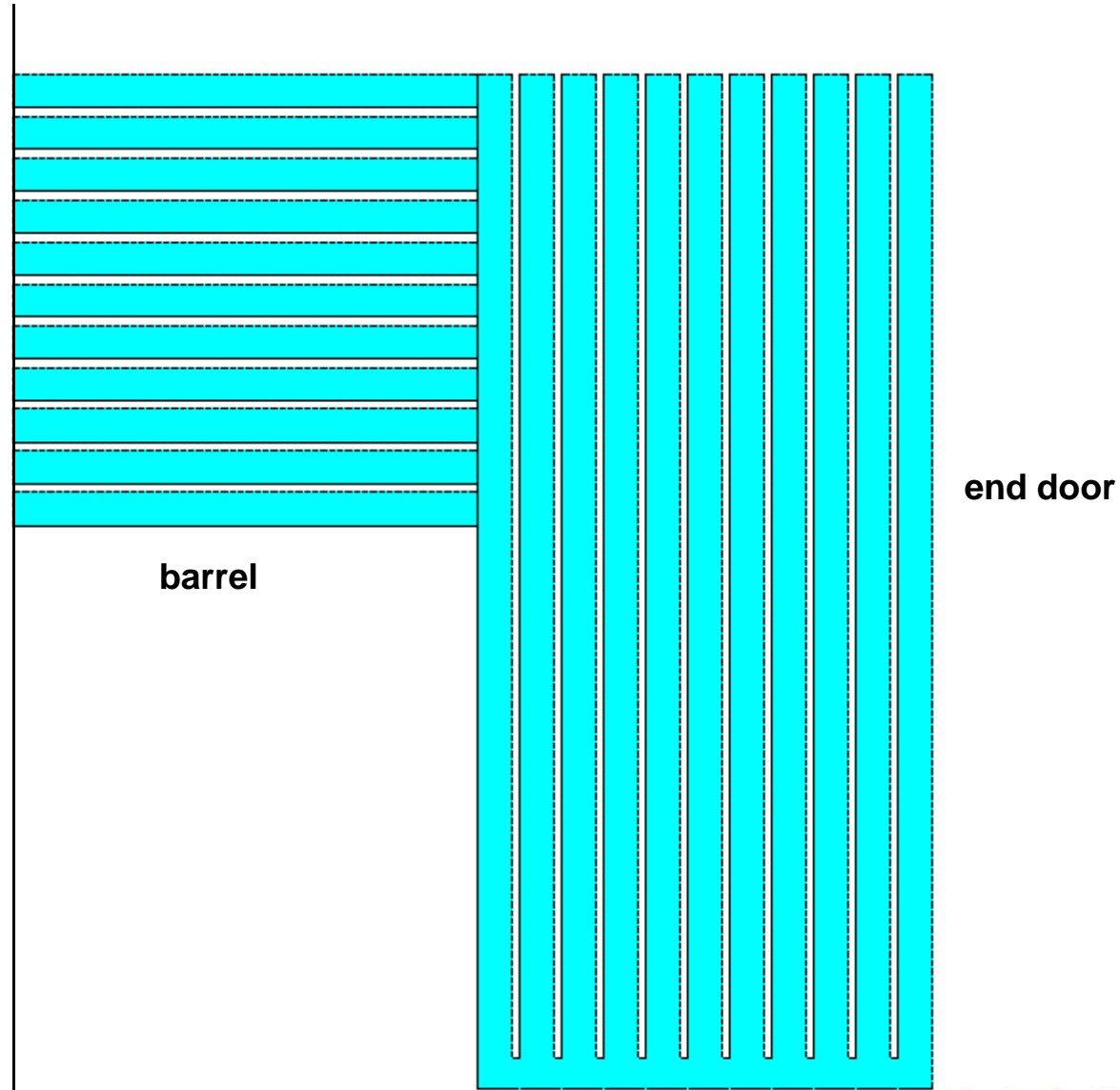


Iron Configuration – Effect on Fringe Fields

- The basic configuration is 11 iron plates, 20 cm thick, in both the barrel and end door regions
- Gaps between plates are 4 cm
- Variations on this geometry were examined in two dimensions to determine the features that most affected fringe fields outside the barrel
- *Fringe field requirements have yet to be specified. Could be as low as 5 g close to the magnet, or >100 g.*
- *The 2-d axisymmetric models did not include the PacMan shielding iron, or the anti-solenoids*
- *For all but one configuration (the “practical design”) no consideration was given as to how to actually build the iron*

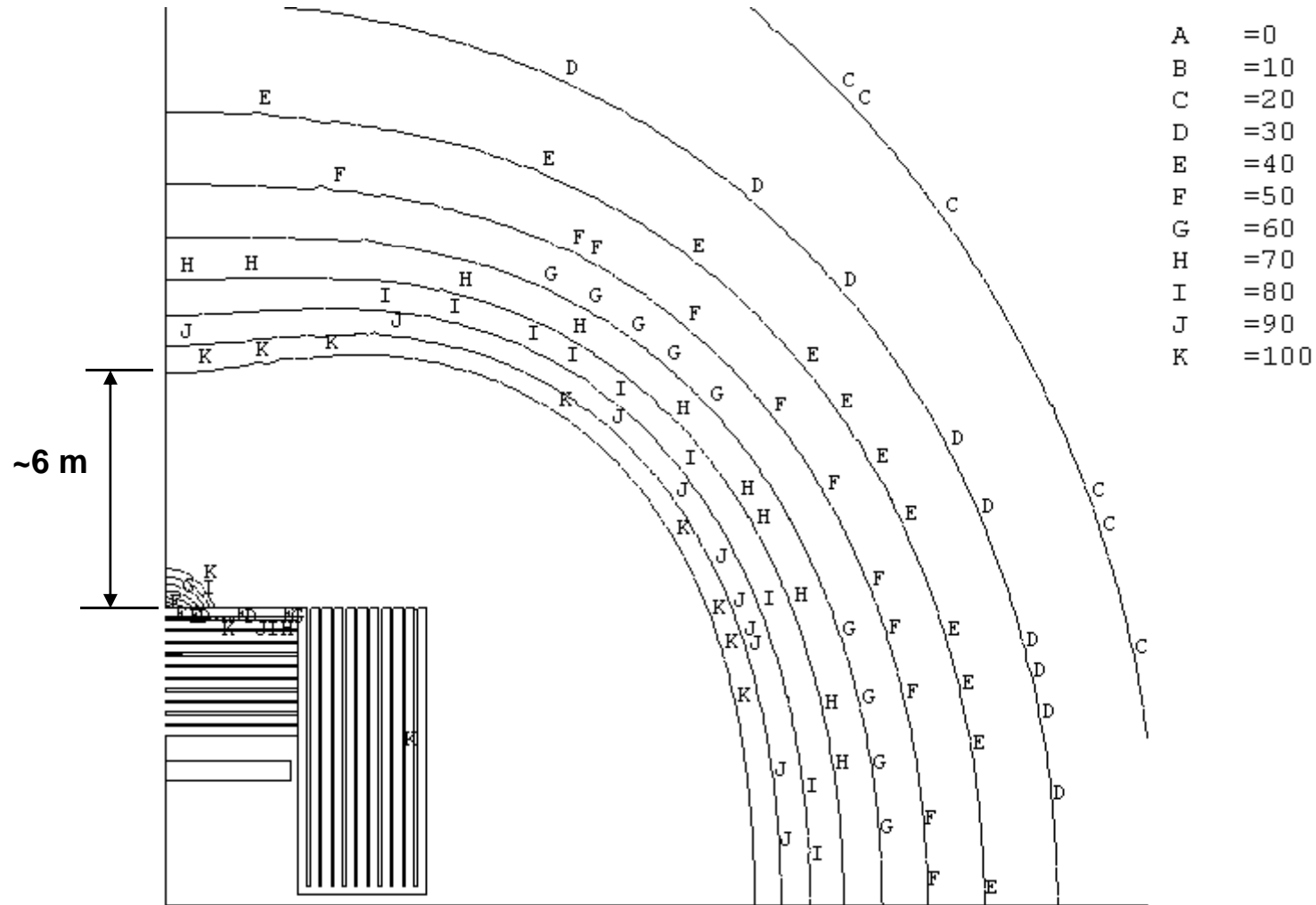


The Simplest Configuration (1/4 shown)





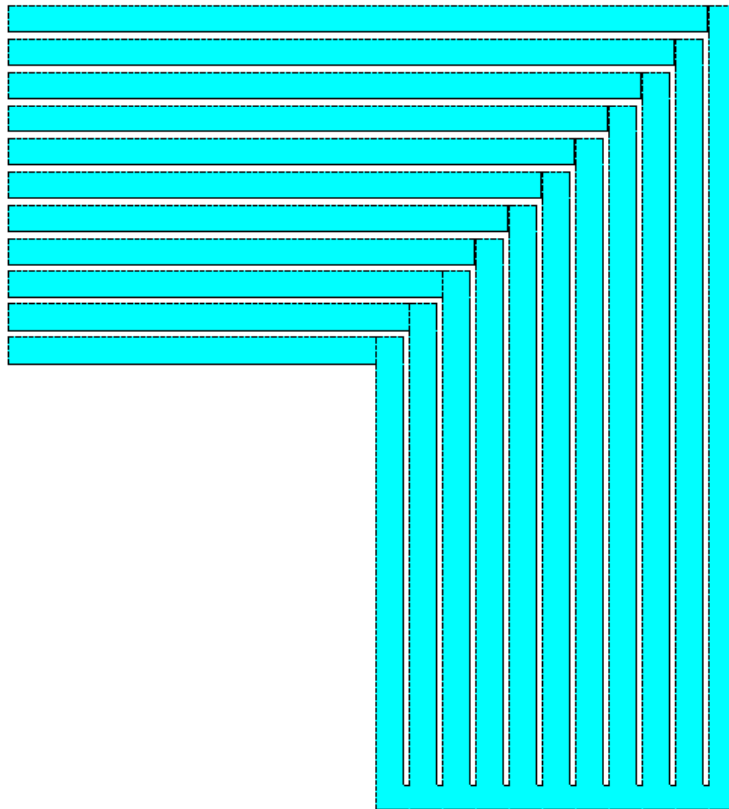
Fringe Field - Simplest Configuration



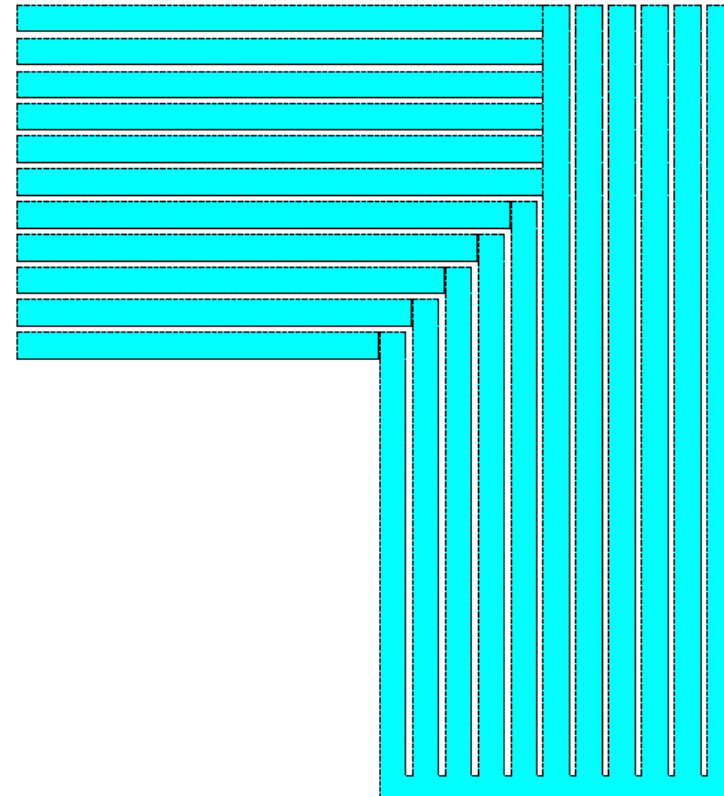


Two Tapered Configurations

Fully Tapered



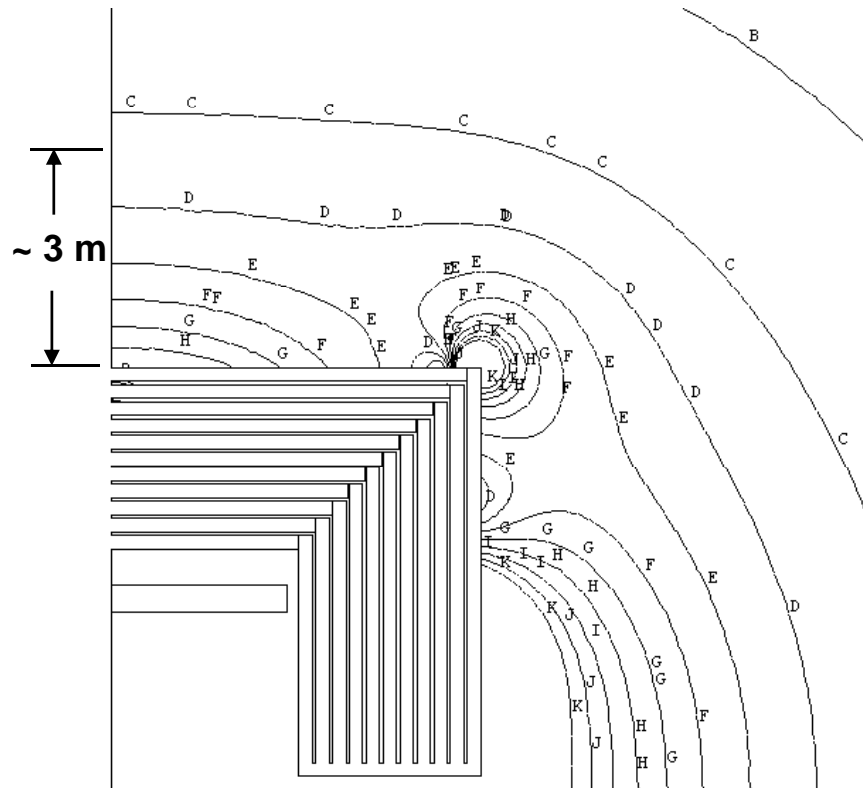
Partially Tapered



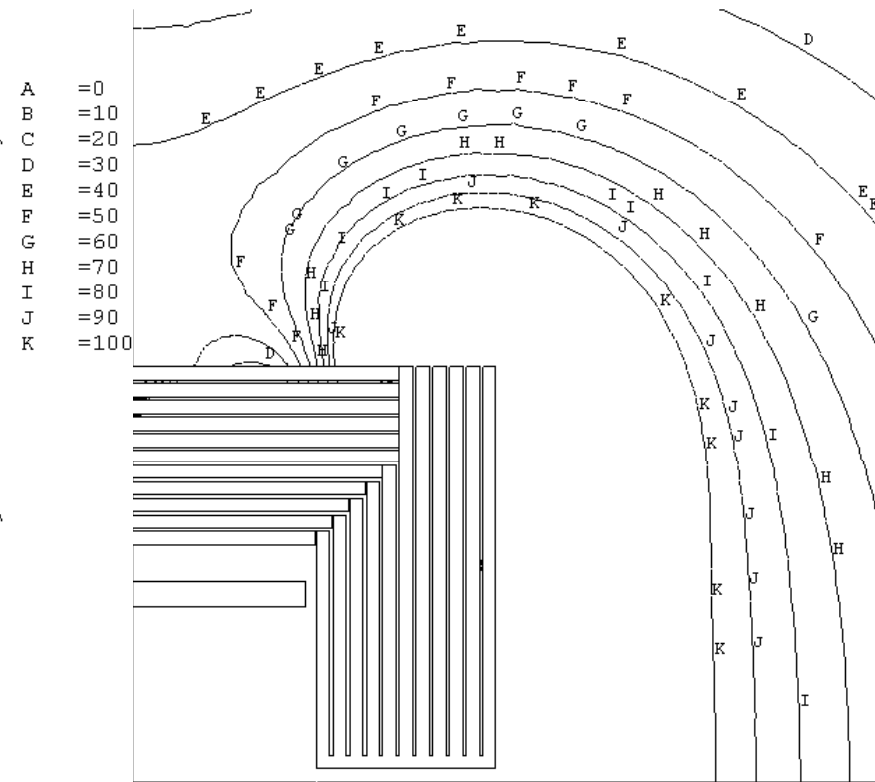


Fringe Field - Two Tapered Configurations

Fully Tapered



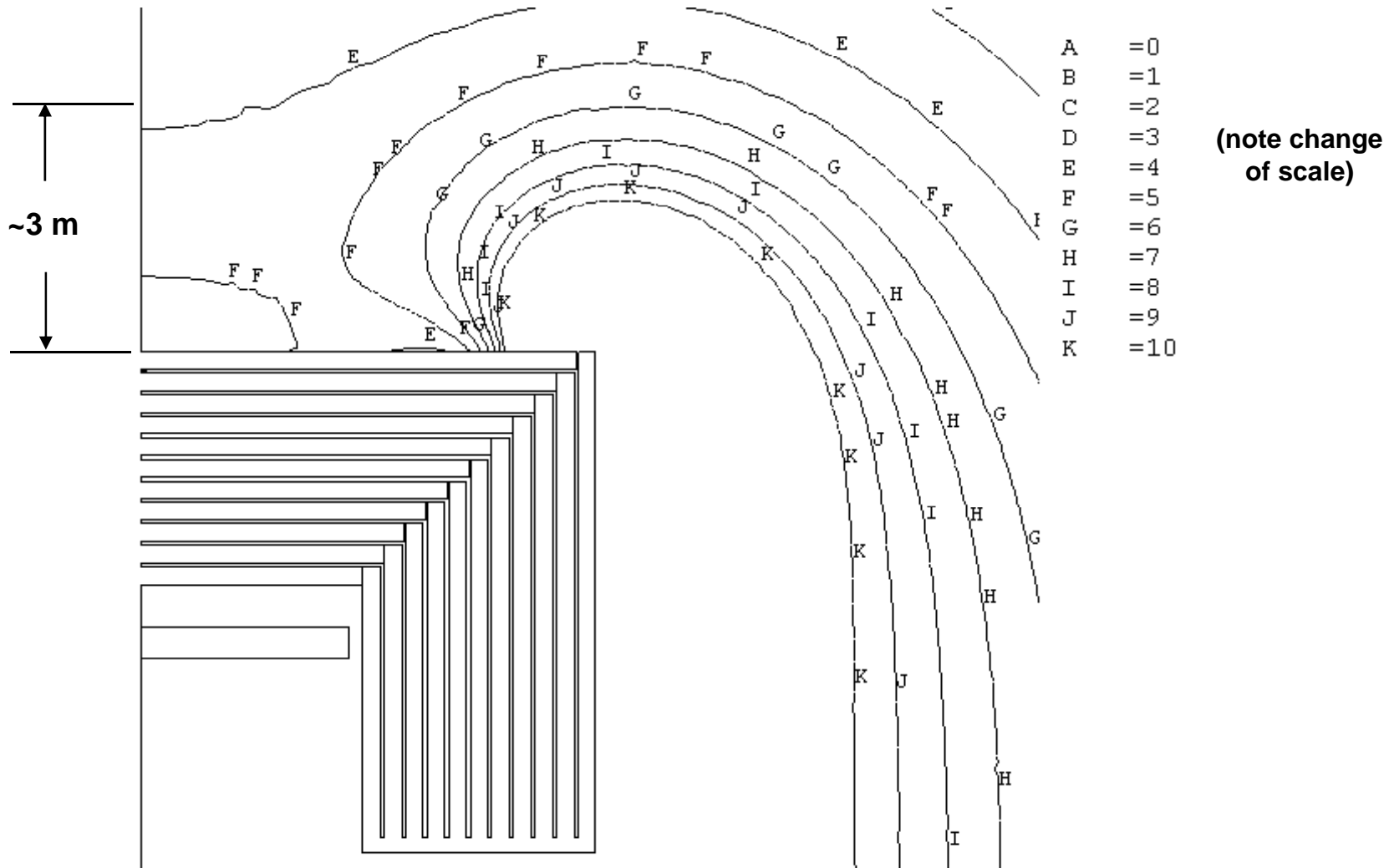
Partially Tapered



- A =0
- B =10
- C =20
- D =30
- E =40
- F =50
- G =60
- H =70
- I =80
- J =90
- K =100



Fully-Tapered Configuration, 22 cm Plates



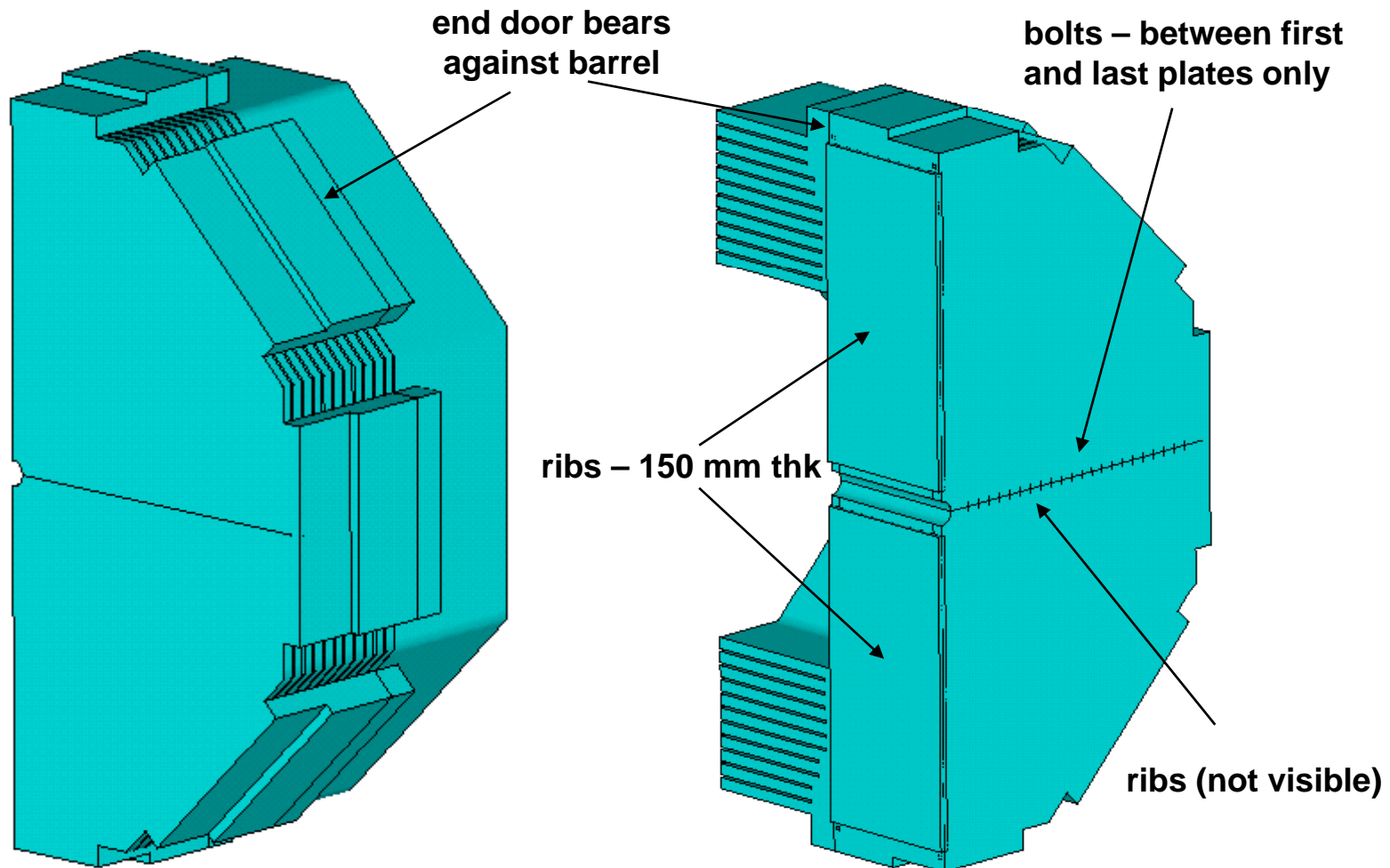


Requirements for a Practical Iron Design

- End doors must resist axial magnetic force of ~14000 tonnes
- Dead weight of the iron alone is in excess of 8000 tonnes
- Crane and space considerations require segmenting of barrel and end door
- Design must allow for adequate detector coverage while maintaining small deflections
- Fringe fields must be within specifications



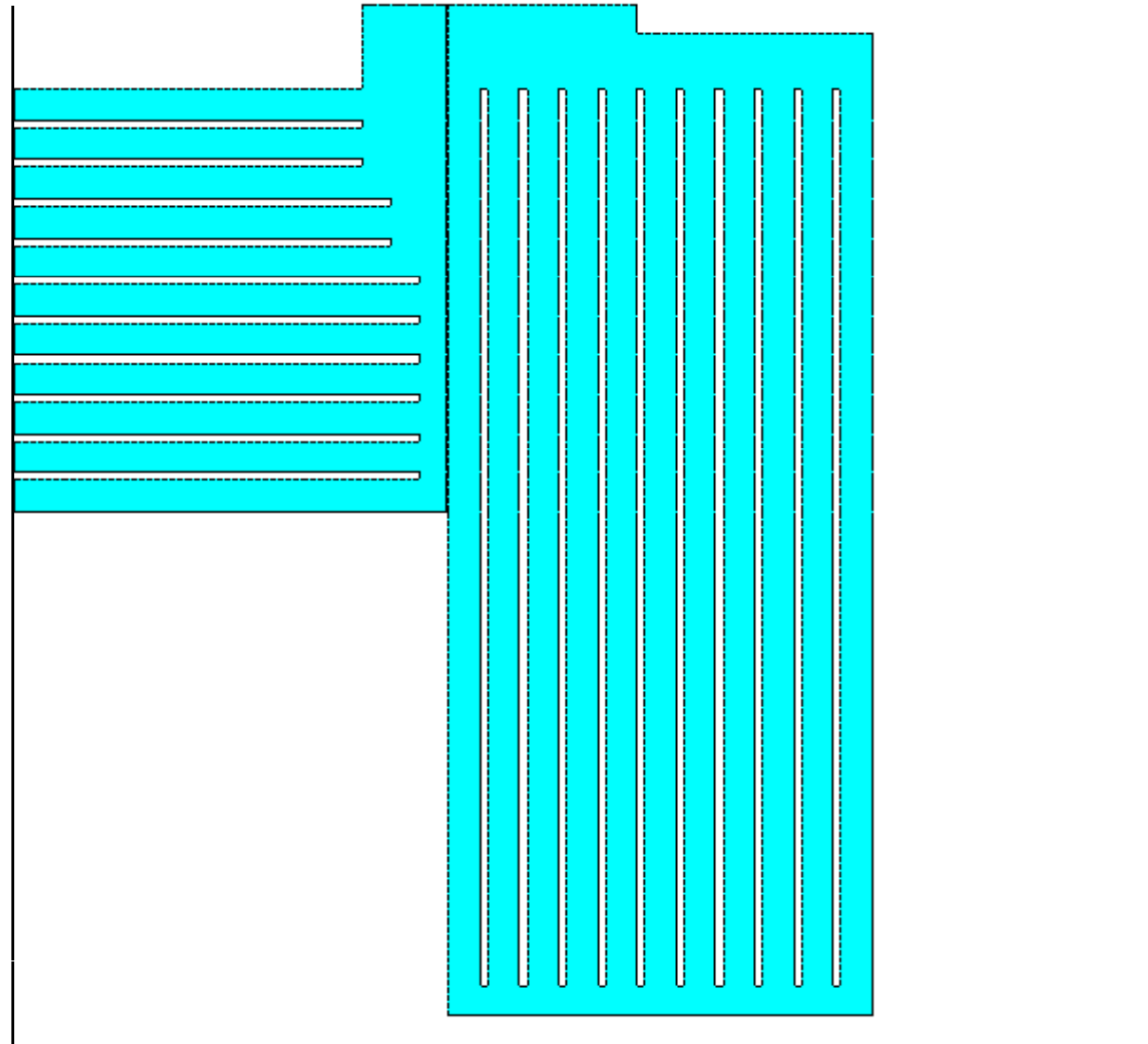
A Practical Iron Design (H.J. Krebs)



total weight (barrel + both end doors) = 8300 tonnes

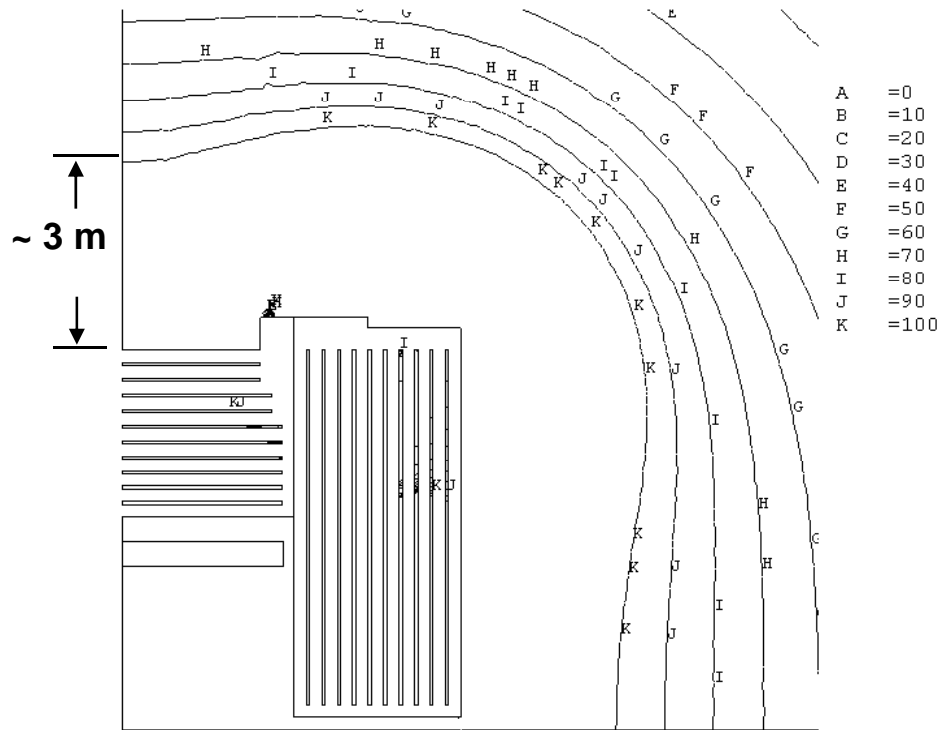


A Practical Iron Design – Magnetic Model

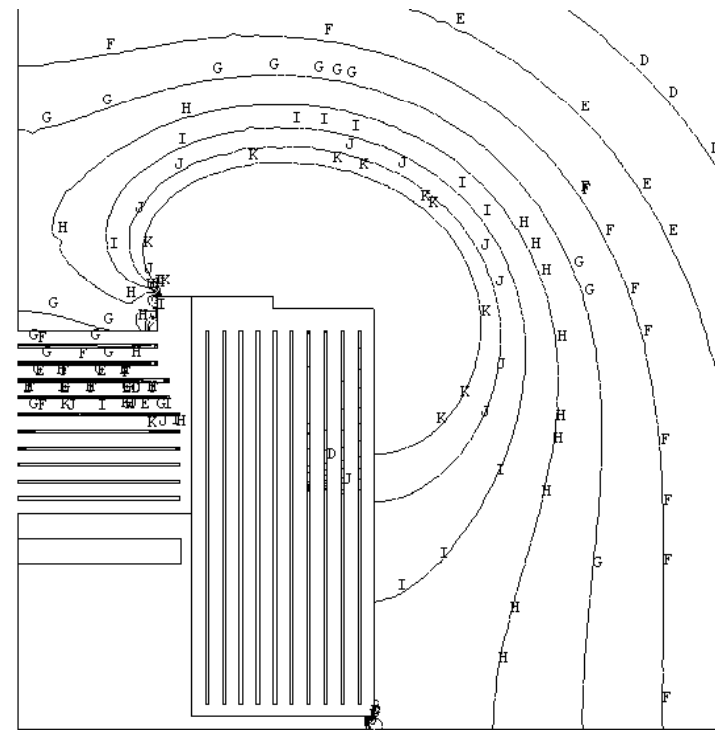




Fringe Fields – Practical Design



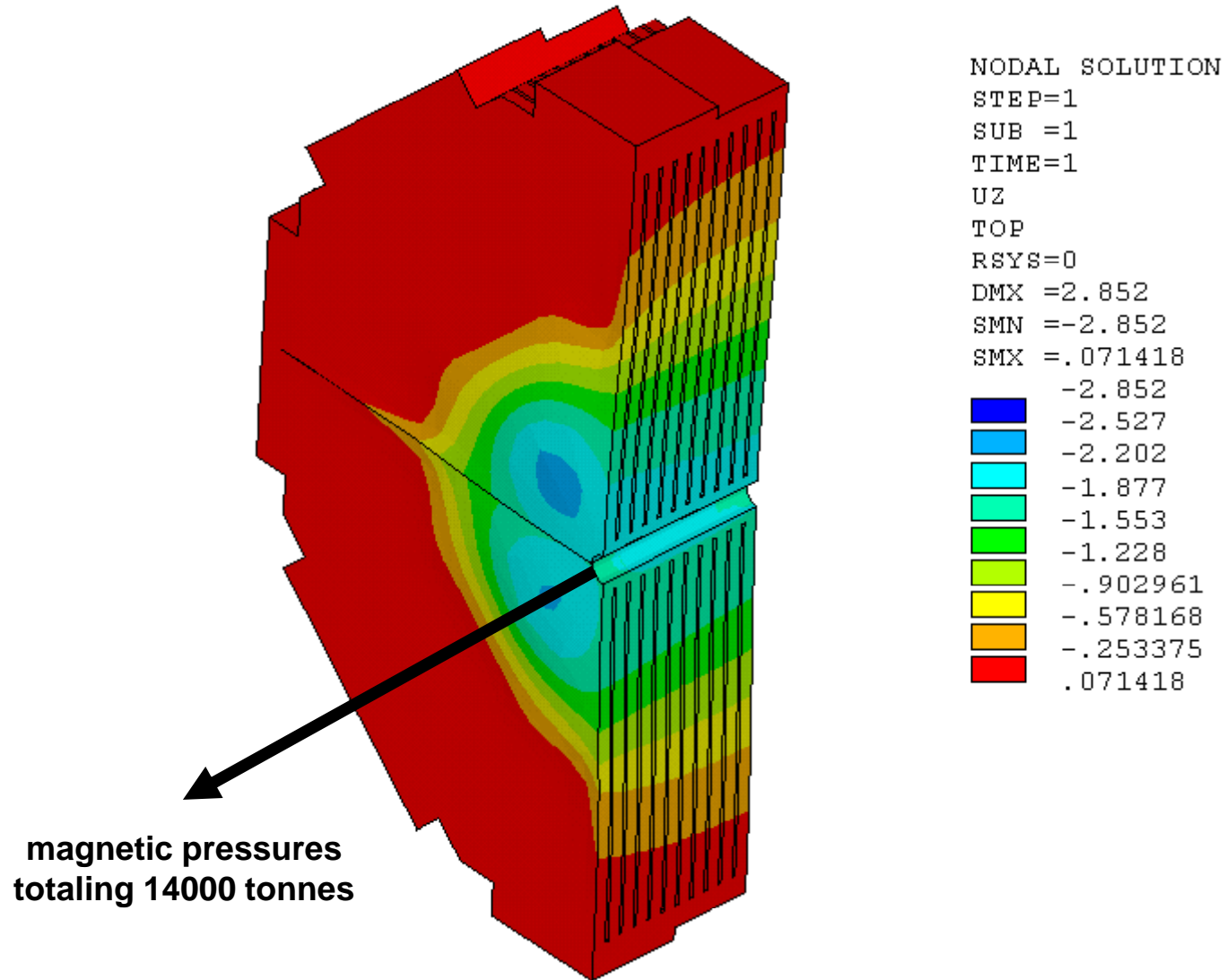
20 cm plates



22 cm plates

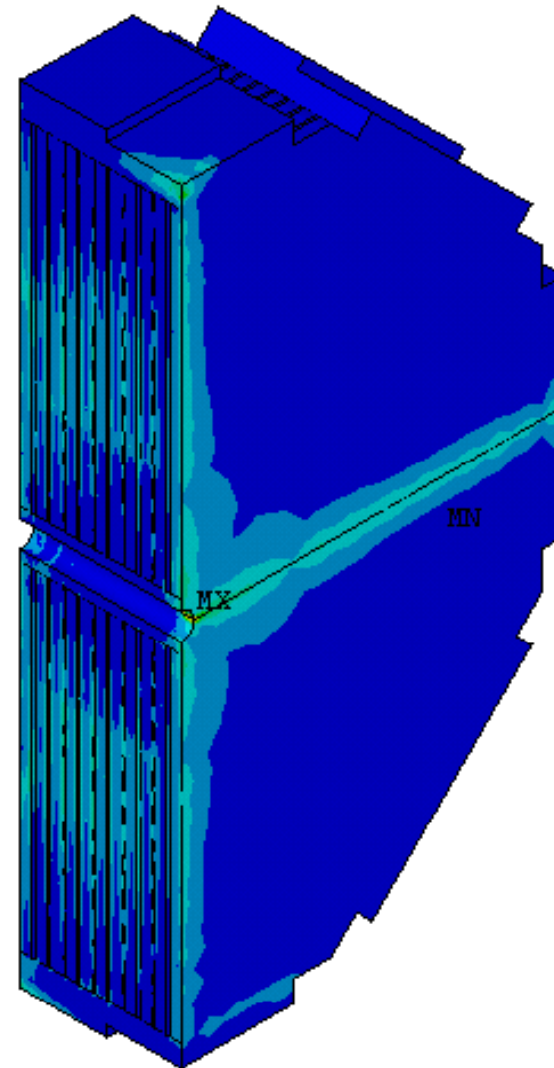


Axial Deflections of End Door - mm





Stresses in End Door - MPa



NODAL SOLUTION
STEP=1
SUB =1
TIME=1
SINT (AVG)
PowerGraphics
EFACET=1
AVRES=Mat
DMX =2.849
SMN =.092389
SMX =196.335
.092389
21.897
43.702
65.507
87.311
109.116
130.921
152.726
174.53
196.335



Conclusions

- Only full tapering with 22 cm plates produces 5 gauss fringe fields near the barrel, but support of end door is problematic
- Practical iron designs will almost certainly be untapered to some extent for ease of assembly and support
- If very low fringe fields (~ 5 g) are required, then thicker iron plates may be called for, increasing weight and cost
- The practical design considered here is *extremely* robust, and could probably be optimized to increase space available for detectors