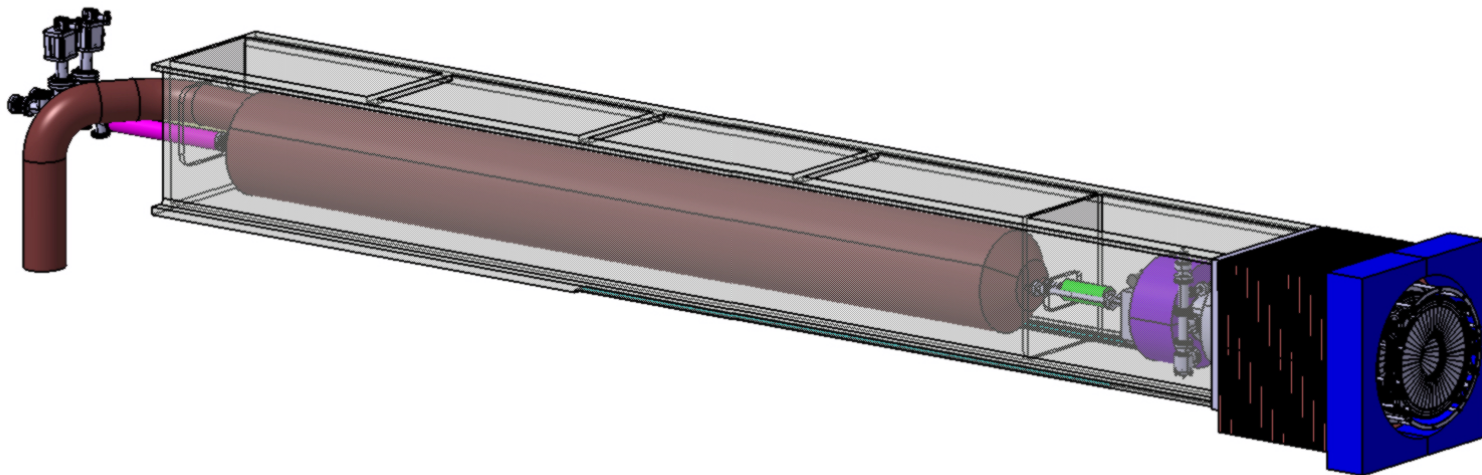


ILD2 square support tube solution

2nd ILD workshop - Cambridge

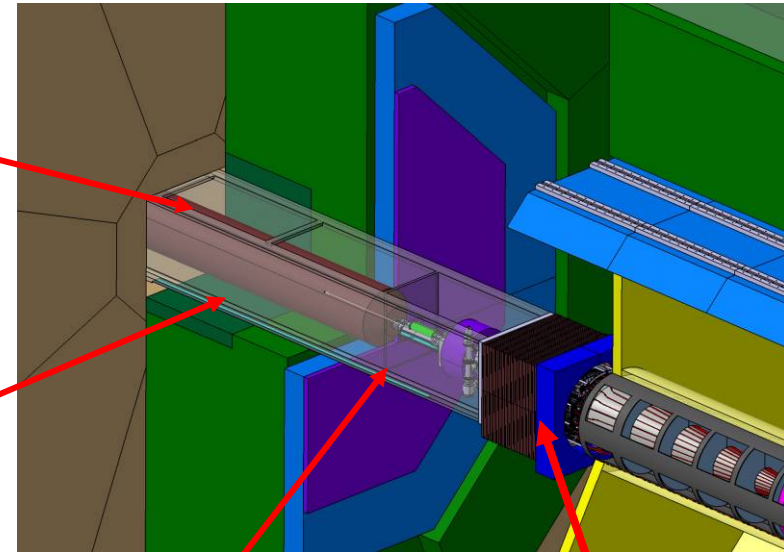


- Requirements on Forward region
- 2 Solutions of a square support tube
 - **Why a square tube?**
 - **Solution 1 : pillar + rails**
 - Principle
 - FEA calculations (deformation, stress, natural frequencies)
 - Alignment method
 - Conclusion
 - **Solution 2 : pillar + tie rods**
 - Principle
 - Alignment method
 - Conclusion
- Conclusion

- Description

QD0
(superconducting magnet)

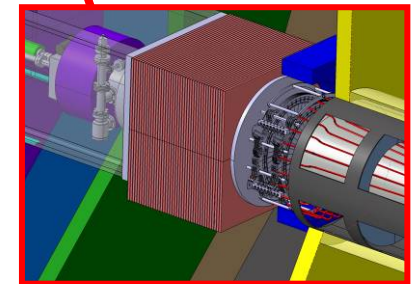
Support tube



Beam line components

- Requirements on support tube

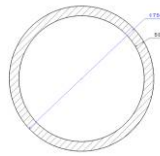
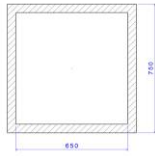
- **Support all the forward components**
- **Good vibration performance (QD0 stability)**
- **Allowable amplitude**
 - Few mm in static load
 - About 50nm for ground motion (IR interface document)
- **Alignment system is needed (in a mm range)**



Forward Cals

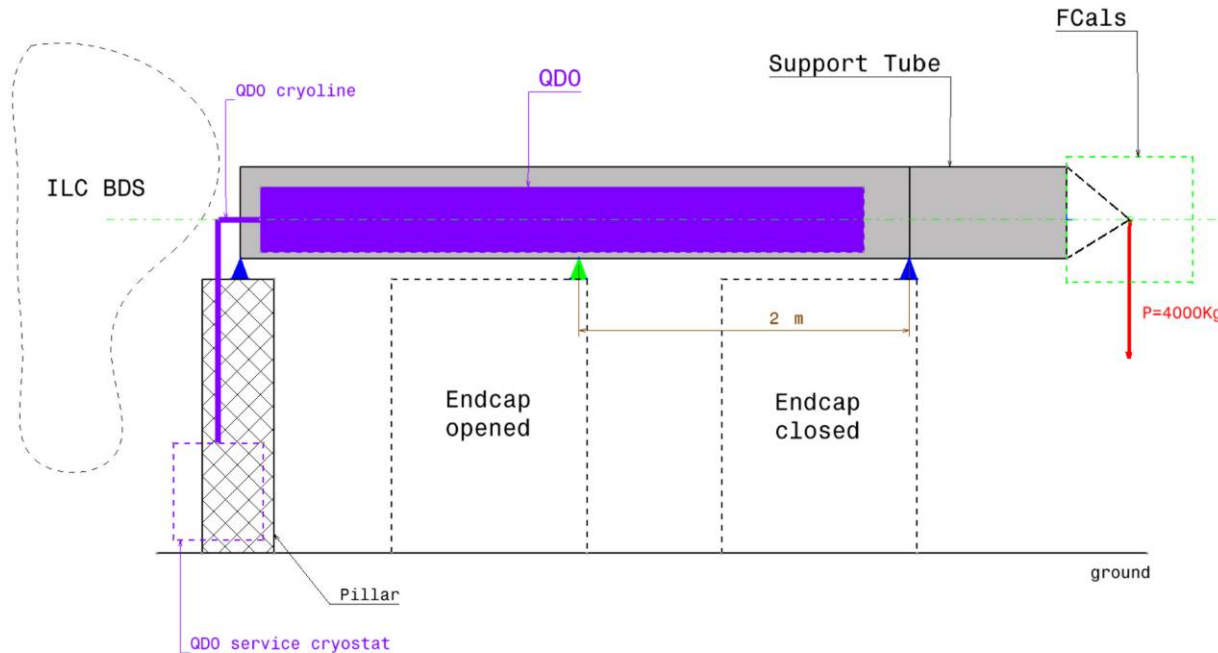
- Why a square support tube?
 - Loading is only weight of each forward components
 - Higher moment of inertia allowed to use less material
- Basic calculation (same thickness and external dimension)

$$def = \frac{Fl^3}{3EI_{Gz}}$$

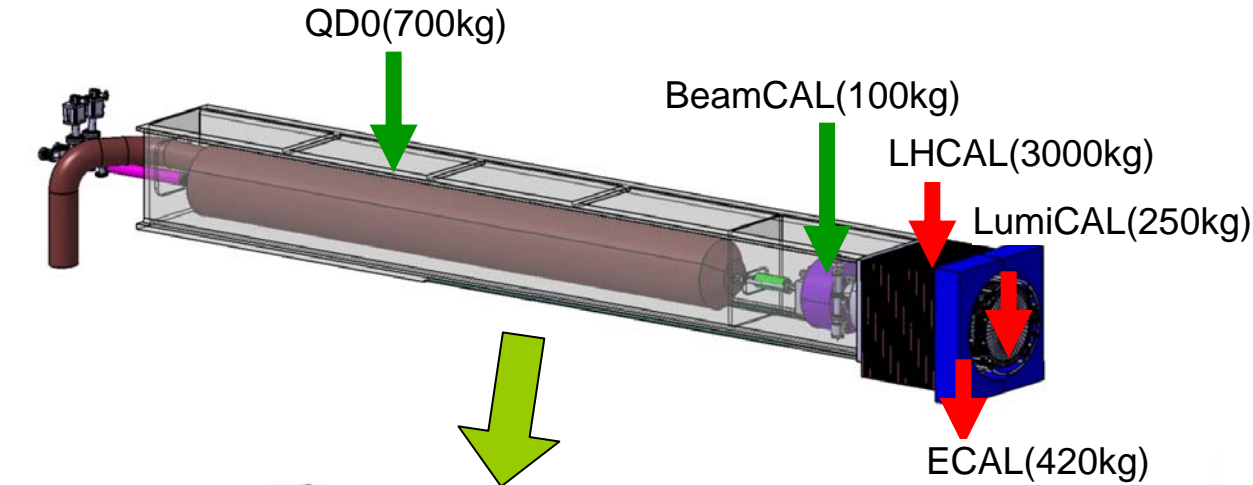
		
Moment of inertia (I_{Gz})	$\frac{\pi}{64}(D^4 - d^4)$	$\frac{bh^3 - b'h^3}{12}$
Calculation	$6,77 \cdot 10^9 \text{ mm}^4$	$11,5 \cdot 10^9 \text{ mm}^4$

→ For same loading and size, deformation is **reduced by 40%**

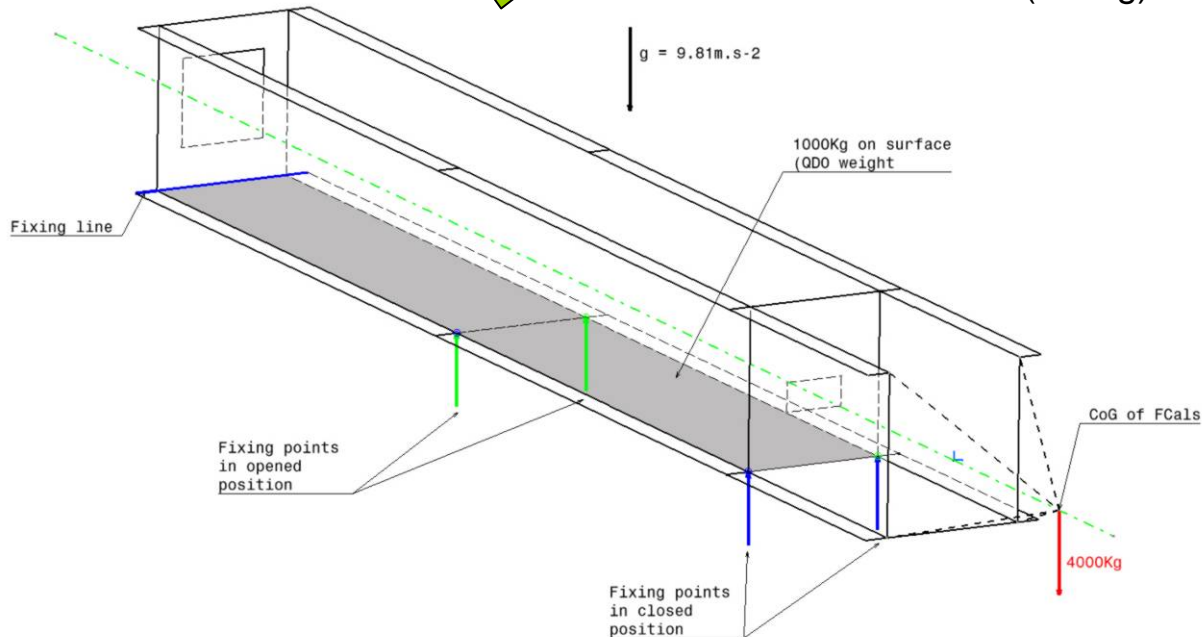
- Supported on
 - **Pillar from ground (machine side)**
 - **EndCap**



- Compared to cantilever solution
 - **Improve vibration stability**
 - **Reduce deformation and stress**



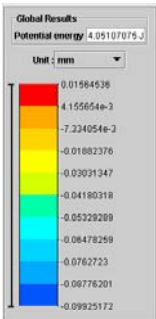
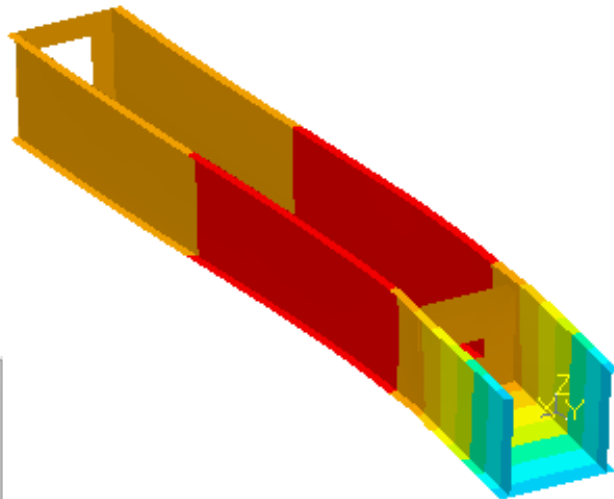
Ab. 4000Kg for FCals



U shape is used for accessing to Forward components
 If needed, structure could be reinforced by tie rods or else.

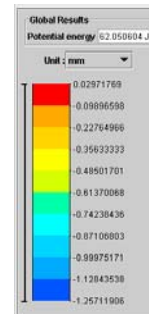
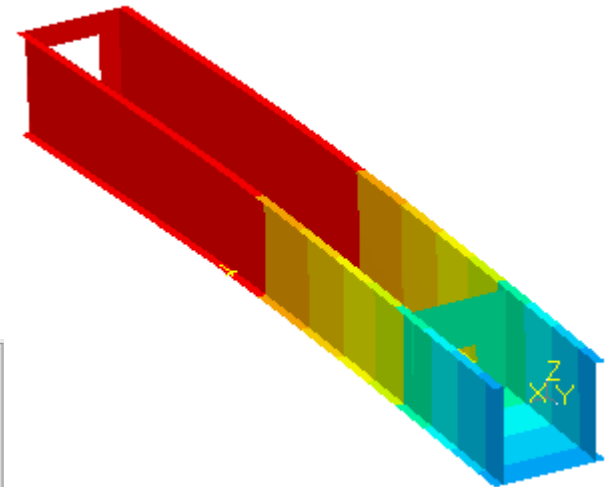
- Conditions
 - In both conditions (open/close)
 - Stainless Steel ($E = 200 \text{ GPa}$)
- Vertical deformation for 50mm thick

Endcap closed



Max def about 100 μm

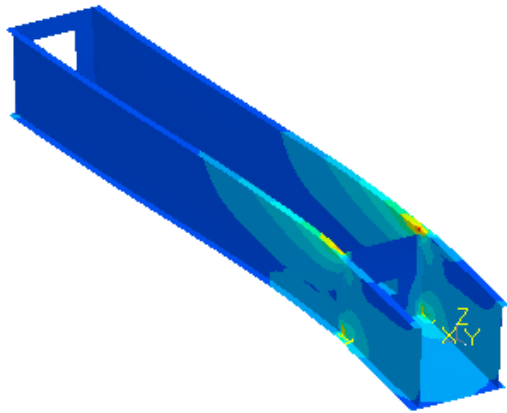
Endcap opened



Max def about 1.25mm

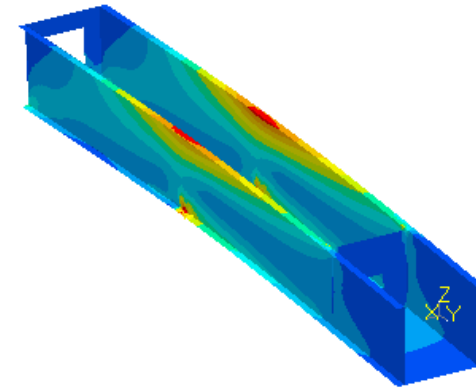
- Stress for 50mm thick

Endcap closed



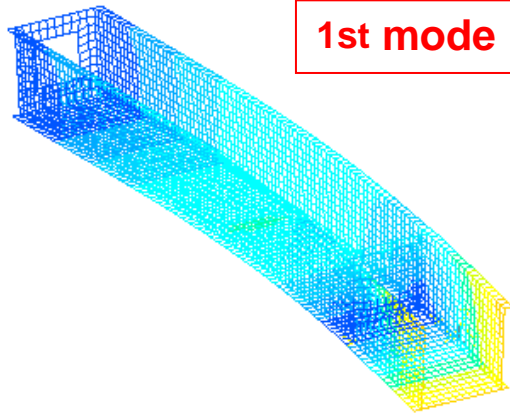
Max stress 10MPa

Endcap opened

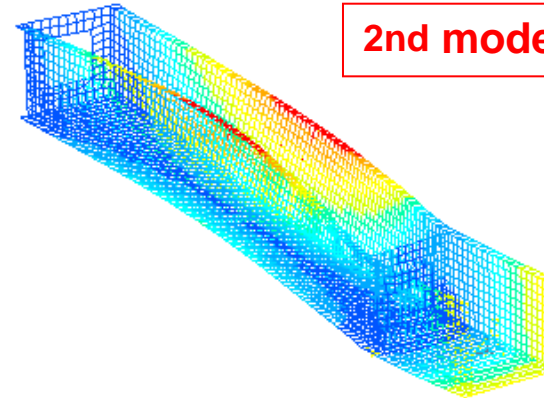


Max stress 16MPa

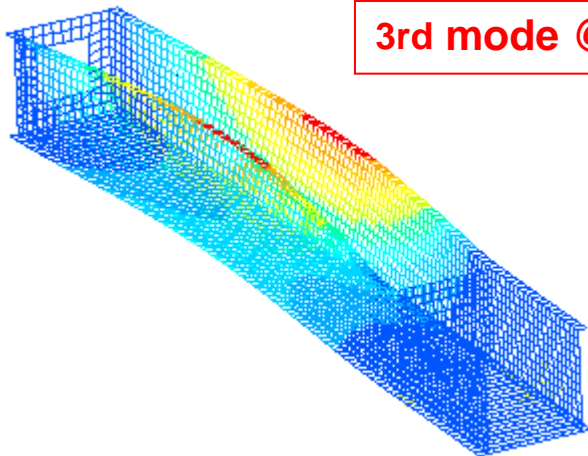
- For 50mm thick and Endcap closed



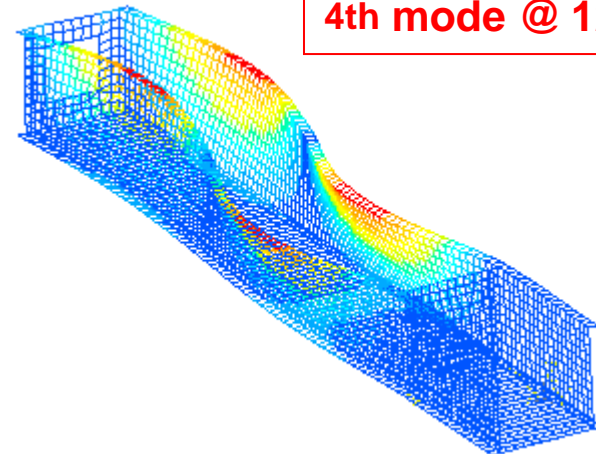
1st mode @ 44Hz



2nd mode @ 60Hz

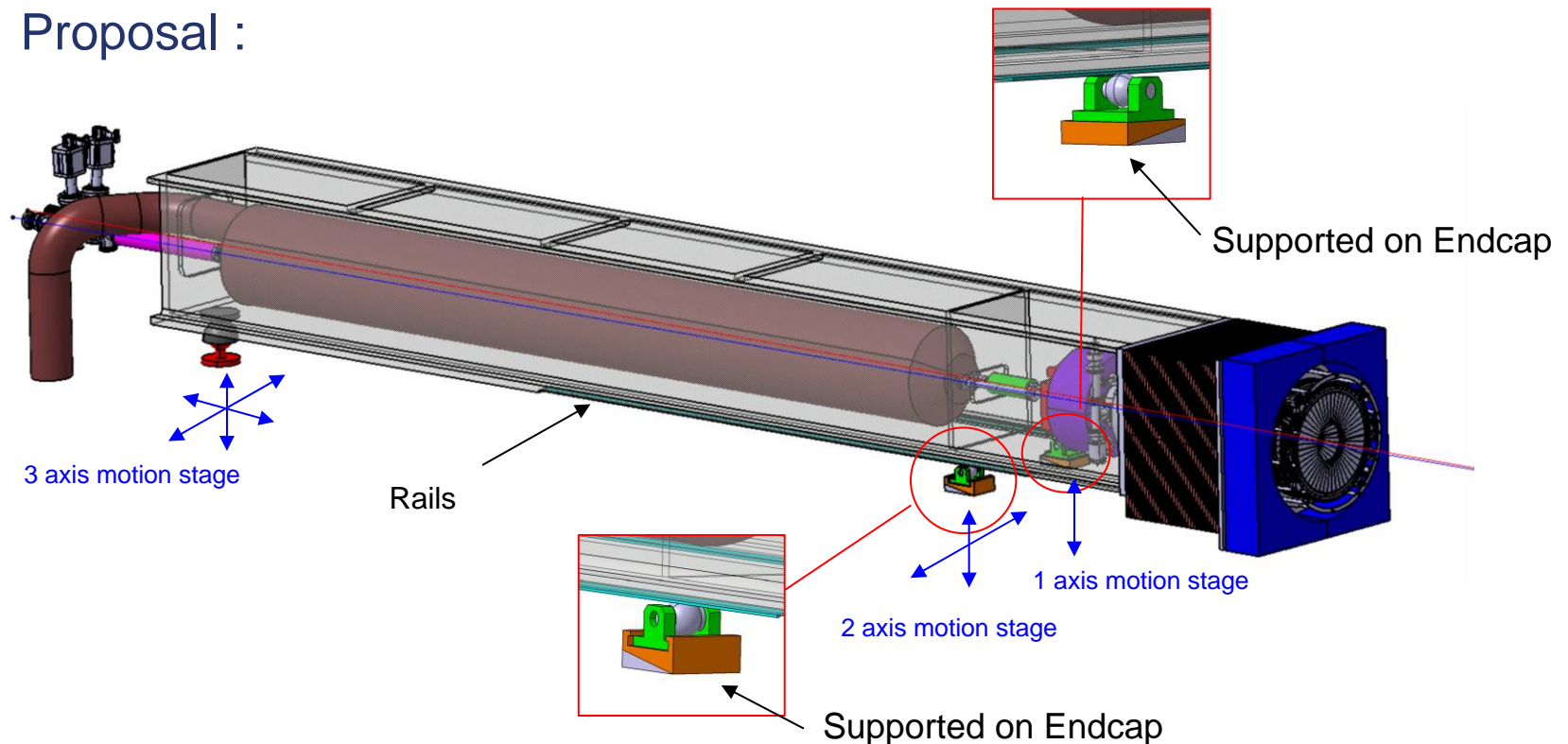


3rd mode @ 71Hz



4th mode @ 125Hz

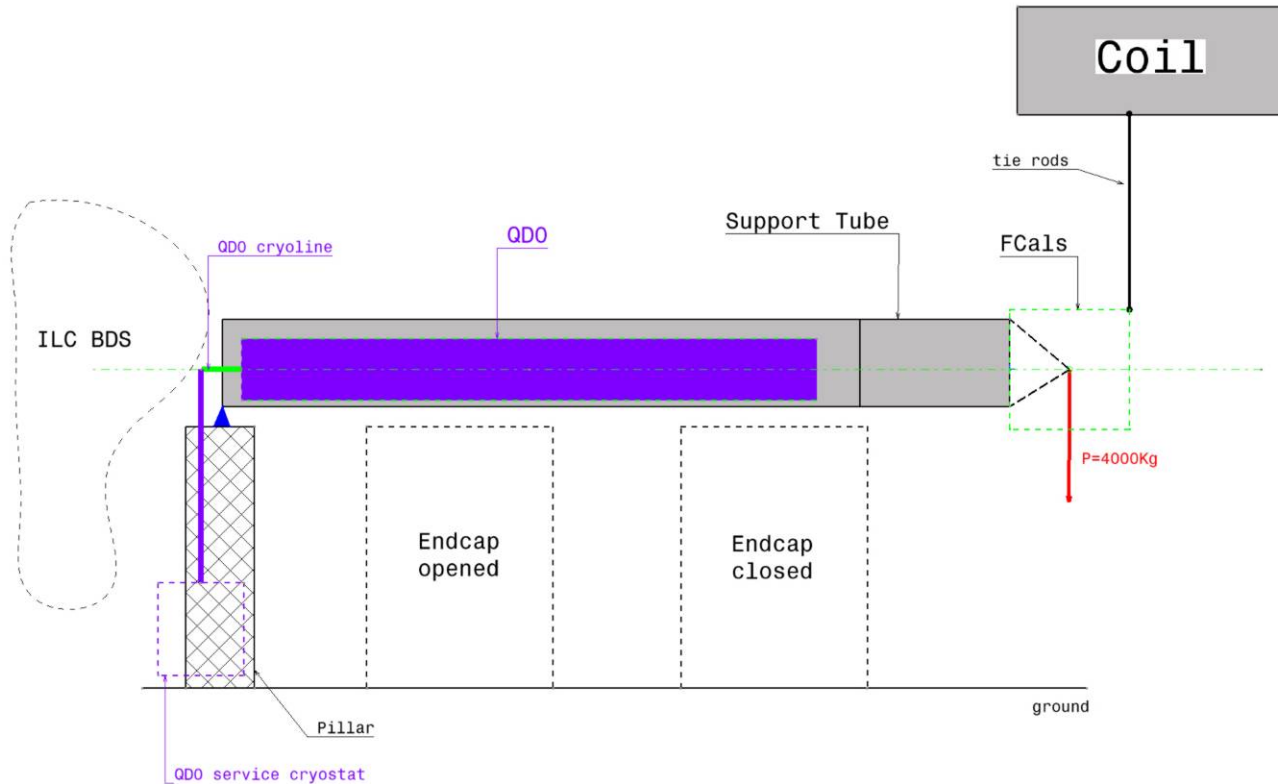
- Need to adjust the position in a mm range
- Choose of an isostatic devide
 - **Allow relative movement of Endcap**
 - **Allow to relax tolerance**
- Proposal :



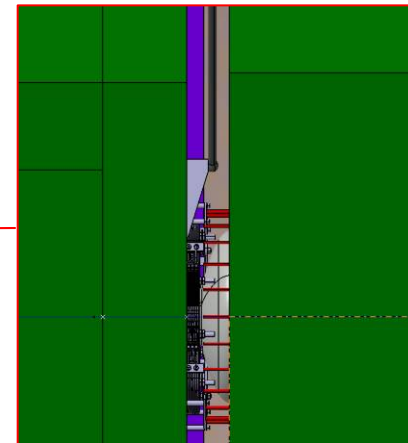
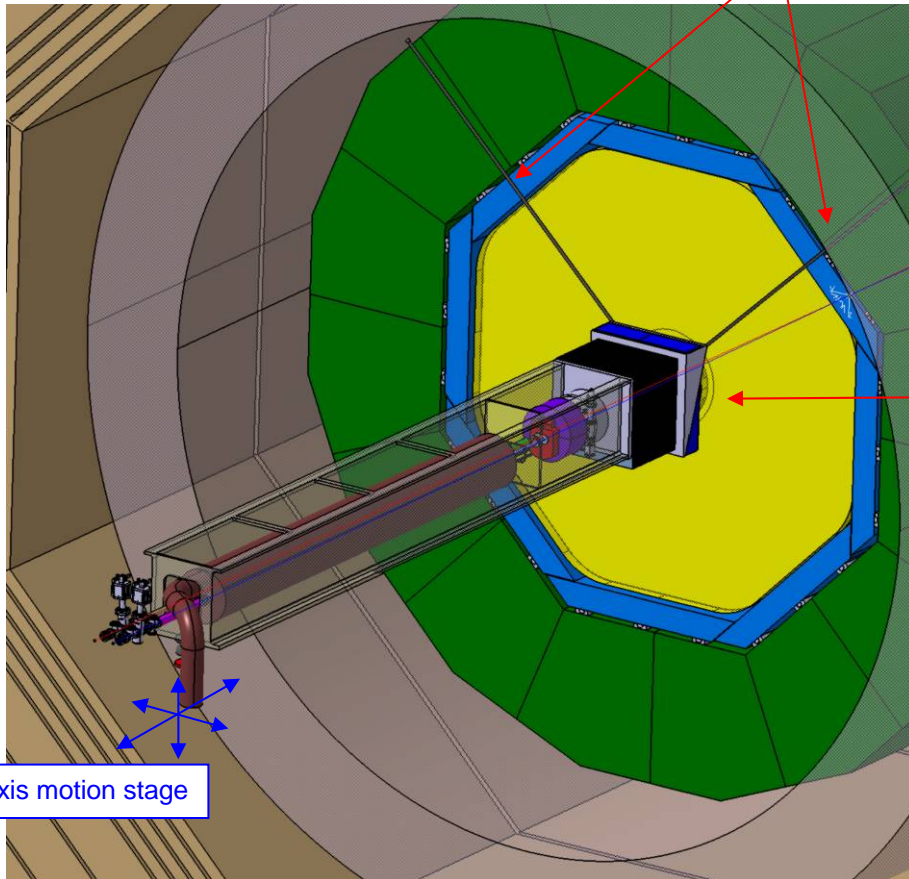
- Many pros
 - **Deformation / stress : OK**
 - **Natural frequencies : OK**
 - **Ground motion calculation under calculation but seem OK**
 - **We can correct & adjust the position of support tube**
 - **No material in front of Calorimeter**

 - **BUT**
 - **Need a complex system to align**
 - **Deformation is a function of the Endcap's position**
 - Problems on the forward component
 - On beam tube
- ⇒ we must decouple Endcap and support tube

- Supported by
 - Pillar from the ground (like solution 1)
 - Tie rods from the magnet to the FCals



Adjustable tie rods



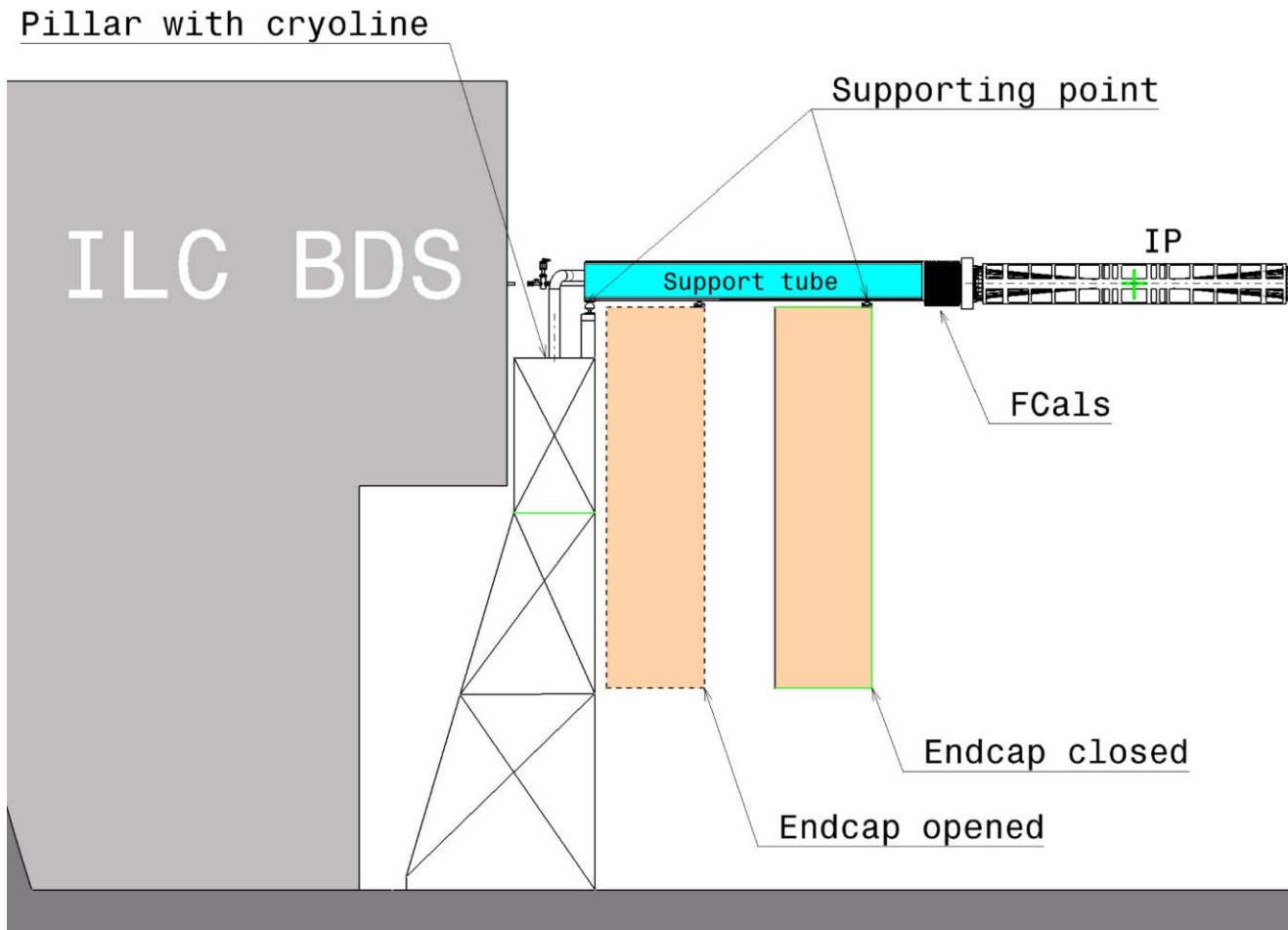
3 axis motion stage

- Some cons :
 - **Add material in front of Forward calor**
(Possible to use Carbon Fiber for tie rods)
 - **Need calculations to optimise the link to support tube**
 - **Interaction with Ecalring dimension**
- But many pros :
 - **Better stability** *(fixed at extremity of the tube)*
 - **Completely independent to the EndCap**
 - **Easier to adjust with the tie rods**
 - **No need of active correction**
 - **Less deformation & stress** *(results may come)*

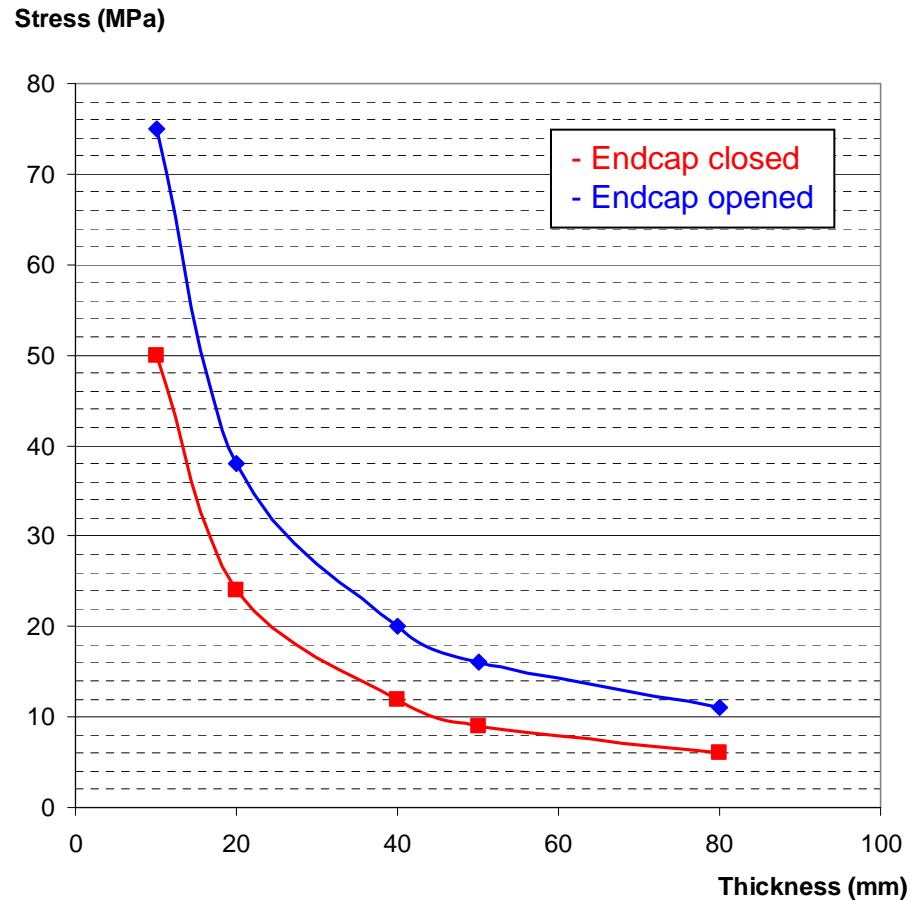
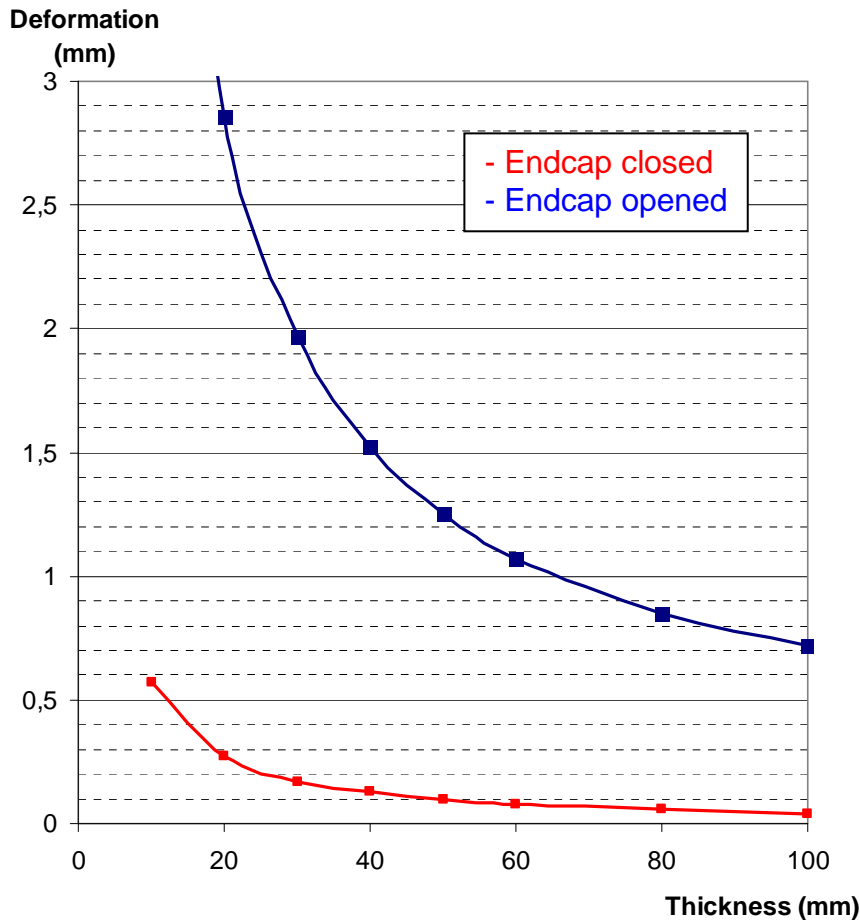
- On using square tube
 - **Stiffer than round (ab. 40%)**
 - **More place for integrating components**

- On supporting method
 - **Each solution has it's own pros and cons**
 - **But I think that the 2nd one is better for mechanical reason :**
 - Less deformation (calculation would be performed)
 - More stability
 - No active correction of alignment
 - ...
 - **Do you agree put some material budget in this area ?**

Extra slides



- Stress & deformation as function as thickness



- Natural frequency mode = $f(\text{thickness})$

