Transverse Beam Stability and Feedback

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Sources of Transverse Beam Motion

- A number of sources for transverse beam motion exists
 - ground motion
 - technical noise
 - jitter amplification by mechanical supports
 - RF gradient and phase jitter and dispersion
 - beam jitter from upstream systems
 - dynamic magnetic field variations
 - temperature variations

- . . .

- Not all are due to the technical installation
 - \Rightarrow beam stability is site dependent
 - \Rightarrow develop beam stabilisation techniques and use what a given site requires

1

Tools to Reduce Beam Motion

- Choose a quiet site
 - e.g. the LEP/LHC tunnel is relatively quiet
- Avoid technical noise
 - identify sources of noise and modify their design if possible
- Avoid amplification of vibrations through supports etc.
 - careful girder design
- Use mechanical feedback and feedforward
- Use motion sensor based feedforward on the beam
- Use beam-based feedback
 - mainly using BPM signals

Strategy to Evidence Beam Stability for CDR

- Perform integrated simulation of main linac, beam delivery system and collision including
 - RF phase and amplitude jitter
 - a realistic model of the ground motion and technical noise
 - realistic transfer through supports, including mechanical feedback
 - realistic sensitivity curves and noise for ground motion sensors for beam-based feedforward
 - a realistic concept of the beam-based feedback
- Have an integrated simulation of main linac, BDS and beam-beam interaction
 - PLACET, benchmarked with LIAR, MAD, Merlin, Lucretia, SLEPT etc., tested at CTF3
 - GUINEA-PIG, benchmarked with CAIN

Feedback

- Can use a simplified treatment for understanding
- Luminosity loss is given by

$$\Delta \mathcal{L} \approx \Delta \mathcal{L}_{uncorrected}(g) + \Delta \mathcal{L}_{noise}(g) + \Delta \mathcal{L}_{residual}(t)$$

 $\Delta \mathcal{L}_{uncorrected}(g)$: loss not yet corrected due to feedback delay $\Delta \mathcal{L}_{noise}(g)$: loss due to noise introduced by feedback (e.g. BPM resolution)

 $\Delta \mathcal{L}_{uncorrected}(t):$ residual loss that the feedback does not correct by design

• We will eventually use a full model of the machine and feedback in the beam simulations

IP Feedback/Feedforward Conceptual Layout



- Currently the following feedback/feedforward systems are foreseen
 - a mechanical feedback for the quadrupoles (ground motion sensors on quadrupoles+actuators)
 - an intra-pulse beam-based feedback (BPMs+kickers)
 - a pulse-to-pulse beam-based feedback system (BPMs+kickers)
 - a feed-forward system based on ground motion sensors using the kickers to move the beam
- \bullet Beam-beam jitter tolerance $0.3\,nm$ for 2% loss

Example of Mechanical Feedback and Noise

- A cantilever with feedback on the stabilisation table
 - not the real mechanical design
 - impact of magnetic field and radiation on sensors remains to be studied
 - used for illustration only
 - ground motion is Annecy



Data from B. Bolzon et al., noise assumed to be realtime measurement noise (A. Jeremie)

Simplified Model

- Ignore incoming beam jitter
- Four independent point-like quadrupoles studied
 - correlations will help, correlation expected strong for micro-seismic peak, will change controller design
 - assume that measured stability is stability of whole quadrupole
- Home-made controller
 - serious study of controler design started in Annecy (B. Caron, L. Brunetti)



Integration with Mechanical and Beam-Based Feedback

Three controllers shown

Note: For Seryi model did not need differential term since low frequency motion is correlated

 $p_Q(\omega)$ power spectrum of beam-beam motion with mechanical feedback

 $T_B(\omega)$ transfer function of beam-beam feedback

 $p_N(\omega)$ noise spectrum of beam-beam feedback



• Expectation for beam-beam offset can be calculated as

$$\langle y^2 \rangle = \int_0^\infty T_B(\omega)^2 p_Q(\omega) + p_N(\omega) d\omega$$

Addition of Feedforward

- Geophone sensitivity function and noise used as example
 - but problematic in magnetic field
- Best controller shown, need to do more detailed analysis of sensor (will reduce performance)



 $\langle y^2 \rangle = \int_0^\infty T_B(\omega)^2 \left(T_{FF}(\omega)^2 p_Q(\omega) + p_{FF}(\omega) \right) + p_N(\omega) d\omega$

Example Sensor: Geophone

 Relevant is real-time measurement

 $T_{FF} = (1 - S(\omega))$

- This sensor will need to be protected from magnetic field or need to design new sensor
- If sensitivity function were $S(\omega) = 1$ would reduce beam-beam offset by factor 2



Sensor Frequency Choice

- Sensor choice is important to match motion spectrum
- Spectrum of relative motion is shown (over 12 m)
 - also motion at $50\,\mathrm{Hz}$
- ⇒ Geophone has not an optimum frequency response for ground motion model B



Cantilever Designs



12

Impact of Cantilever

- Combination of ground motion, mechanical stabilisation, beam feedforward (simplified), beam-beam feedback and cantilever is shown
- Note: based on earlier calculations update for new ones planed
- \Rightarrow The cantilver increases the quadrupole motion
 - but not very much if resonance freqeuncy is good



Ground Motion Models

- Some examples are shown
 - Annecy and CMS hall floor
 - models based on Andrei Seryi's measurements
- LEP/LHC tunnel is relatively quiet
- Model B has similar shape as Annecy or CMS hall floor
 - B10 if we amplify one peak by factor 10 agrees even better



Ground Motion Correlation

- Ground motion is correlated
- Correlation has an impact on the luminosity performance
 - e.g. relative offsets of final quadrupoles is important (relevant distance $\approx 12 \,\mathrm{m}$)
- \Rightarrow high frequency part is uncorrelated



Another Model for CERN Site

- Consider ground motion as combination of
 - ground motion model A
 - technical noise modeled as (Ch. Collette)

$$P(\omega) = P_0 \frac{1}{1 + \left(\frac{\omega}{\omega_0}\right)^6}$$

• Parameters vary as function of position



$$\omega_0 = 40\pi$$
 and $P_0 = 0.5 \times 10^{-18}$ used

 \Rightarrow Should be able to choose representative model(s)

Impact of Ground Motion

- Assumed a direct oneto-one transfer to beam line elements and simplified feedback
- Perfect stabilisation (air hook) is assumed at all frequencies
- Also multipoles in final doublet area are stabilised
- Note: in A stabilisation can increase luminosity loss as machine drifts away from stabilised magnets



⇒ A medium noisy site (B) is almost OK, if we stabilise the final doublets

Example: Impact of Quadrupole Stabilisation

 Assume stabilisation of all quadrupoles according to table in Annecy

- for illustration only

- Need to replace the transfer function of that table more realistic model
 - iteration with stabilisation



Technical Noise

- Assume that technical noise has litte correlation
 - \Rightarrow jitter of each element around its nominal position
- Use previous fit

 $P(\omega) = \frac{0.5}{1 + \left(\frac{\omega}{40\pi}\right)^6} \,\mathrm{nm^2/Hz}$

- RMS offset can be calculated as $\langle y^2 \rangle = \int_0^\infty \left(P(\omega) T_S^2(\omega) + N_S(\omega) \right) T_B^2(\omega) d\omega$ used no stabilisation for plots
- \Rightarrow Stabilisation needs to provide $T_S(\omega)$ and $N_S(\omega)$ to reduce $\sqrt{\langle y^2\rangle}$ to specification



Main Linac Quadrupole Offset Tolerance

- Full simulation of main linac and BDS
- \Rightarrow The multi-pulse emittance is a good measure of the luminosity loss
- \Rightarrow The collimator wakefields add somewhat to the luminosity loss
 - Tolerance for 1 % dynamic luminosity loss is ≈ 1.3 nm



B. Dalena

Beam Delivery System QuadrupoleOffset Tolerance



J. Snuverink

Main Linac Feedback Design

- Use 40 corrector pairs and 41 sets of eight BPMs
- Upper plot shows reduction of emittance bue to feedback
- \bullet Lower plot shows multi-pulse emittance induced by $100 \ nm$ BPM resolution
 - \Rightarrow can run with full proportional gain



Beam Delivery System Feedback Design

- Design is under development
- First results using a fewto-few correction show that BPM resolution should be better than 30 nm for fast feedback



Intra-Pulse Interaction Point Feedback

- Simple beam-beam feedback based on deflection angle at IP
- \bullet Assuming 37 ns latency one can hope for factor 2 gain in tolerance
- \bullet Only cures offsets, $\,\mu m$ BPM resolution is sufficient, but large aperture

Collaboration with JAI

- 4.5 4 iteration 3 iteration L $[10^{34} \, \mathrm{cm^{-2} \, s^{-1}}]$ 3.5 2 iteration 3 37 ns 2.5 74 bunches 156 ns train 1.5 0.5 iteration $0^{\scriptscriptstyle L}_0$ 25 50 75 100 125 150 175 200 time [ns]
 - Thanks to Javier Resta Lopez



Main Linac and BDS Mechanical Feedback/Feed-Forward

- In the main linac and BDS ground motion sensor based beam feed-forward can be used
- Aim is to make the system cheaper
 - no mechanical feedback on quadrupoles
 - measurement of quadrupoles motion
 - correction by orbit correctors
- Requires is good system knowledge

 \Rightarrow Juergen's thesis

- More challenging than the local mechanical stabilisation but could be less costly
 - \Rightarrow could be an alternative described in CDR

Conclusion

- Beam-based feedback design is under development
 - need to complete the BDS feedback
- Need more input from and interaction with the stabilisation group
 - noise sources
 - mechanical design and feedback
 - sensors
- Controler design started and needs continuation
 - integration of stabilisation and beam physics
- Integration is making progress
 - but quite a way to go
- Exploration of other beam jitter sources
 - e.g. stray fields study will start at CERN

Reserve

Dynamic Imperfections

- Luminosity loss is part of the emittance budget
- \bullet But limit luminosity fluctuation to less than 10%
 - total luminosity fluctuation is not straightforwad

Source	budget	tolerance
Damping ring extraction jitter	0.5%	kick reproducibility $0.1\sigma_x$
Transfer line stray fields	?%	data needed
Bunch compressor jitter	1%	
Quadrupole jitter in main linac	1%	$\sigma_{jitter} \approx 1.8 \mathrm{nm}$
RF amplitude jitter in main linac	1%	0.075% coherent, $0.22%$ incoherent
RF phase jitter in main linac	1%	0.2° coherent, 0.8° incoherent
RF break down in main linac	1%	$rate < 3 \cdot 10^{-7} m^{-1} pulse^{-1}$
Structure pos. jitter in main linac	0.1%	$\sigma_{jitter} pprox 880 \mathrm{nm}$
Structure angle jitter in main linac	0.1%	$\sigma_{jitter} \approx 440 \mathrm{nradian}$
Crab cavity phase jitter	2%	$\sigma_{\phi} \approx 0.017^{\circ}$
Final doublet quadrupole jitter	2%	$\sigma_{jitter} \approx 0.17(0.34) \mathrm{nm}$ -
		$0.85(1.7)\mathrm{nm}$
Other quadrupole jitter in BDS	1%	
	?%	

 \Rightarrow Long list of small sources adds up

 \Rightarrow Impact of feedback system is important

Comment on Magnetic Field Stability

- The magnet has different oscillation modes
- The external vibration of each is not necessarily identical with the field vibration
- Feed-forward will help
 - can identify the modes with the sensors
 - can determine the correlation with beam motion experimentally in situ
 - can use feed-forward to compensate additional magnetic motion
- Also would want to have the freedom to include feed-forward from direct ground measurement before mechanical feedback

Available Signals at Interaction Point

- The IP feedback/feed-forward system controls the beam-beam offset
- Available beam signals are for each beam pulse
 - the beam-beam deflection from the post-collision BPMs
 - the incoming beam jitter from the pre-collision BPMs
 - the incoming beam offset from the pre-collision BPMs
 - other beam-beam signals (energy loss, coherent and incoherent pairs,
 ...)
- Other available signals from ground motion sensors are the mechanical motion
 - of the ground from ground
 - of the quadrupole support
 - of the final quadrupoles

Interaction Point Feedback Design

- Currently the following feedback/feedforward systems are foreseen
 - a mechanical feedback for the quadrupoles (ground motion sensors on quadrupoles+actuators)
 - an intra-pulse beam-based feedback (BPMs+kickers)
 - a pulse-to-pulse beam-based feedback system (BPMs+kickers)
 - a feed-forward system based on ground motion sensors using the kickers to move the beam
- More complex systems need to be integrated but not on CDR timescale
 - etc. waist shift correction with beamstrahlung monitors
- Beam-beam jitter tolerance 0.3 nm for 2% loss



Proposed Conceptual Layout



- Sensors are used for mechanical feedback
- Feed-forward kicker does not need to be identical with intra-pulse feedback kicker
- Expected beam-beam offset due to quadrupole slice offsets δ_i and kicker strength k can be calculated via

$$\Delta y = -a(k_{ff} + k_b) + \sum_i b_i \delta_i$$

- Choose k_{ff} such that $\Delta y = 0$ is expected
 - \Rightarrow final beam motion is determined by sensor noise
 - and imperfections in system knowledge

Simplified Model

- Ignore incoming beam jitter
- Four independent point-like quadrupoles studied
 - correlations will help, correlation expected strong for micro-seismic peak, will change controller design
 - assume that measured stability is stability of whole quadrupole
- Quadrupole stabilisation feedback and beam feed-forward modelled by using sensor noise
- Beam-based feedback adds kicker strength k_b
- Simple home-made controller used:

$$k_b(n) = g_i k_b(n-1) + g_p \frac{\Delta y(n-1)}{a} + g_{d2} \left(k_b(n-1) - k_b(n-2) \right) + g_d \left(\frac{\Delta y(n-1)}{a} - \frac{\Delta y(n-2)}{a} \right)$$

Beam Feedback Transfer Function

- Control on velocity cures low frequency perturbations better but causes more amplification at high frequencies
- Serious study of controler design started in Annecy (B. Caron, L. Brunetti)



Main Linac Feedback Performance

- The multi-pulse emittance growth as a function of the feedback gain is calculated
 - corresponds to
 - $\mathcal{L}_{uncorrected}(g)$
 - primitiv controler used

