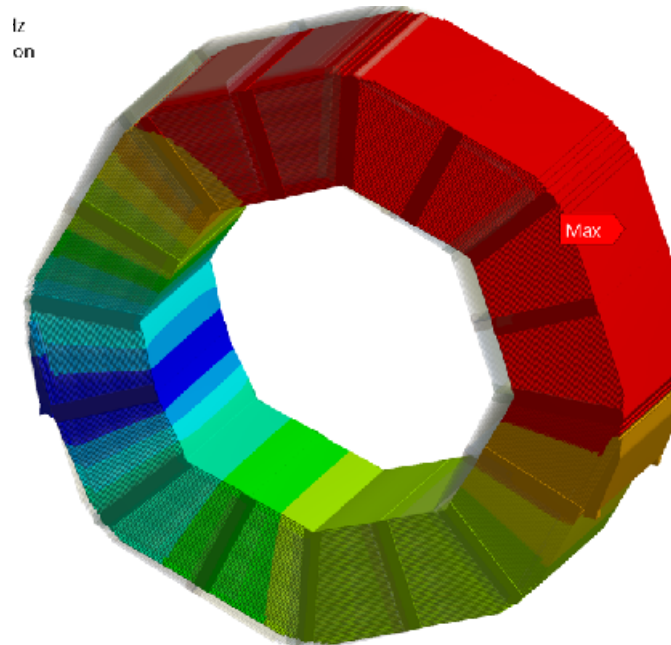


Methods for a seismic stability validation of the AHCAL structure

Karsten Gadow, Martin Lemke, Felix Sefkow



E-JADE



Mini-Workshop on ILC Infrastructure and CFS for Physics and Detectors
KEK, Tsukuba, September 28, 2017

Outline

- Previous studies
- Sub-structuring method
- Validation with a toy model

Static deformations

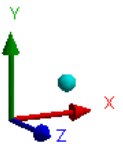
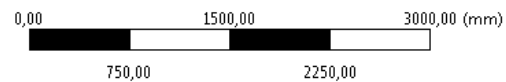
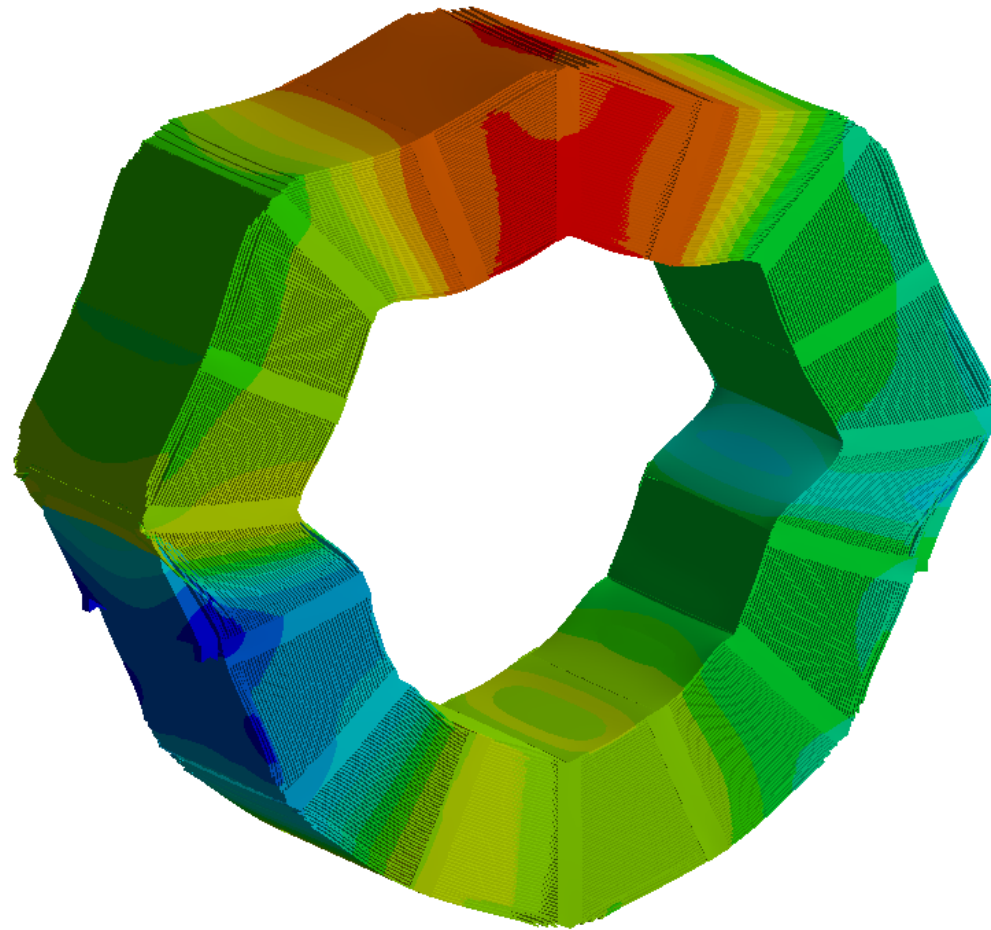
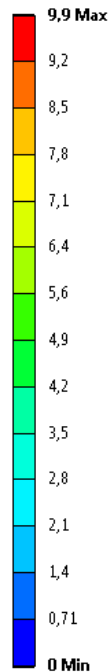
J: Model, Static Structural

Figure

Type: Total Deformation

Unit: mm

Time: 1



- outdated version
 - plates now thicker

Local stress peaks

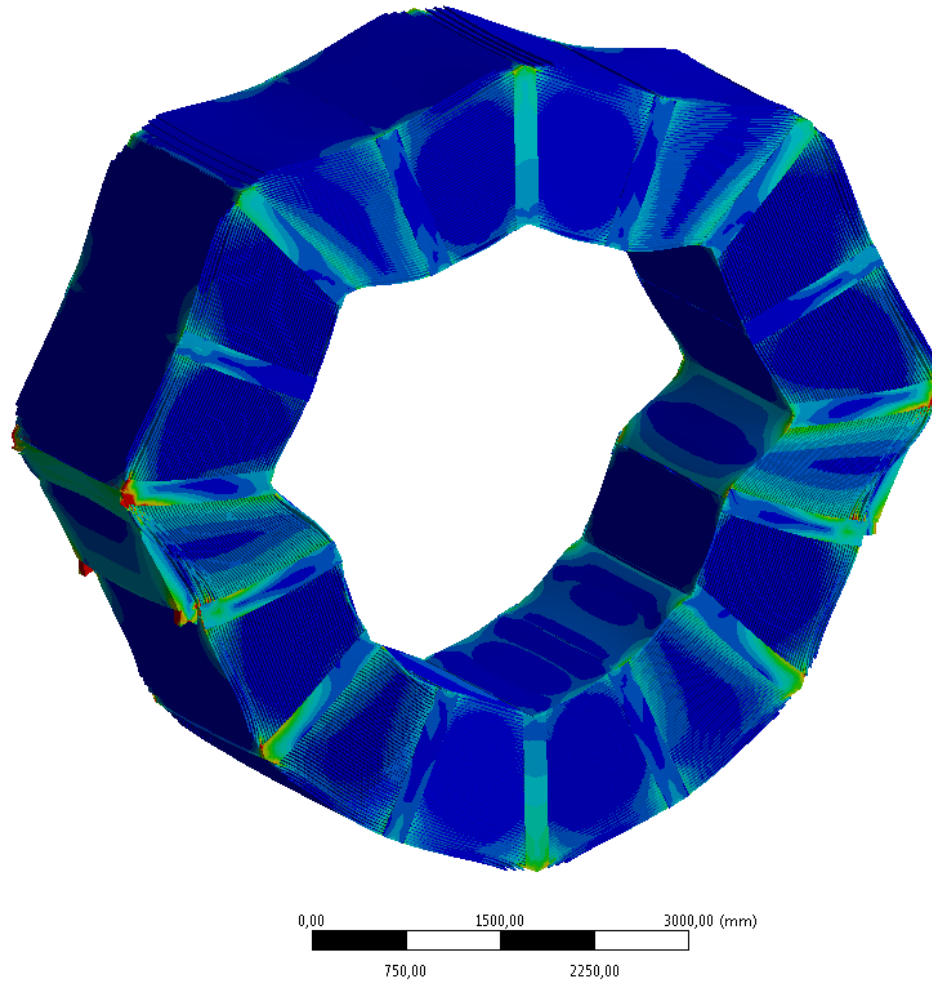
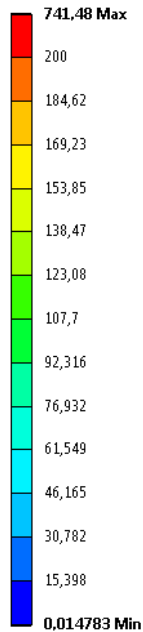
J: Model, Static Structural

Figure

Type: Equivalent (von-Mises) Stress - Top/Bottom

Unit: MPa

Time: 1



- local analysis to be refined, structure to be optimised

Local stress peaks

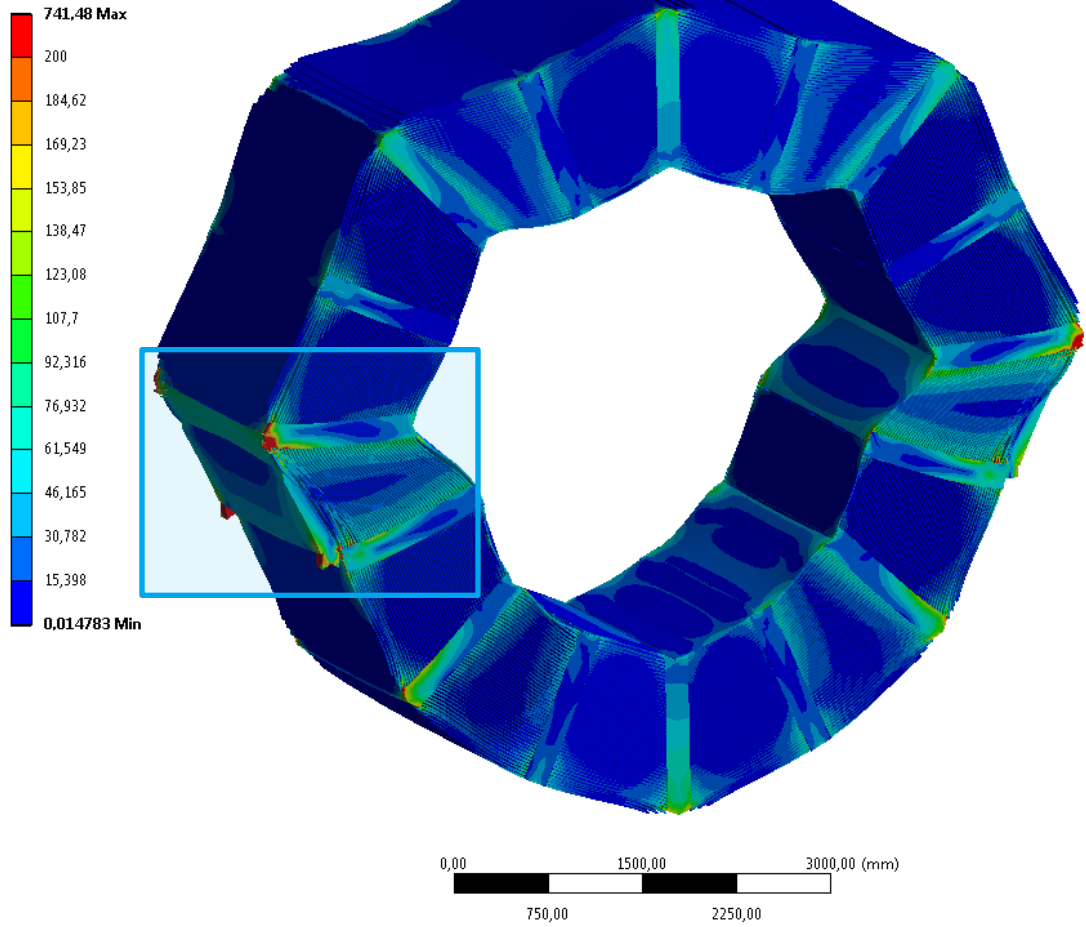
J: Model, Static Structural

Figure

Type: Equivalent (von-Mises) Stress - Top/Bottom

Unit: MPa

Time: 1



- local analysis to be refined, structure to be optimised

Local stress peaks

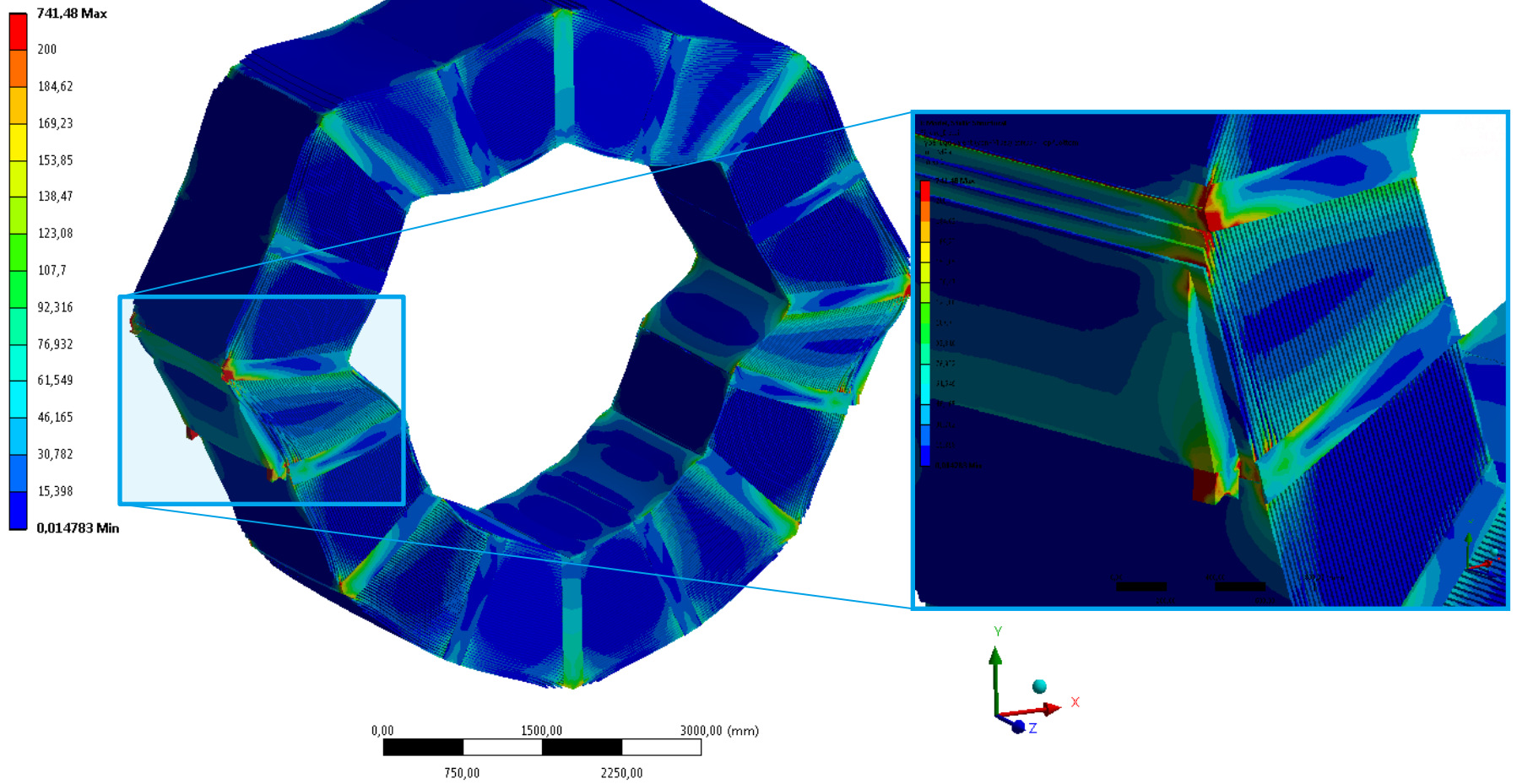
J: Model, Static Structural

Figure

Type: Equivalent (von-Mises) Stress - Top/Bottom

Unit: MPa

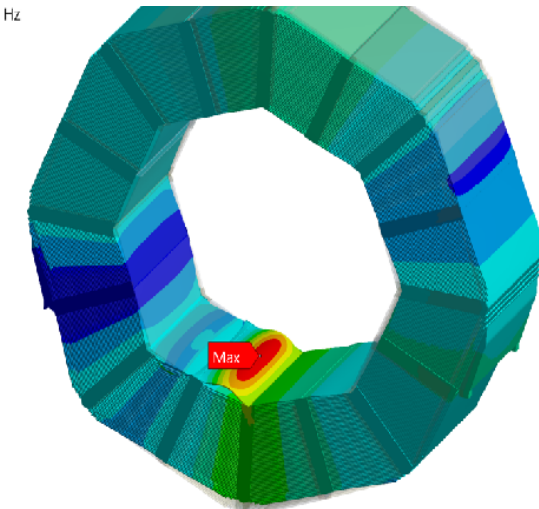
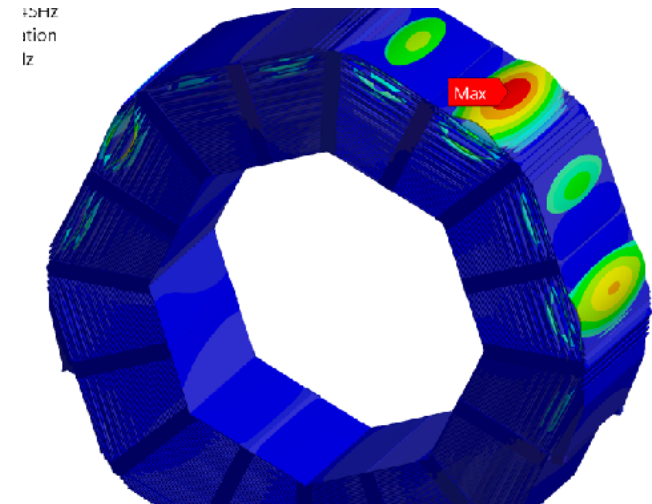
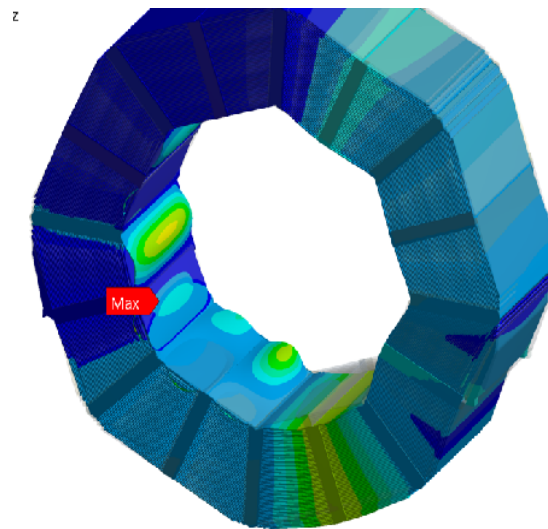
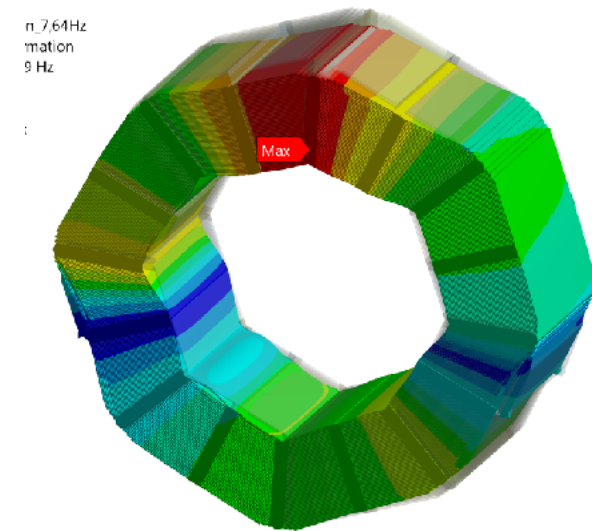
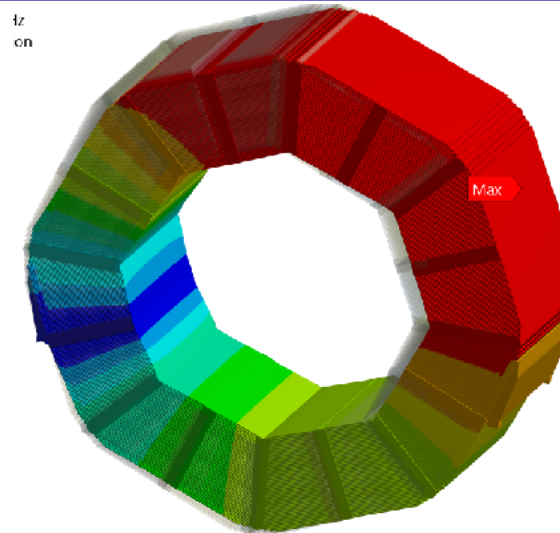
Time: 1



- local analysis to be refined, structure to be optimised

Eigen mode analysis

- > Swinging barrel: 3Hz
- > Swinging module: 8Hz
- > Swinging plate: 6Hz
- > Higher modes: 15 Hz
- > Several plates: 45 Hz

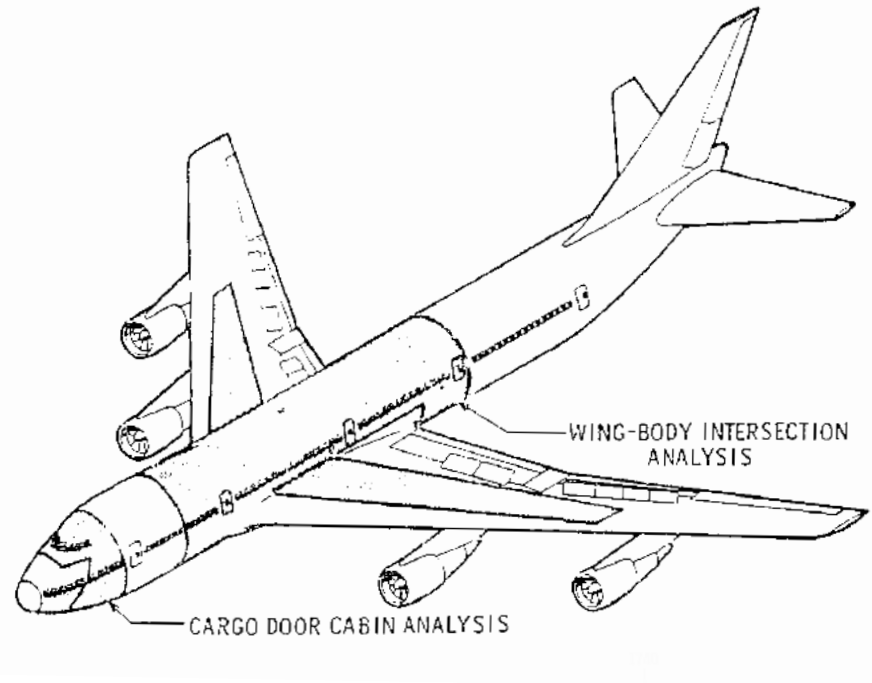
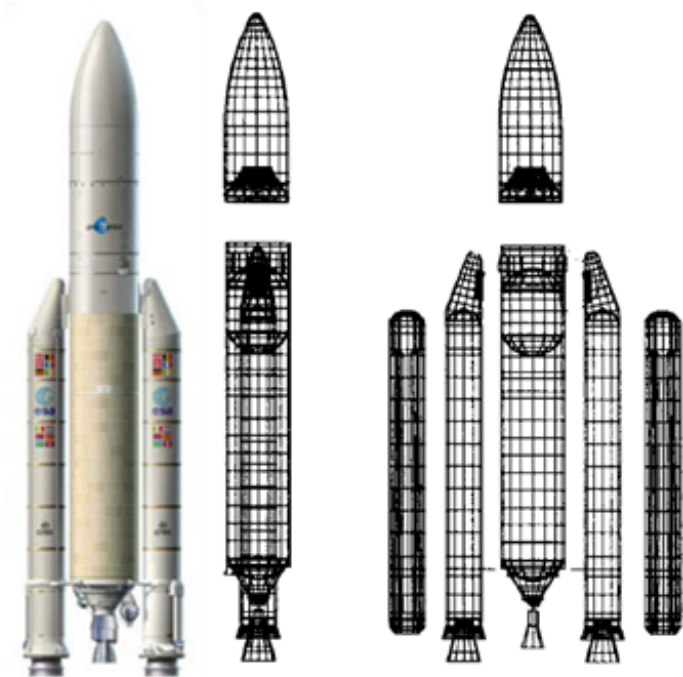


Computational challenges

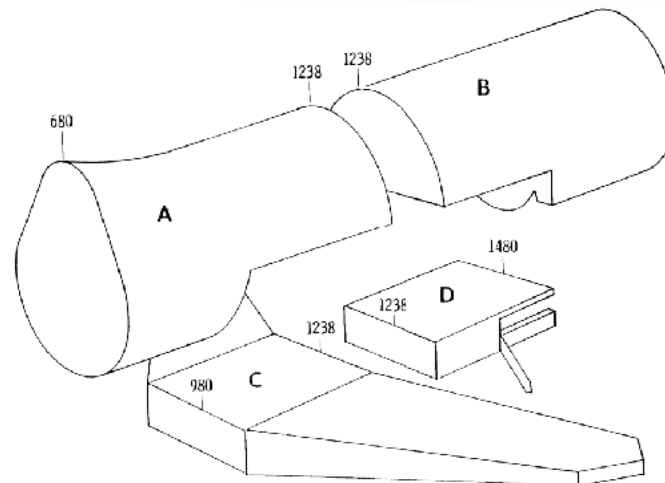
- Within reasonable effort, first 200 eigen modes calculated
- In order to obtain response spectrum with a frequency sweep, need to introduce damping: further complication
- Computation failed

- Possibilities to simplify:
 - omit details: use shells, beams, point masses, rigid bodies
 - loss of realism and predictive power
- More efficient approach: **sub-structured analysis**
 - condense group of elements into a “super-element”
 - model behaviour for the overall structure in a matrix describing the characteristic properties of the super-element
 - rigidity matrix exact, mass and damping matrix approximative

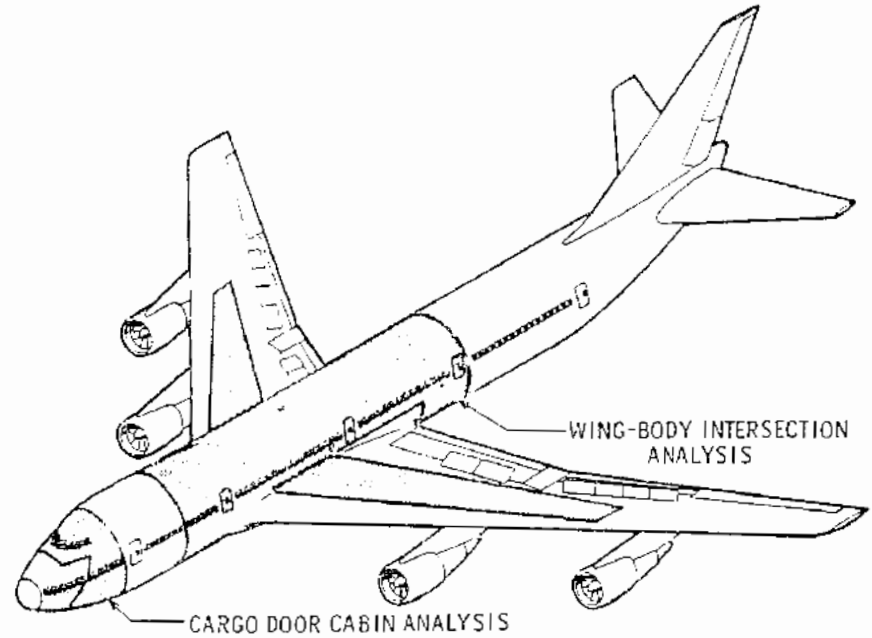
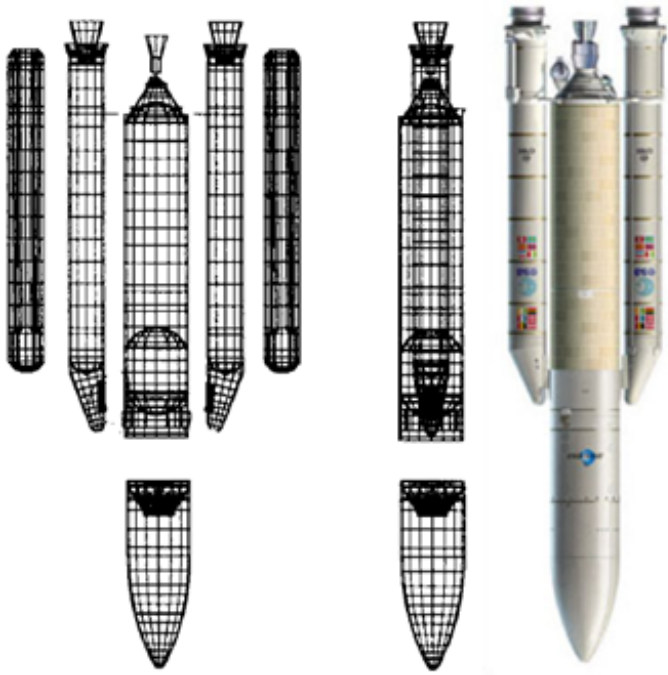
Examples



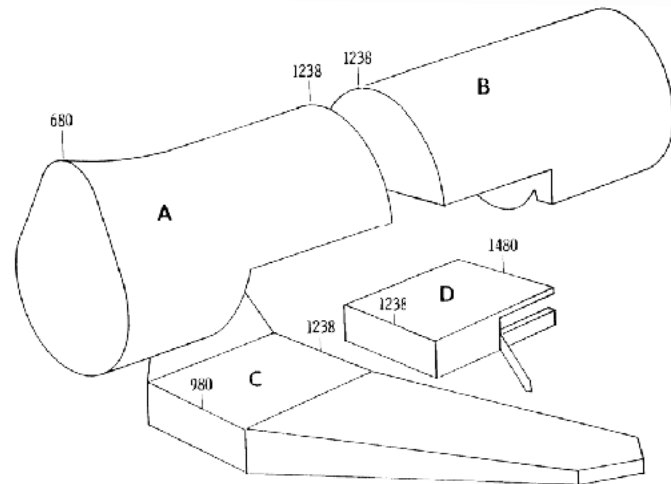
- Method commonly used in aerospace and automotive engineering since 1970s



Examples

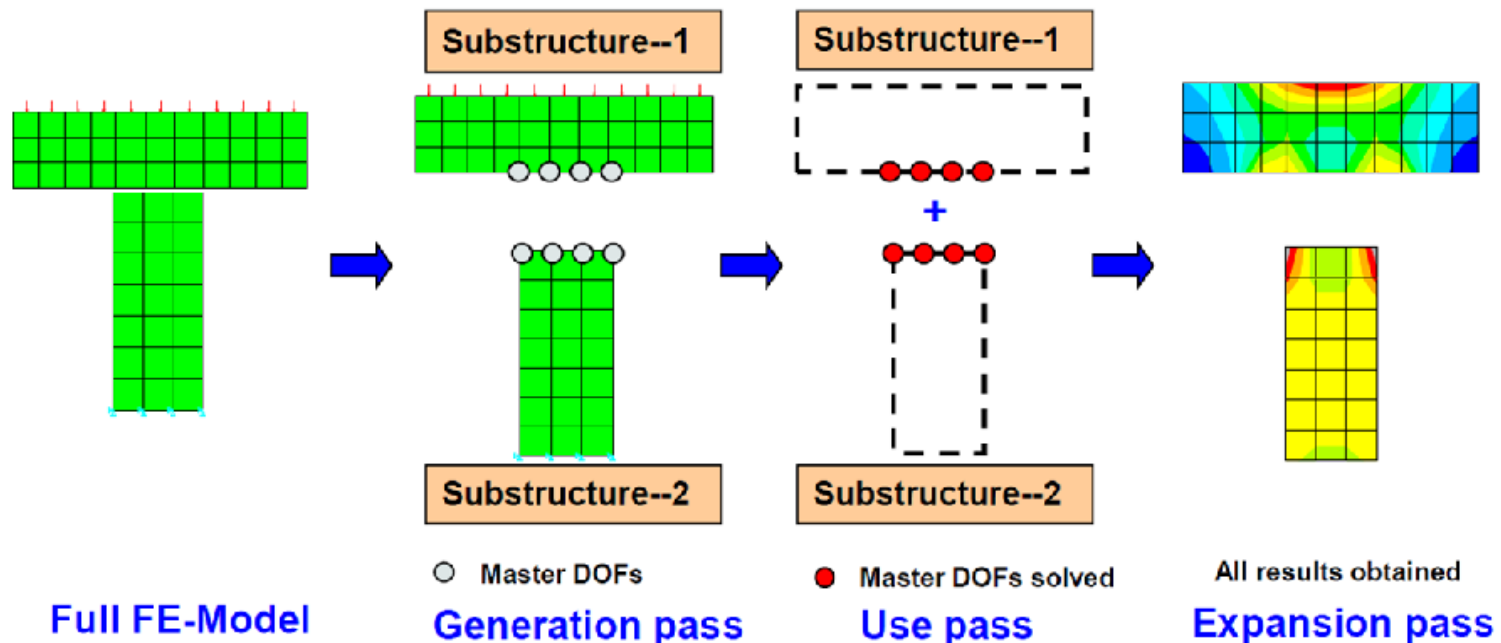


- Method commonly used in aerospace and automotive engineering since 1970s



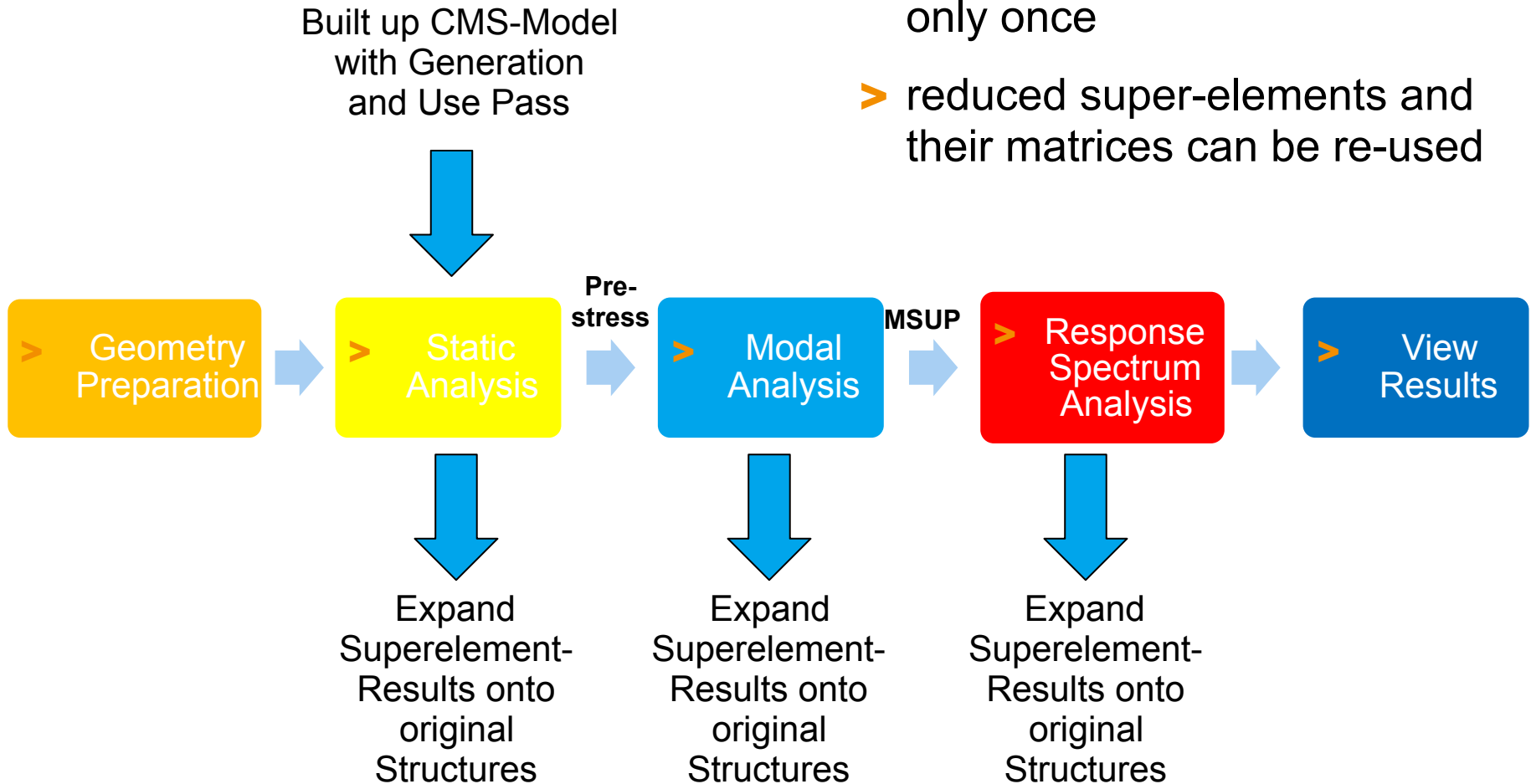
Sub-structured analysis method

- "Component mode synthesis
- Using ANSYS parametric design language APDL
- Generation pass: calculate matrices and master degrees of freedom (MDOF) at super-element boundaries
- Use pass: integrate full structure, using MDOFs
- Expansion pass: back-propagate results into super-elements



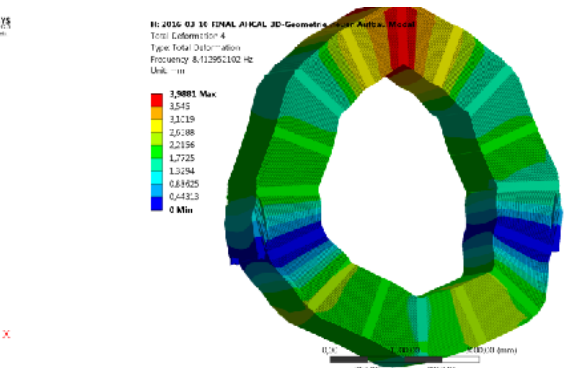
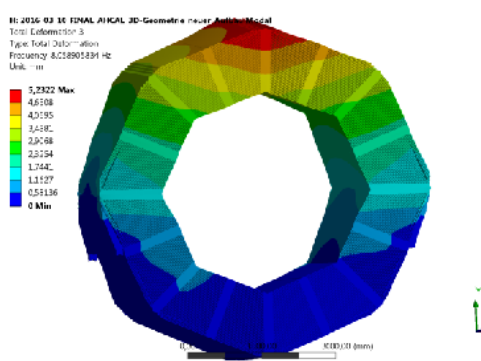
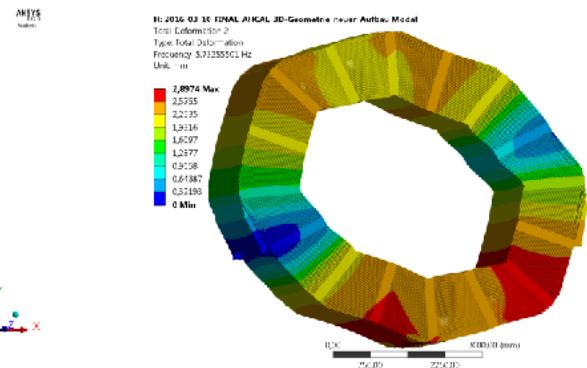
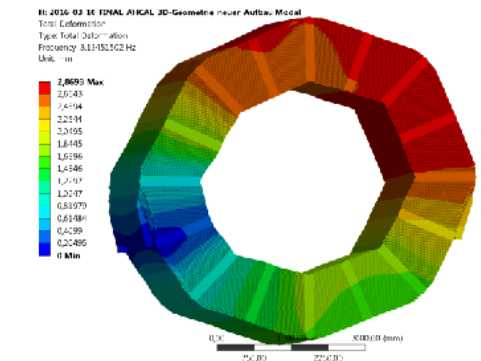
Substructuring – Implementation

- > Generation pass must be done only once
- > reduced super-elements and their matrices can be re-used



➤ Results CMS-Model:

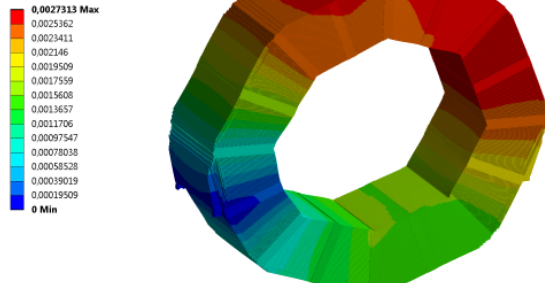
Shell-Model (prestressed)		Shell-Model (free)		CMS-Model (free)	
Nr.	f [in Hz]	Nr.	f [in Hz]	Nr.	f [in Hz]
1	2,97	1	2,83	1	3,13
2	5,27	2	5,18	2	5,73
3	6,11	3	7,07	3	8,06
4	7,65	4	7,46	4	8,41
5	9,16	5	9,94	5	10,84
6	9,85	6	11,60	6	12,91
7	11,68	7	13,65	7	15,02
8	13,32	8	14,70	8	16,47
9	14,65	9	15,37	9	16,83
10	14,67	10	17,18	10	18,07
11	15,76	11	18,75	11	20,38
12	17,37	12	19,29	12	21,14
13	18,32	13	20,21	13	22,44
14	19,37	14	21,46	14	23,34
15	20,29	15	22,48	15	24,44
16	21,99	16	23,63	16	26,12
17	22,83	17	25,52	17	27,06
18	24,05	18	31,37	18	32,28
19	24,49	19	33,39	19	38,16
20	25,23	20	35,12	20	39,50
21	31,22	21	41,07	21	41,03
22	35,29	22	42,68	22	41,03
23	38,82	23	42,68	23	41,03
24	39,76	24	42,72	24	41,03
25	40,52	25	42,72	25	41,03
26	40,55	26	42,72	26	41,03
27	41,00	27	42,72	27	41,03
28	41,27	28	42,72	28	41,04
29	42,32	29	42,81	29	41,04
30	43,78	30	43,33	30	41,04



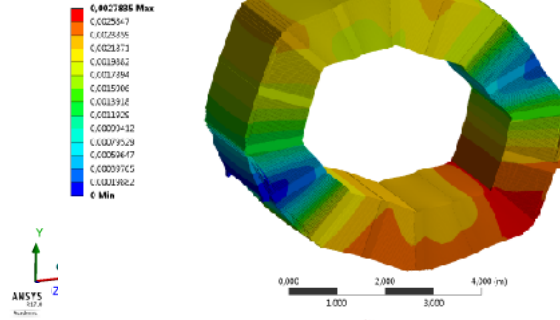
General AHCAL-Model

➤ Results 3D-Model:

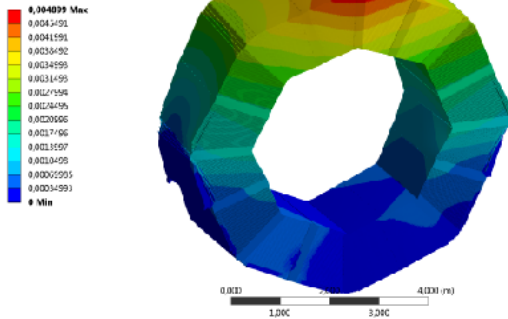
I: 2016 03 11 Shell-Model Modal
Total Deformation
Type: Total Deformation
Frequency: 2.826279326 Hz
Unit: m



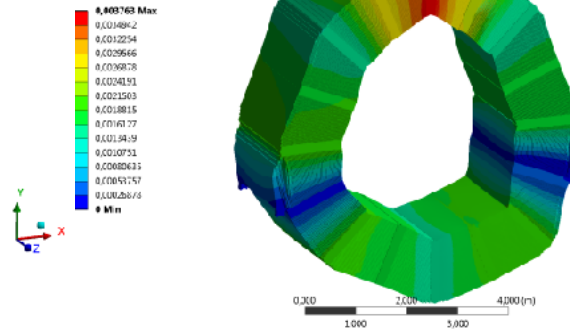
I: 2015 03 11 Shell-Model Modal
Total Deformation
Type: Total Deformation
Frequency: 5.140519060 Hz
Unit: m



E: 2030 03 11 Shell-Model Modal
Total Deformation
Type: Total Deformation
Frequency: 7.007708938 Hz
Unit: m



A1: Shell-Model Modal
Total Deformation
Type: Total Deformation
Frequency: 7.150111106 Hz
Unit: m

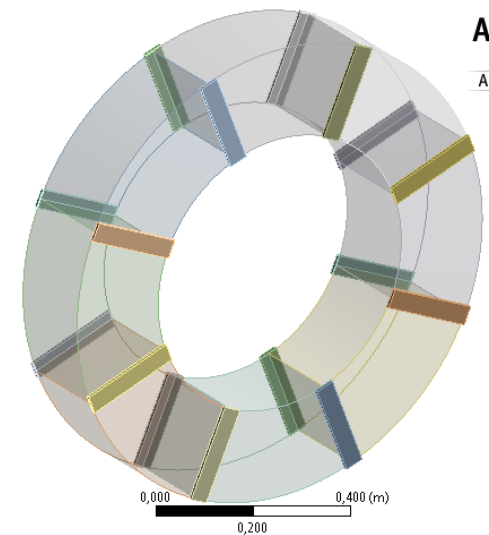


Shell-Model (prestressed)		Shell-Model (free)		CMS-Model (free)	
Nr.	f [in Hz]	Nr.	f [in Hz]	Nr.	f [in Hz]
1	2,97	1	2,83	1	3,13
2	5,27	2	5,18	2	5,73
3	6,11	3	7,07	3	8,06
4	7,65	4	7,46	4	8,41
5	9,16	5	9,94	5	10,84
6	9,85	6	11,60	6	12,91
7	11,68	7	13,65	7	15,02
8	13,32	8	14,70	8	16,47
9	14,65	9	15,37	9	16,83
10	14,67	10	17,18	10	18,07
11	15,76	11	18,75	11	20,38
12	17,37	12	19,29	12	21,14
13	18,32	13	20,21	13	22,44
14	19,37	14	21,46	14	23,34
15	20,29	15	22,48	15	24,44
16	21,99	16	23,63	16	26,12
17	22,83	17	25,52	17	27,06
18	24,05	18	31,37	18	32,28
19	24,49	19	33,39	19	38,16
20	25,23	20	35,12	20	39,50
21	31,22	21	41,07	21	41,03
22	35,29	22	42,68	22	41,03
23	38,82	23	42,68	23	41,03
24	39,76	24	42,72	24	41,03
25	40,52	25	42,72	25	41,03
26	40,55	26	42,72	26	41,03
27	41,00	27	42,72	27	41,03
28	41,27	28	42,72	28	41,04
29	42,32	29	42,81	29	41,04
30	43,78	30	43,33	30	41,04



Status May 2017, and recent work

- Status 4/2016: validated against full shell model for deformations and eigen-modes
- Resumed fall 2016. following parental leave of key engineer
 - Common project with DESY central mechanics service
 - Progress is slow (< 0.2 FTE)
- Go one step back and establish method with a simpler wheel-type toy model first
- Full analysis chain for toy model appeared in reach for May meeting
- However still many computational problems
- Intensified ANSYS support
 - increasing interest at DESY
 - civil engineering, astrophysics
 - ANSYS provider uses AHCAL as demonstrator
- Concluded toy model study

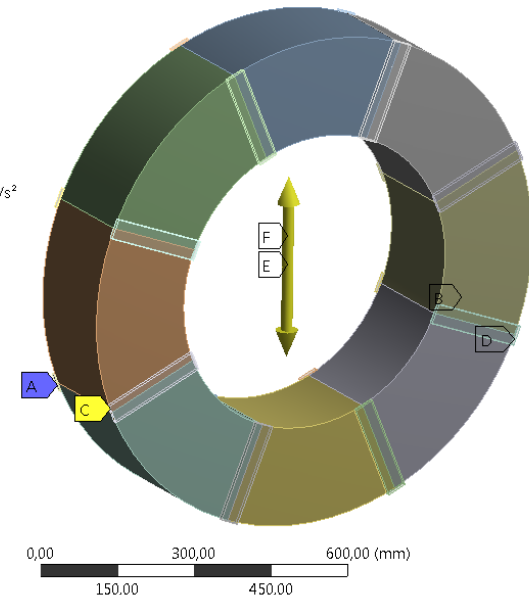


Test model: ring structure with 8 sectors

- > At small scales, represent problem, including several segments, contacts between segments via connection plates, and overall ring shape
- > Support analogue to complete AHCAL model, but additional ECAL masses not included
- > Can easily be adapted for different investigations
- > The full pass through the analysis can in principle be transferred to the AHCAL FE model

AC: MIT_Daempfung_Model, Static Structural
Static Structural
Time: 1, s

- A Fixed Support 2
- B Displacement 2
- C Displacement 3
- D Displacement 4
- E Standard Earth Gravity: 9806,65 mm/s²
- F Acceleration: 98100, mm/s²



ANSYS
R18.0
Academic

Computational progress

- > Keep data amount and computing needs under control
- > Re-organise analysis flow and re-use of results
- > Generate rotation-symmetric structure automatically

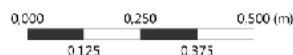
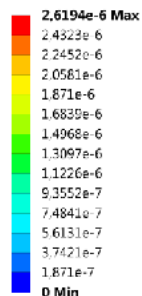
- > For toy model quasi-interactive analysis
 - > few 10 seconds turn-around for static case

- > 1. Static deformation under gravity
- > 2. Eigen-Mode analysis
- > 3. Frequency sweep: find resonances
- > 4. Excitation with realistic pulse

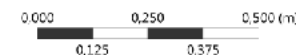
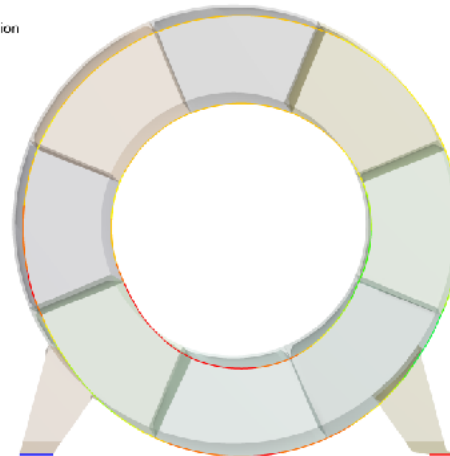
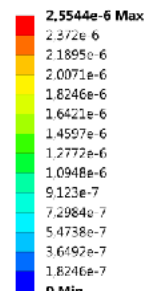
3D <=> CMS: Static analysis

> Left 3D,
right CMS

E: Standard_mesh - Static Structural
Total Deformation
Type: Total Deformation
Unit: m
Time: 1



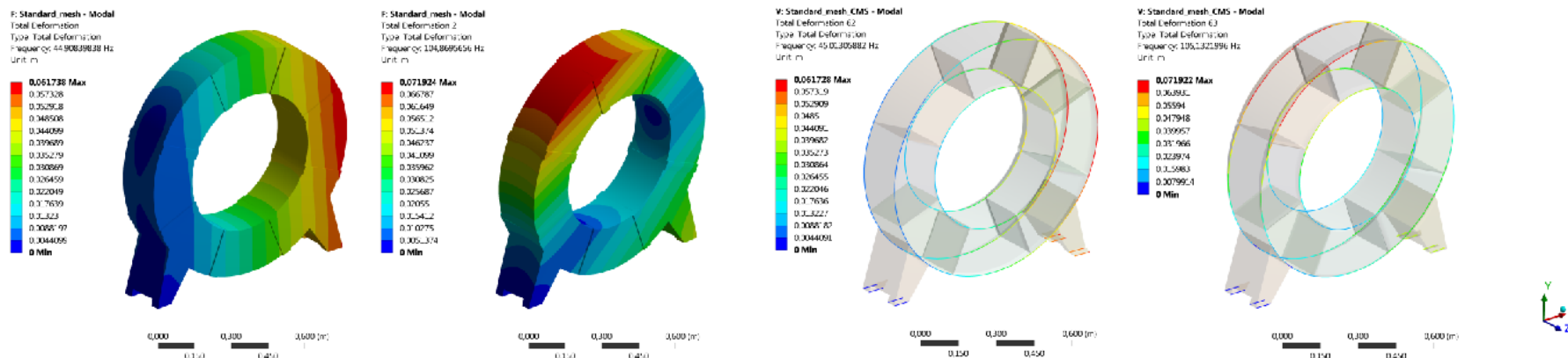
U: Standard_mesh_CMS - Static Structural
Total Deformation 2
Type: Total Deformation
Unit: m
Time: 1



Pos.	general Meshing	Type (3D/ CMS)	Mesh-Nodes	Total-CPU-time [in sec]	Used RAM [in MB]	Result-File size [in MB]	Max. Deformation [in m]
1	very coarse mesh	3D	13.328	15	211	4,50	2,567E-06
2	coarse mesh	3D	31.821	13	557	11,10	2,586E-06
3	standard mesh	3D	186.370	52	4.818	60,00	2,619E-06
4	fine mesh	3D	315.817	169	10.589	98,00	2,635E-06
5	very coarse mesh	CMS	11.730	21	121	0,90	2,396E-06
6	coarse mesh	CMS	31.766	20	214	1,00	2,494E-06
7	standard mesh	CMS	238.900	148	828	1,50	2,554E-06
8	fine mesh	CMS	369.796	93	1.703	2,00	2,605E-06

3D <=> CMS: Modal analysis

➤ Left 3D (Mode1, Mode2),
Right CMS (Mode1, Mode2)



Pos.	general Meshing	Type (3D/ CMS)	Mesh- Nodes	Total-CPU- time [in sec]	Used RAM [in MB]	Result-File size [in MB]	Eigenmode 1 at f [in Hz]	Eigenmode 2 at f [in Hz]	Eigenmode 3 at f [in Hz]	Eigenmode 4 at f [in Hz]
1	very coarse mesh	3D	13.328	64	427	284,25	45,08	105,12	136,72	307,56
2	coarse mesh	3D	31.821	116	1.064	826,06	45,11	105,20	139,41	309,64
3	standard mesh	3D	186.370	74	8.144	3.939,51	44,91	104,87	138,56	308,07
4	fine mesh	3D	315.817	890	17.231	5.778,50	44,87	104,73	138,01	307,38
5	very coarse mesh	CMS	11.730	120	162	7,38	45,57	106,86	144,21	317,34
6	coarse mesh	CMS	31.766	166	270	10,38	45,25	105,89	141,38	312,64
7	standard mesh	CMS	238.900	1.885	1.308	23,63	45,01	105,13	139,10	309,03
8	fine mesh	CMS	369.796	1.215	1.772	33,00	46,06	106,15	139,47	308,53

3D <=> CMS: Harmonic analysis

> computing needs for frequency sweep

Pos.	general Meshing	Type (3D/ CMS)	Mesh-Nodes	Total-CPU- time [in sec]	Used RAM [in MB]	Result-File size [in MB]	Max. Deformation [in m]
1	very coarse mesh	3D	13.328	566	250	3.670,00	4,847E-03
2	coarse mesh	3D	31.821	1.232	590	11.261,11	4,838E-03
3	standard mesh	3D	186.370	3.640	4.874	60.526,67	4,882E-03
4	fine mesh	3D	315.817	10.200	11.016	93.694,34	
5	very coarse mesh	CMS	11.730	1.703	162	125,69	4,739E-03
6	coarse mesh	CMS	31.766	22.620	232	181,50	4,807E-03
7	standard mesh	CMS	238.900	15.840	1.233	421,63	4,860E-03
8	fine mesh	CMS	369.796	24.540	1.758	595,00	4,647E-03



Boundary conditions harmonic analysis

- > apply acceleration amplitude in all 3 directions to supports
- > Sweep 0Hz - 3.000Hz

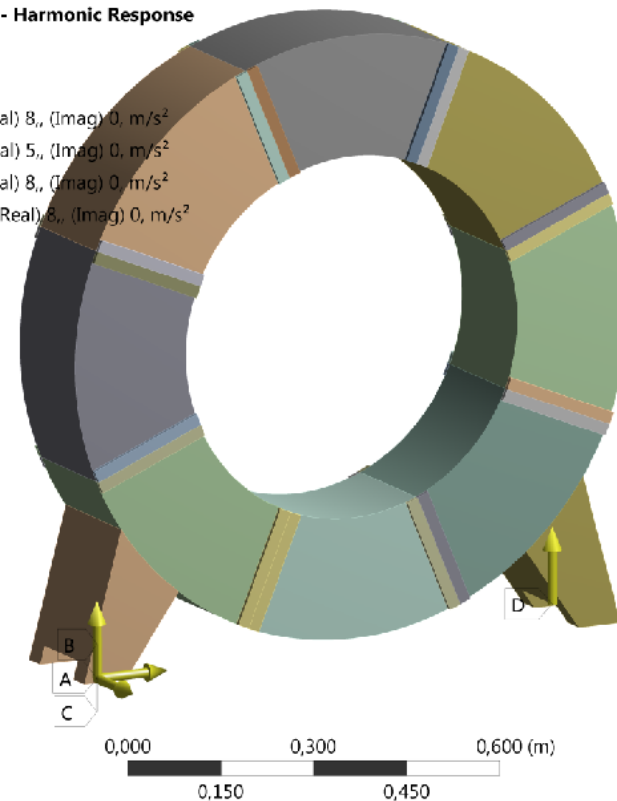
Details of "Acceleration_fix_x"	
Scope	
Boundary Condition	Fixed Support
Definition	
Base Excitation	Yes
Absolute Result	Yes
Define By	Magnitude - Phase
<input type="checkbox"/> Magnitude	8, m/s ²
<input type="checkbox"/> Phase Angle	0, °
Direction	X Axis
Suppressed	No

W: Standard_mesh_CMS - Harmonic Response

Harmonic Response

Frequency: 0, Hz

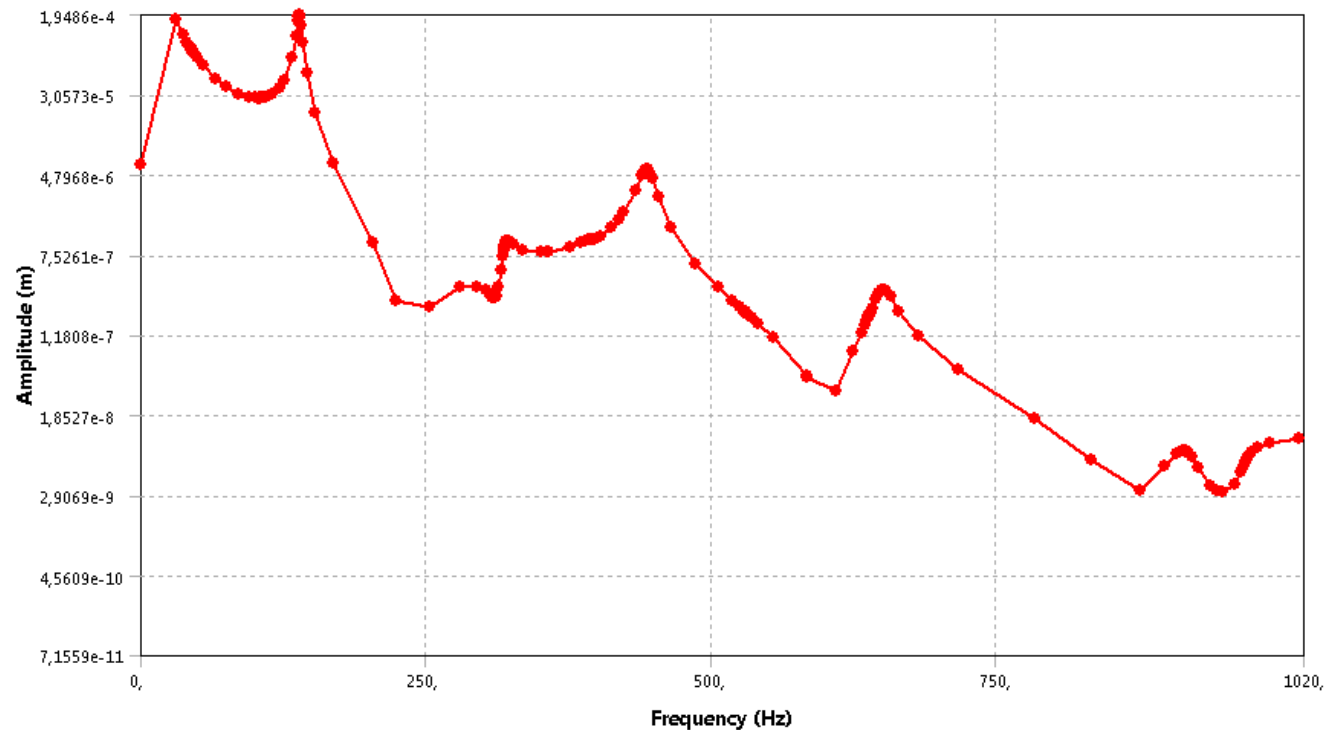
- A** Acceleration_fix_x: (Real) 8,, (Imag) 0, m/s²
- B** Acceleration_fix_y: (Real) 5,, (Imag) 0, m/s²
- C** Acceleration_fix_z: (Real) 8,, (Imag) 0, m/s²
- D** Acceleration_displ_y: (Real) 8,, (Imag) 0, m/s²



3D <=> CMS: Harmonic analysis

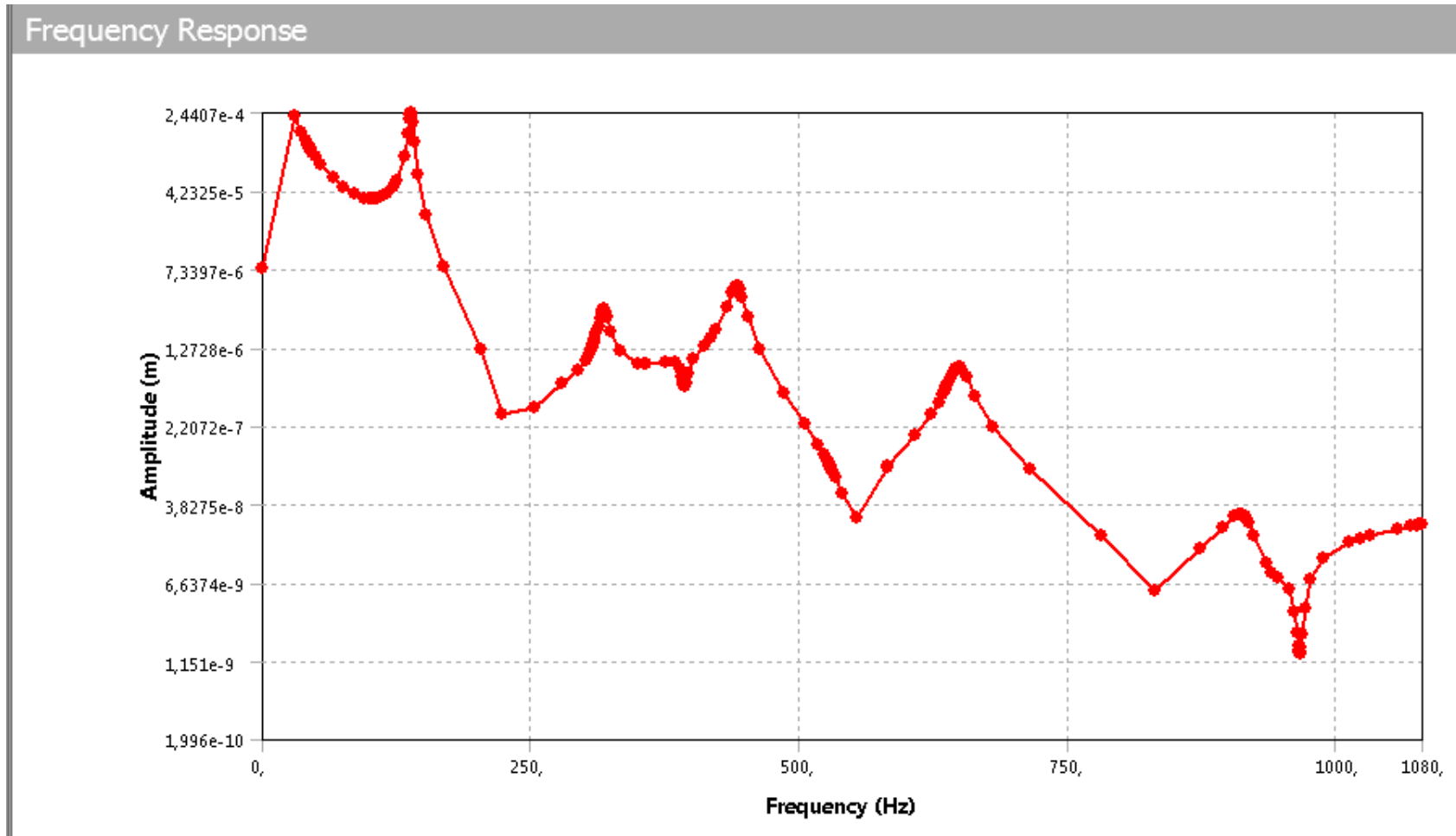
- CMS-Modell with „Standard Mesh“
- Results for x direction: maximum displacement over full side contour

Frequency Response



Vergleich 3D <=> CMS: Harmonische Analyse (MSUP)

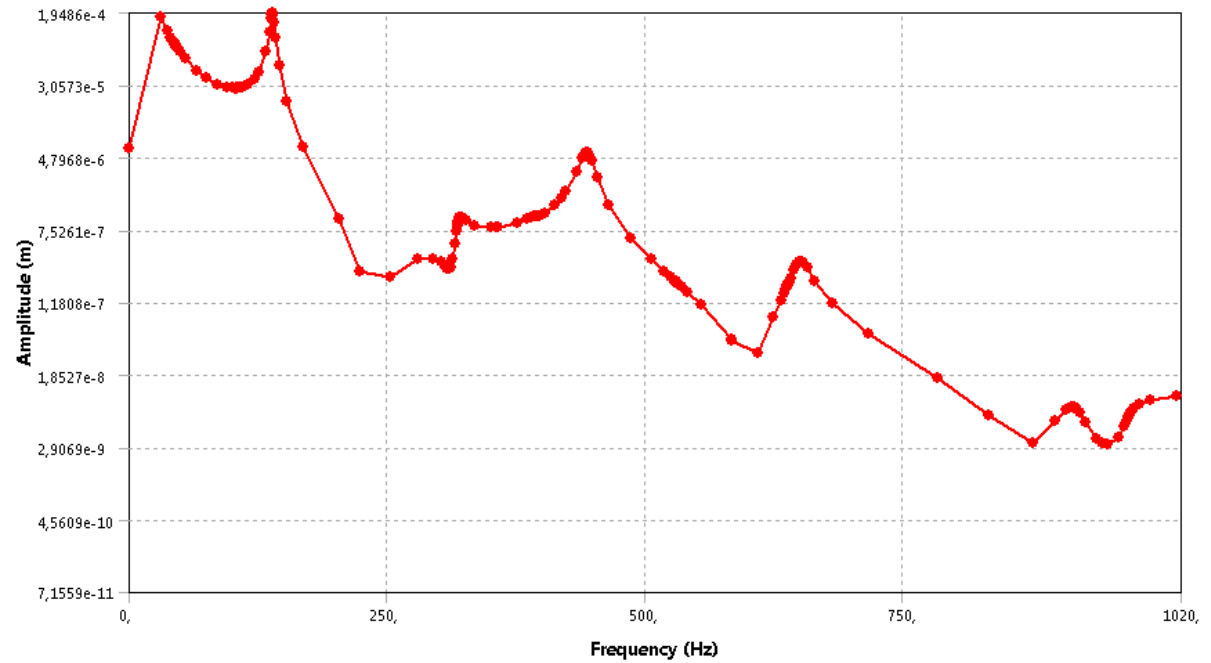
- Same Bode plot for 3D-Model with „Standard Mesh“
- Maximum in x-direction



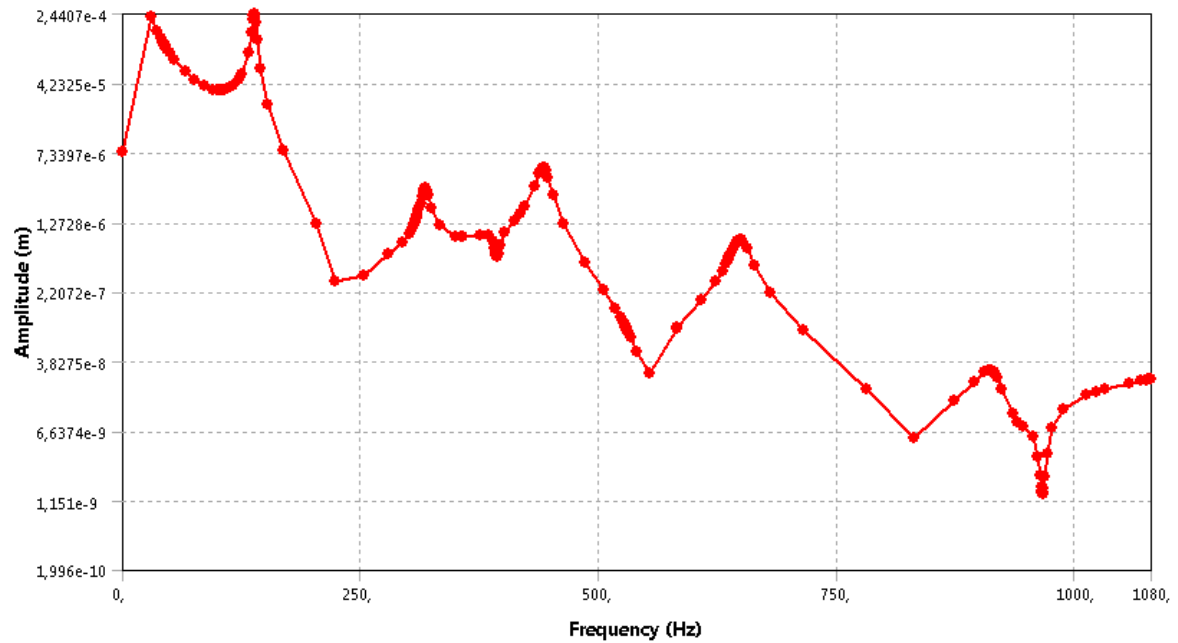
Both

Frequency Response

> CMS

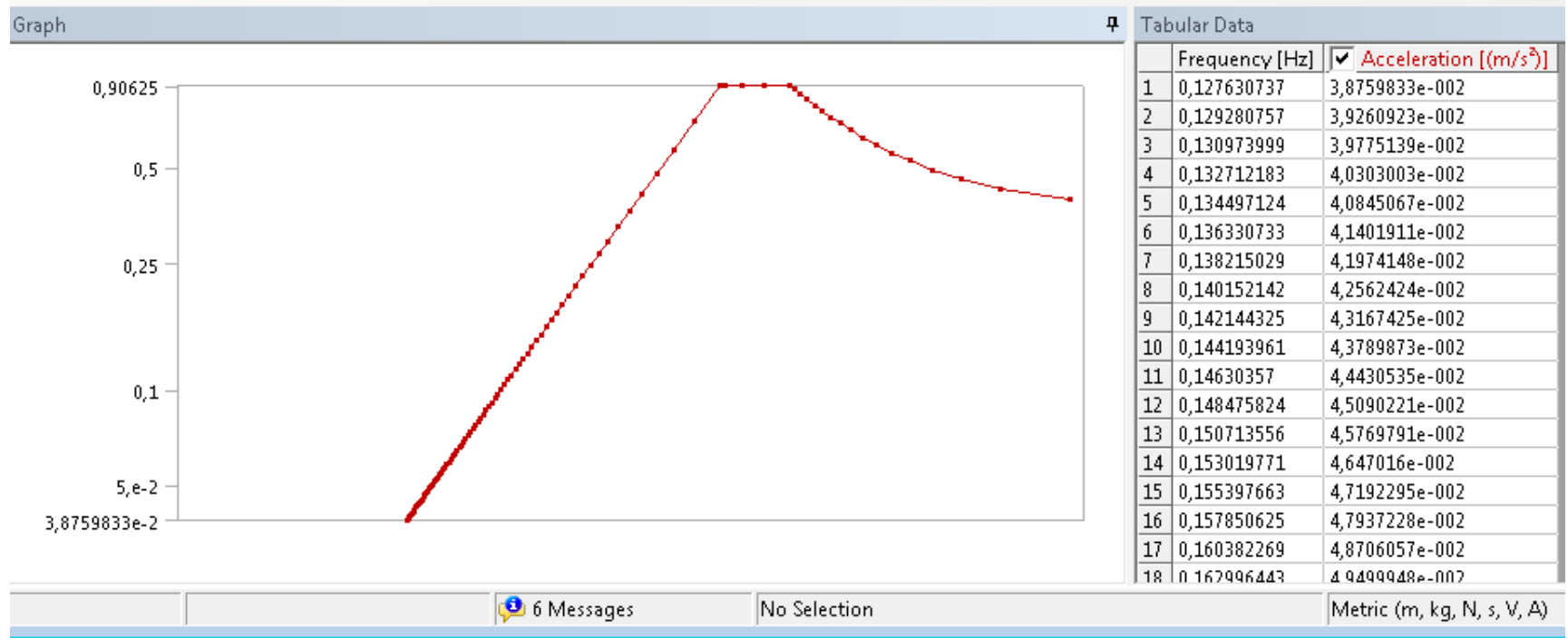


> 3D



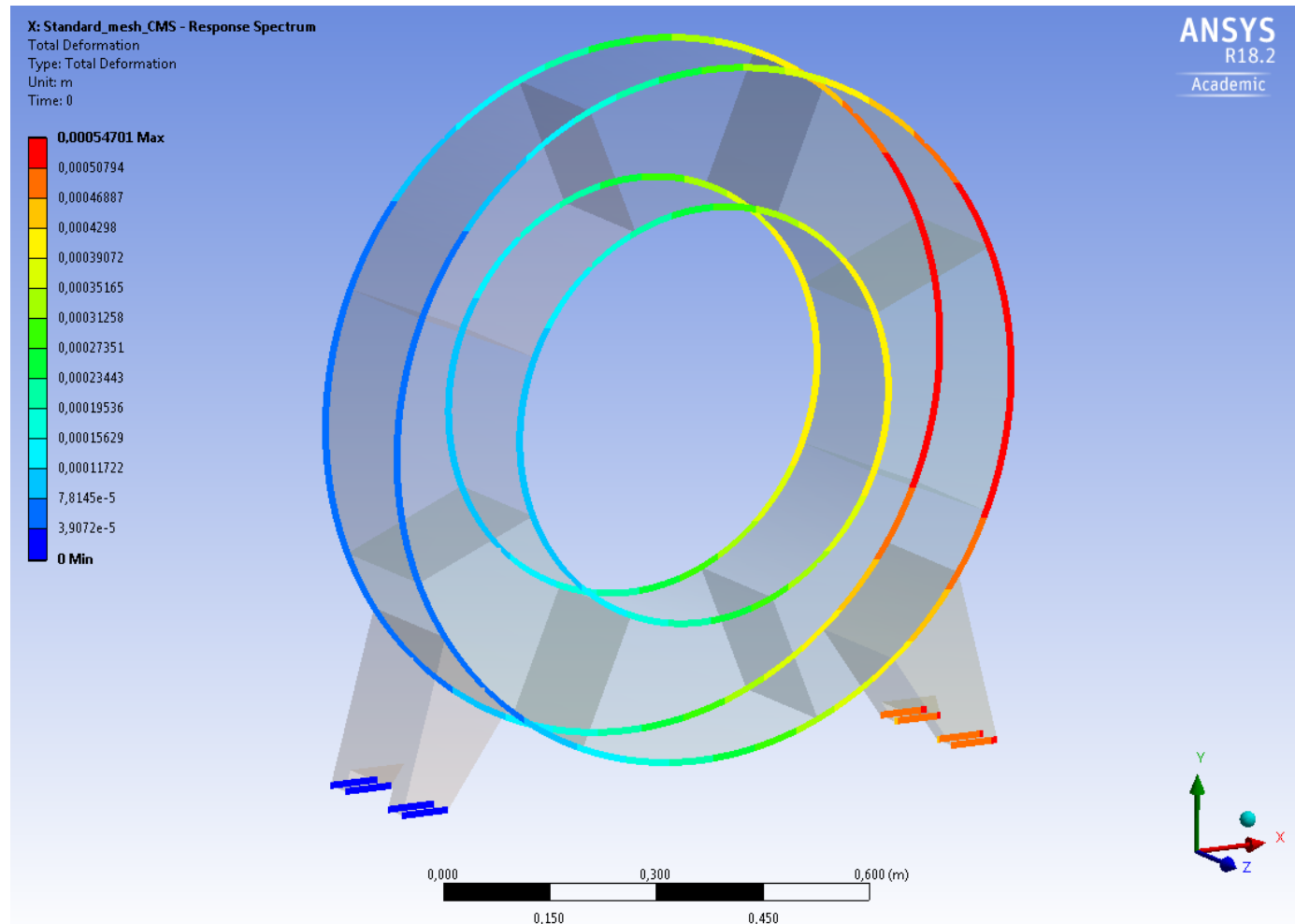
CMS: Response spectrum analysis

- Using earth quake data by NIED (Ichoniseki, 2011)
- For each axis input pulse on all supports
- Full pass through analysis successful



CMS: Response Spektrum Analysis

➤ Maximum amplitude 0.5mm



Summary & Outlook

- > CMS method is our only possibility for a dynamical calculation of complex calorimeter model
- > Method validated against full 3D mesh calculation on a toy model that reproduces main features
 - > final step still to be cross-checked
- > Significant reduction in computing resource needs

- > Next: apply to AHCAL structure
- > Will also not be straight-forward
 - > definition of sub-structures and boundaries needs engineering intuition
 - > cross-checks with 3D model only possible up to modal analysis
 - > then systematic studies
- > However, we now know that we can do it



Conclusion

- Challenging project, new difficulties encountered at every step
- Circumventing computing power limitations requires brain power

- Next steps:
- Complete analysis
- Outlook:
- Validation
- use existing structures
- on a shaker



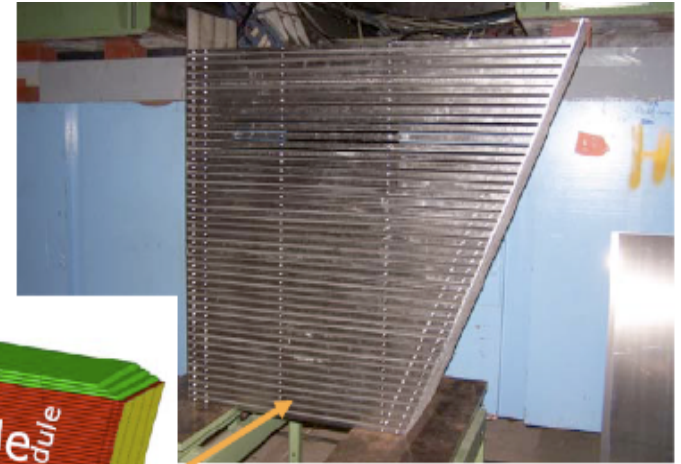
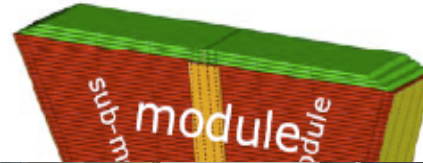
Conclusion

- Challenging project, new difficulties encountered at every step
- Circumventing computing power limitations requires brain power
- Next steps:
 - Complete analysis
 - Outlook:
 - Validation
 - use existing structures
 - on a shaker



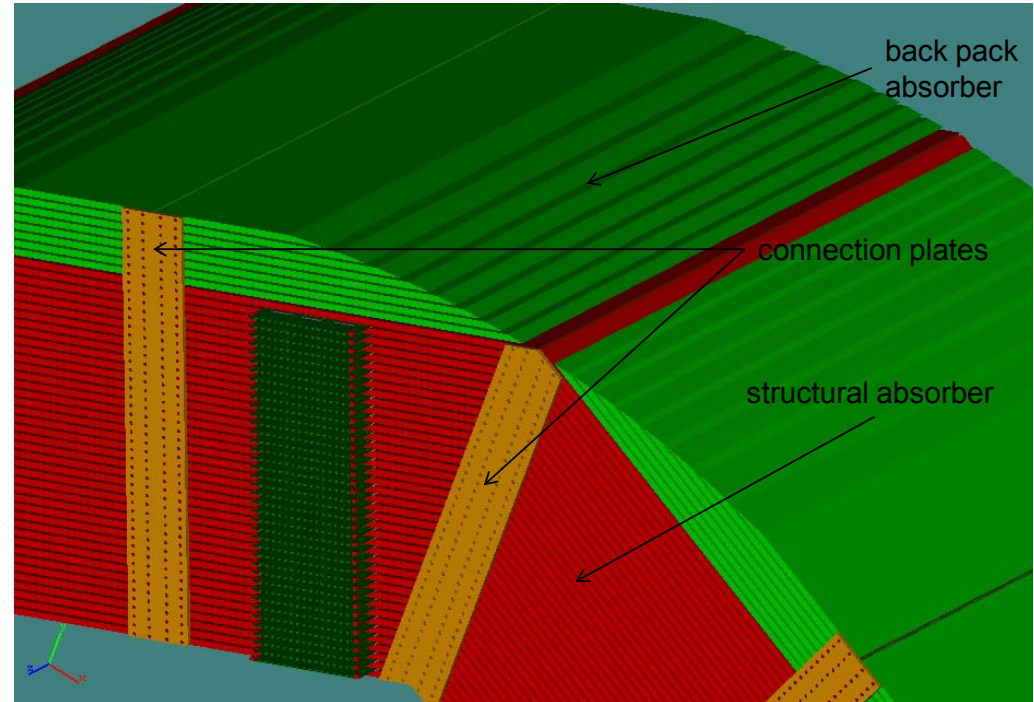
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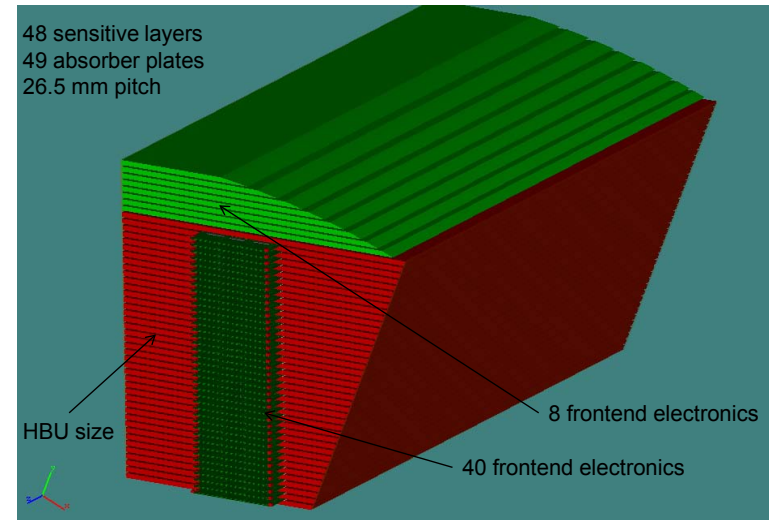
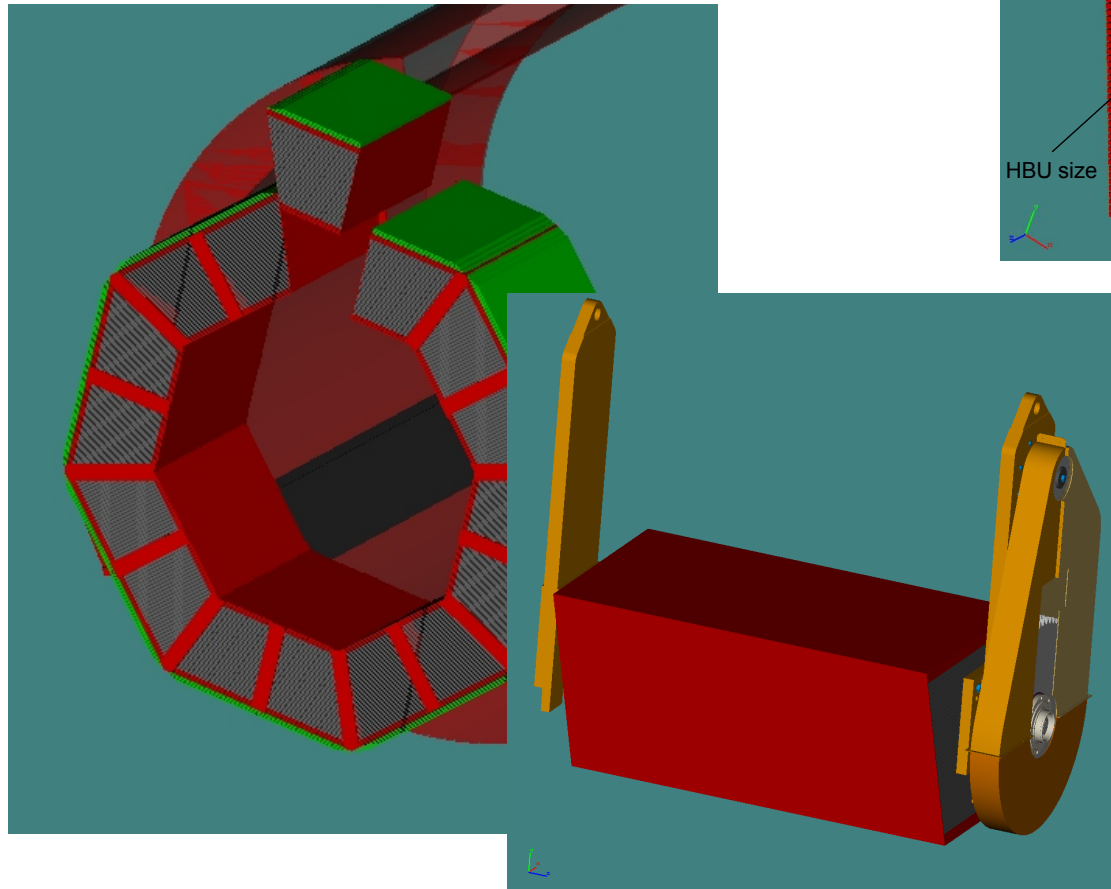
Design challenges

- Stainless steel
- Fine longitudinal sampling
 - 2cm plate thickness only
- No cracks, minimal un-instrumented regions
- Inside coil radius:
 - compact design to maximise no. of hadronic interaction lengths
 - tight tolerances over large dimensions
- Accessible electronics
 - external: short access
 - internal: longer shutdown or upgrade
- Earth quake stability
 - computational challenge



Small modules

- Small sectors (<18t) for easy transport and assembly in situ



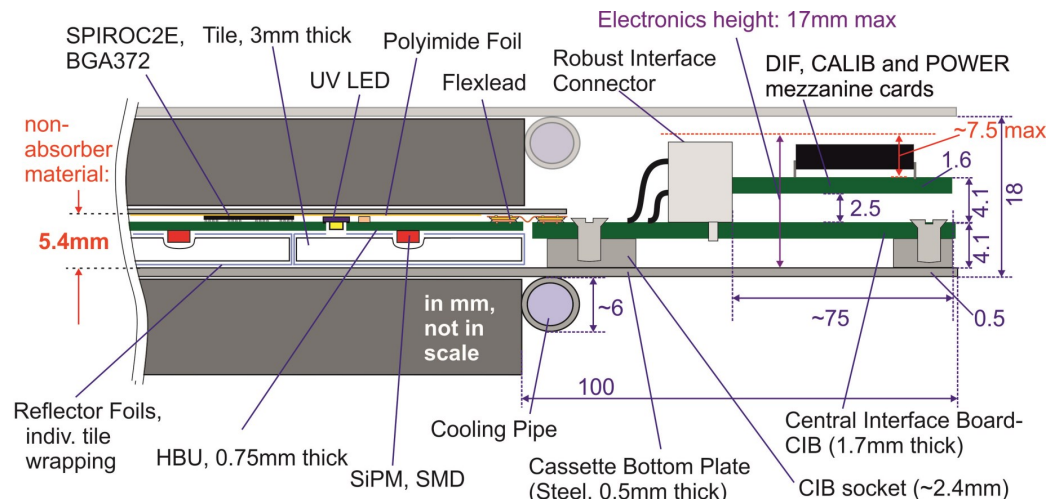
Design features

- Uniform sampling along shower axis
- Structure is made from rolled steel plates
 - flatness with roller levelling within 1mm verified
 - no machining: cost effective
- Assembly with screws
 - moderate tolerances
 - damping oscillations
- Thin side walls (5mm)
- 16 phi sectors, 2 rings
- Flexible structure, matched to flexibility of scintillator cassettes
- Varying layer width no problem for scintillator

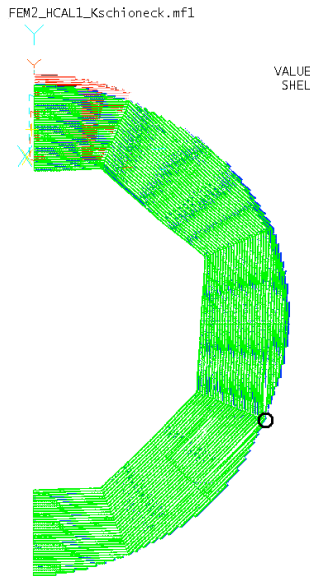


Cassettes

- Housing the scintillator and electronics active layers
- Made from stainless steel and contributes to absorbing material
- Sum of absorber plate and cassette thickness = 20 mm / layer
 - for both AHCAL and SDHCAL
 - material in absorber plate contributes to rigidity of structure
 - material in cassette contributes to load on structure
- AHCAL Physics prototype and early designs had 16 + 2x2 mm
- Present design and new prototype have 19 + 2x0.5 mm
- Weight is 17 kg /m²
 - cf SDHCAL 48 kg / m² with 2x 2.5 mm cassette

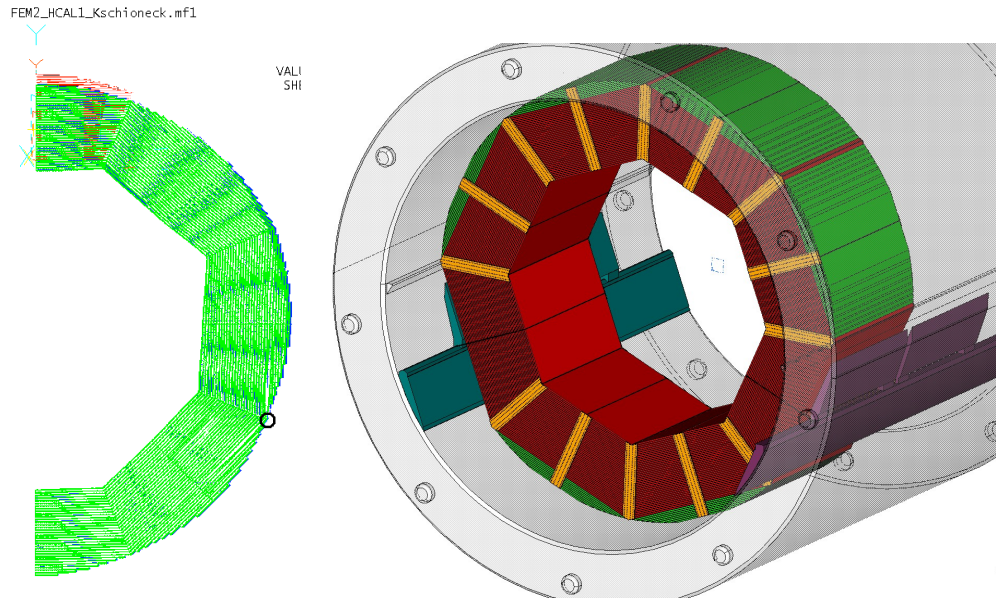


Orientation



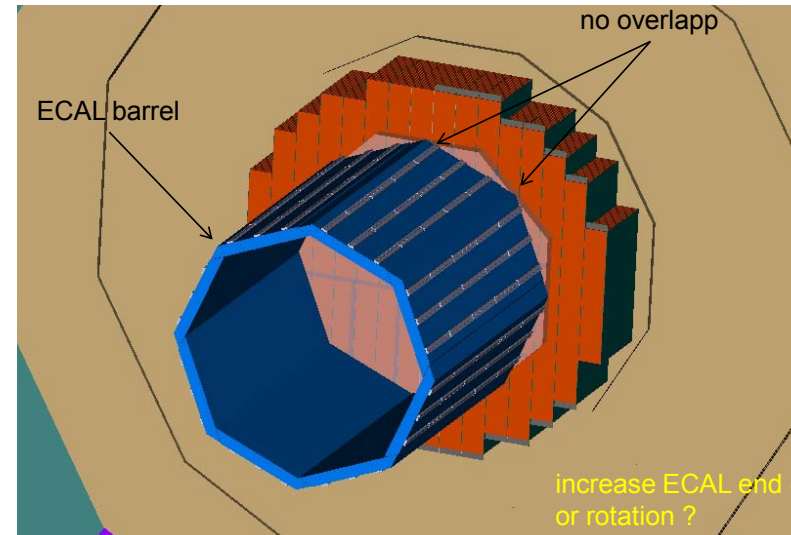
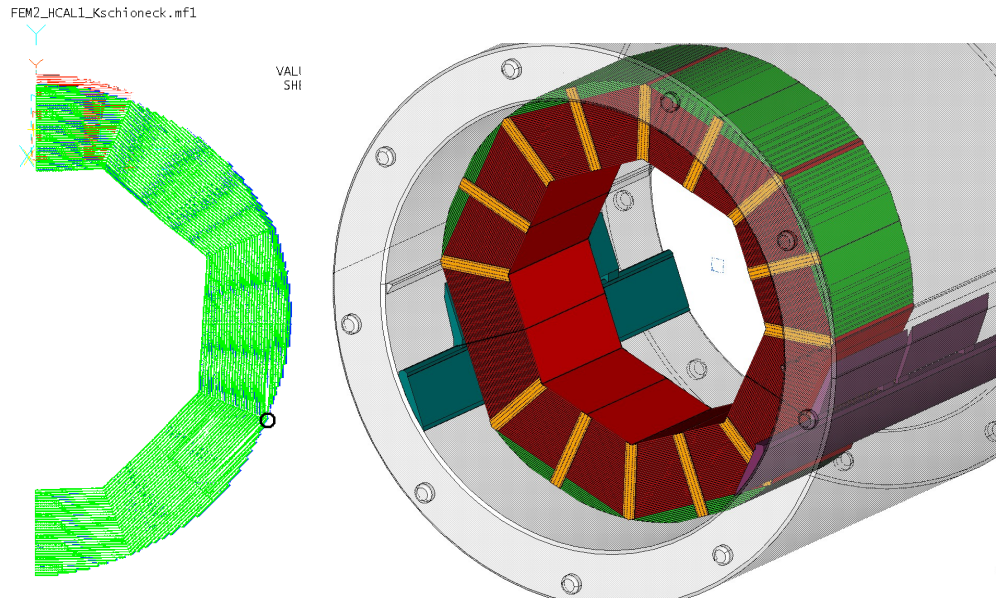
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- However, this leads to a conflict with ECAL endcap module structure and square insert

Orientation



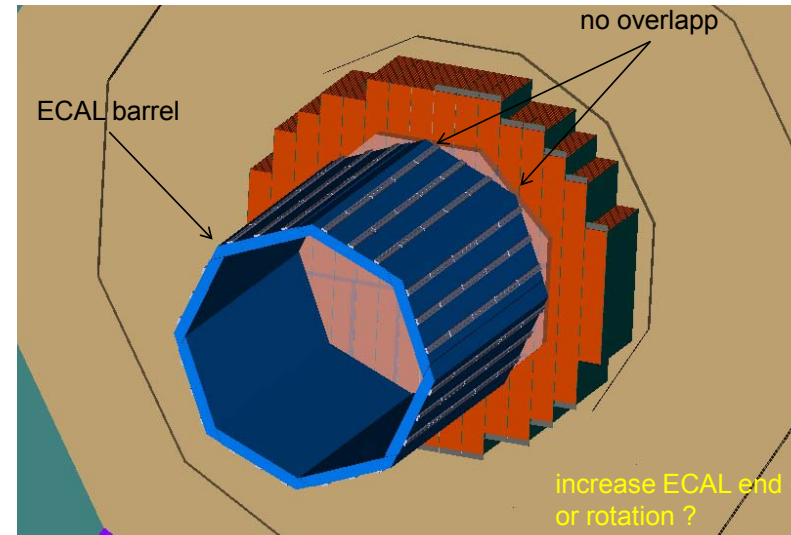
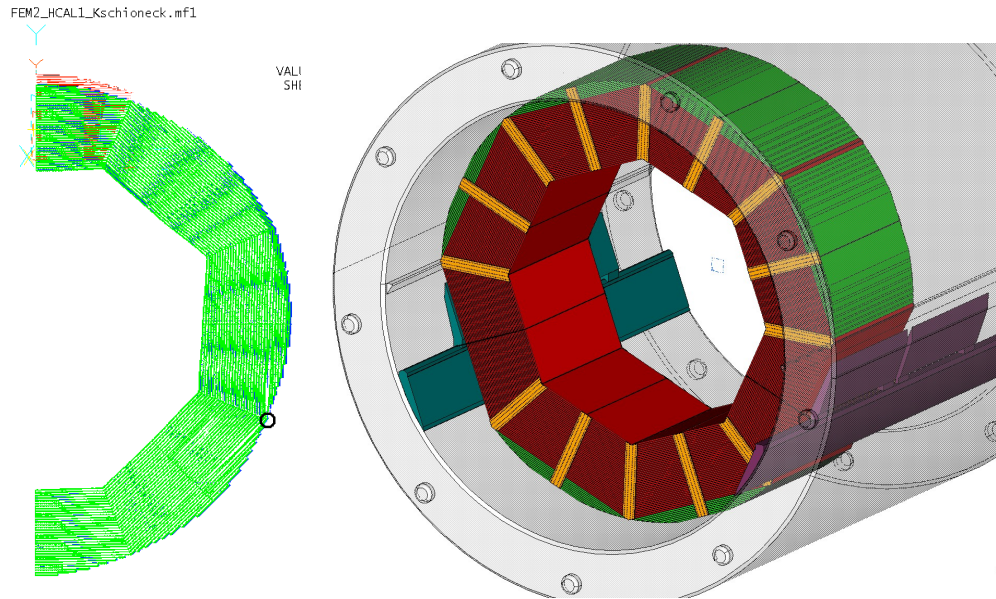
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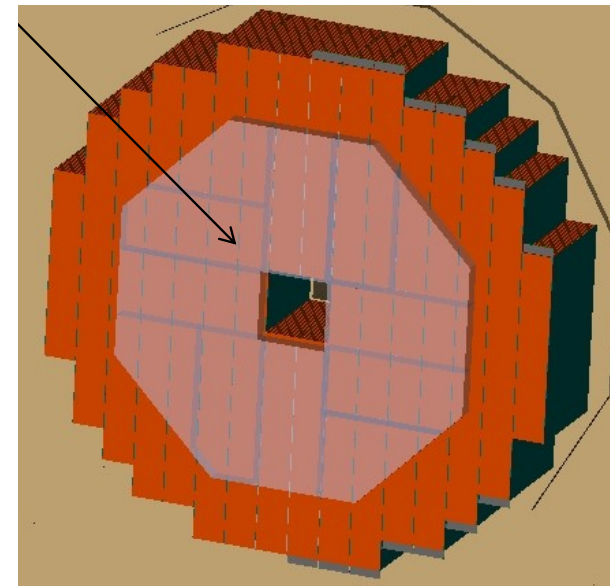


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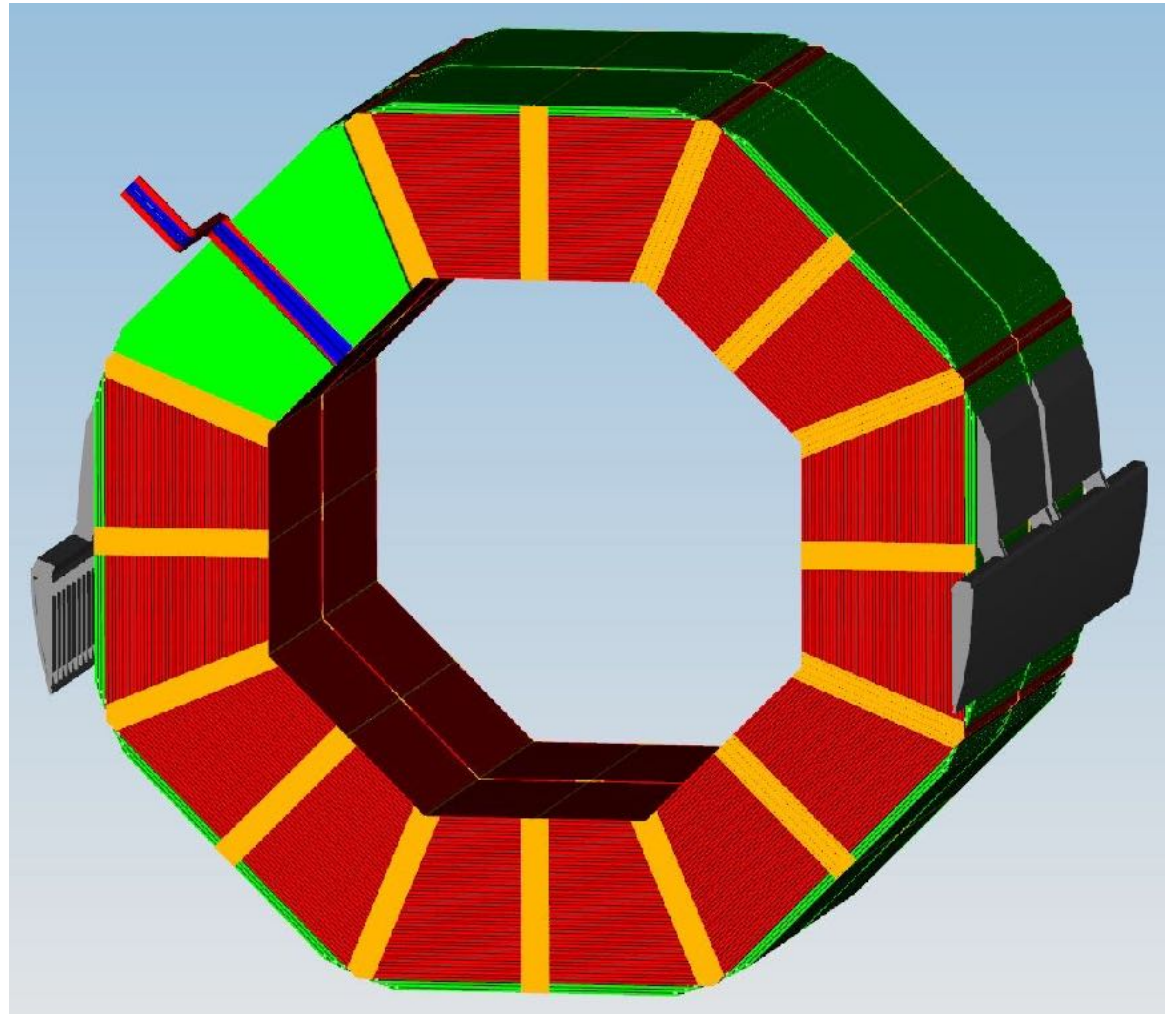


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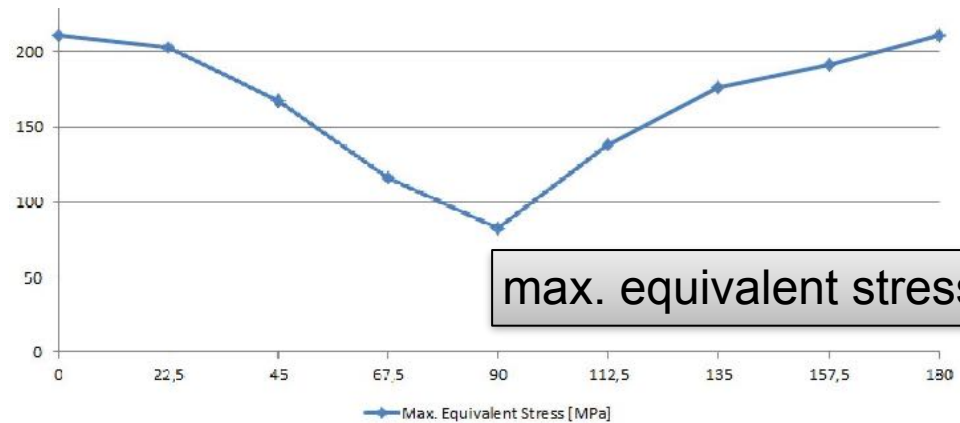
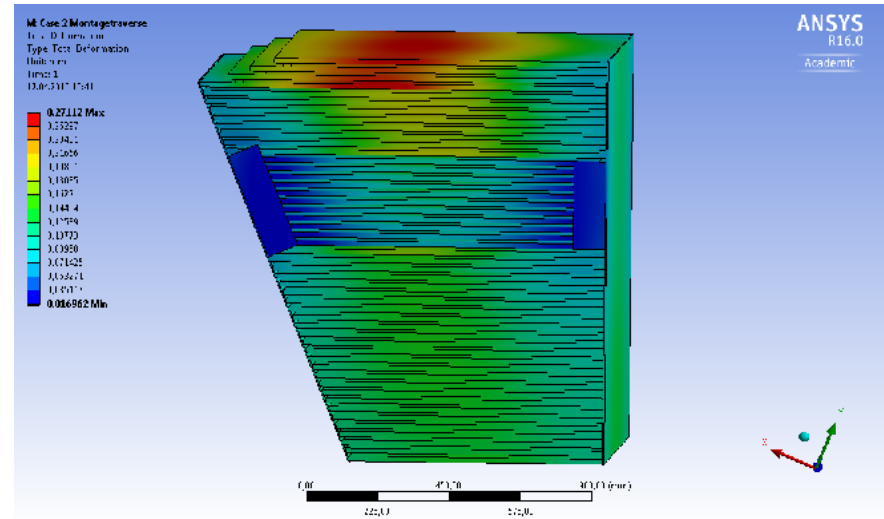
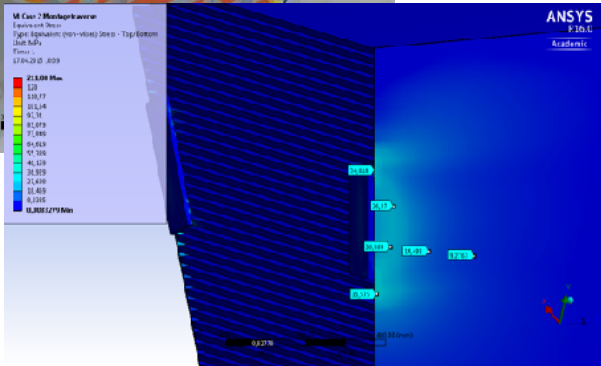
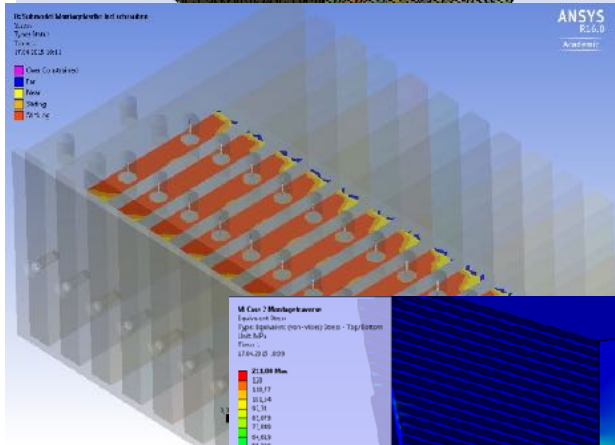
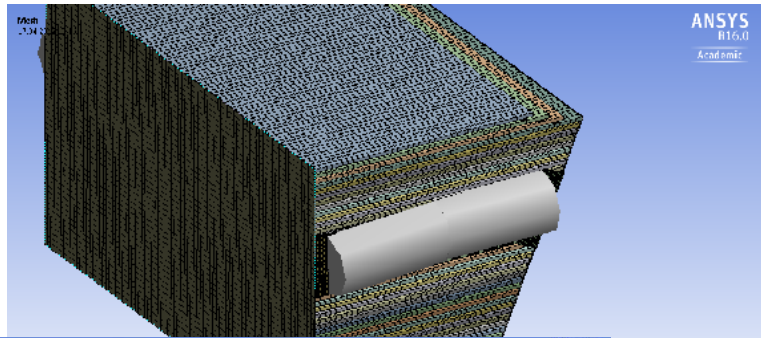


Alternative

- Integration into cryostat done
- Stability calculation to be re-done
- Expect somewhat larger deformations
- On the other hand, stability improves with
 - up-to-date plate thickness
 - 16 -> 19mm
 - possibly even thicker plates
 - smaller radius



Module installation



- need some modification of module connection pates